

Black Liquor Evaporation Book

Table of Contents

1. Introduction (to be written)
2. Evaporation Fundamentals
 - 2.1. Evaporator Hardware
 - 2.1.1. Evaporator effects
 - 2.1.2. Concentrator effects
 - 2.1.3. Direct contact evaporators
 - 2.2. Heat & mass balances in black liquor evaporation
 - 2.3. Design principles for black liquor evaporation
 - 2.4. Heat transfer in black liquor evaporators
 - 2.5. Evaporators and concentrators design considerations for troubleshooting
 - 2.6. Auxiliary processes
 - 2.6.1. Soap recovery (Jim is editing)
 - 2.6.2. Fiber removal (Jim is editing)
 - 2.6.3. Condensate segregation and stripping
 - 2.6.4. The basics of foul condensate stripping
 - 2.6.5. Collecting and burning noncondensable gases
3. Black Liquor Properties Relevant to Evaporation
 - 3.1. Introduction to black liquor properties
 - 3.2. Chemical composition of black liquor
 - 3.2.1. Black liquor composition
 - 3.2.2. Molecular size and conformation of Kraft lignin and polysaccharides
 - 3.2.3. Extractives
 - 3.2.4. Turpentine
 - 3.3. Thermal and transport properties
 - 3.3.1. Viscosity
 - 3.3.2. Boiling point rise
 - 3.3.3. Density
 - 3.3.4. Heat capacity and Enthalpy
 - 3.3.5. Heat of dilution
 - 3.3.6. Enthalpy
 - 3.3.7. Surface tension
 - 3.3.8. Thermal conductivity
 - 3.4. Summary
 - 3.5. Nomenclature
4. Control of Evaporator Fouling (Niko is writing)
5. Research needs in black liquor evaporation (to be written)

2. Evaporation Fundamentals

Black liquor evaporators and concentrators are designed to separate water from a solution/suspension of black liquor solids. Evaporation is heat driven, using condensing steam to evaporate water from black liquor to increase the dry solids content of the liquor. For evaporators to function efficiently, they must be able to:

- a. transfer large quantities of heat to black liquor across metal surfaces to evaporate water through nucleate or convective boiling,
- b. separate the vapor generated from the concentrated black liquor,
- c. utilize energy efficiently, evaporating several kg of water per kg of live steam condensed, and
- d. operate within constraints imposed by the black liquor being concentrated. These include scaling and fouling of heat transfer surfaces, corrosion, foaming, and thermal stability of the organic matter in black liquor.



Figure 1. Three evaporator bodies as part of a multiple effect set.

2.1 Evaporator Hardware

Configuration of black liquor evaporators and concentrators: The terminology used to describe black liquor evaporators and their configuration can be confusing when they are first encountered. The terminology reviewed here describes the components and configuration of black liquor evaporators and concentrators.

Black liquor evaporators and concentrators are configured from individual units, connected in series or parallel configurations. An *evaporator body* is a single evaporator unit that includes one heating element and one flash chamber. It is the basic building block of multiple effect evaporators and concentrators.

An *evaporator effect* consists of one or bodies with liquor boiling at the same pressure. Several bodies may be linked together in different configurations. Two or more bodies are often operated as a single evaporator effect, with a common steam supply to them and sequential black liquor flow through them. Figure 2 shows two evaporator bodies A and B linked to form one evaporator effect with a common steam supply to each body. Black liquor is fed to body B, from there to Body A, and finally is withdrawn from body A as concentrated product.

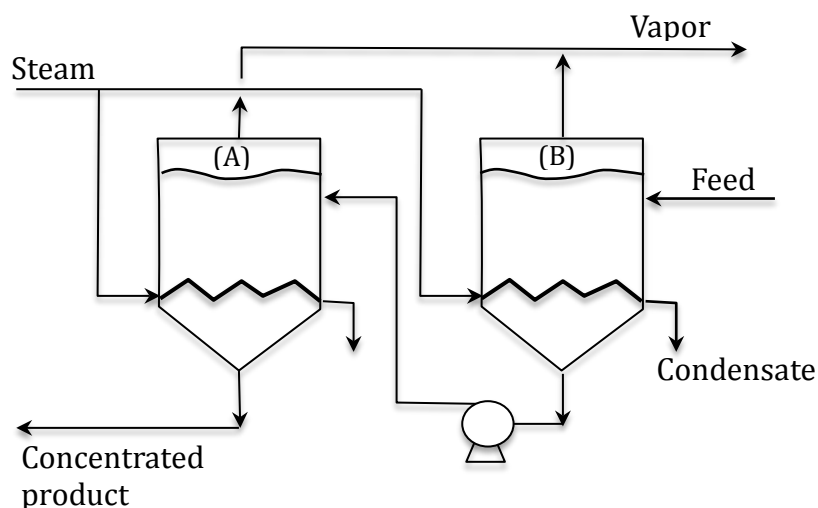


Figure 2. Diagram of a single evaporator effect consisting of two bodies, A and B.

A multiple-effect evaporator set consists of more than one evaporator effect in which the vapor from one effect is used as the heating medium for the next lower pressure effect. Figure 3 illustrates the steam, liquor, and vapor flow sequence for a counterflow, 3-body, 3-effect evaporator.

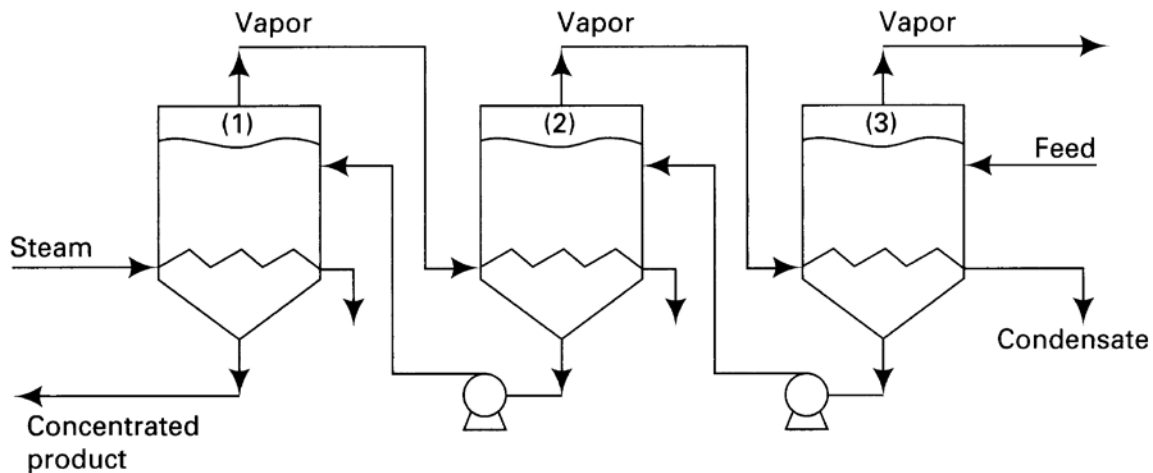


Figure 3. The steam, liquor, and vapor flow sequence for a counterflow, 3-body, 3-effect black liquor concentrator.

2.1.1 Evaporator Effects

Concentration of black liquor became an essential operation with the development of kraft pulp mills. Evaporation plays an important role in recovering pulping chemicals and in production of steam and electricity for mill consumption or resale. This section deals with the types of black liquor evaporators that are in common use and operation in today's kraft pulp mills. These include rising film long tube vertical (LTV) evaporators, falling film evaporators, forced circulation evaporators, and direct contact evaporators.

Rising film, or long tube vertical (LTV) evaporators (Figure 1 and 4) were the standard black liquor evaporators for decades, and many older mills still employ them.ⁱ Figure 5 illustrates their design. They consist of a tube bundle with tubes spaced and held in place by a tube sheet. Black liquor enters each evaporator body below the bottom tube sheet.

The tubes and tubes sheets (Figure 6) are made of 18 BWG, 304 SS.

The tube bundle is enclosed in a steam shell where either primary steam or vapor from a previous effect condenses to heat liquor within the tubes, driving evaporation. Black liquor is propelled upward within the tubes by the rising vapor generated.



Figure 4. Photo of one body of a rising film evaporator.

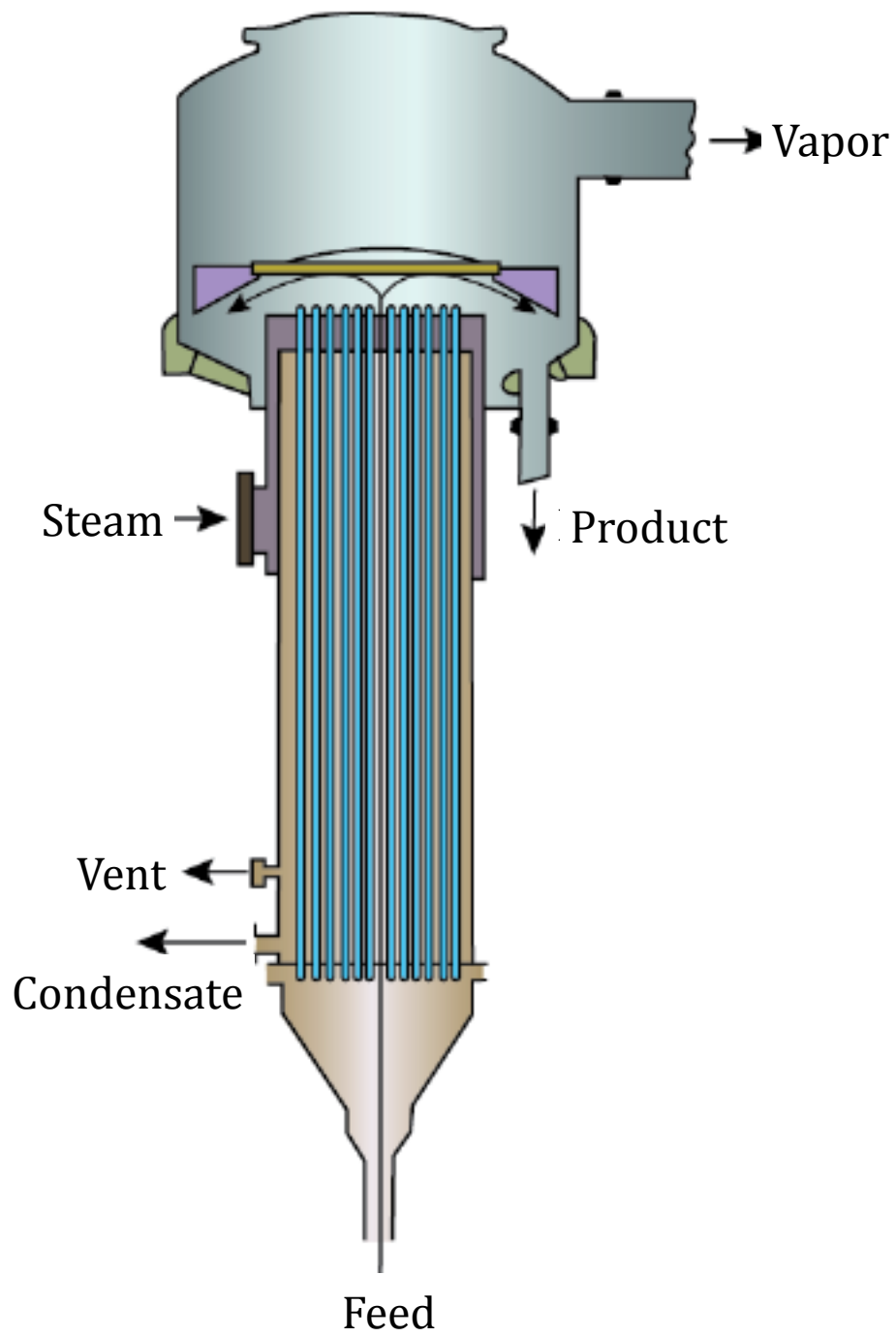


Figure 5. Diagram of a climbing film LTV effect.



Figure 6. Evaporator tube bundle and tubesheet. Tube diameter is 5 cm.

The heating element is a shell and tube heat exchanger, consisting of 5 cm OD tubes, typically 7.2 m to 9.0 m in length. Black liquor entering each effect is usually below its boiling point at the pressure within the effect, flows into the bottom liquor chamber and then upward, into the evaporator tubes. Steam condensing on the outside of the tubes heats the liquor. The lower section of the tube bundle acts as a preheat section, heating the liquor to its boiling point. The liquor temperature increases as the liquor rises upward in the tubes, and the pressure decreases. Evaporation begins at the height within the tubes where the vapor pressure of water in the black liquor equals the system pressure.¹

The rate of heat transfer to black liquor in the lower, subcooled region of the tubes is poor. Long subcooled regions have to be avoided because the heat transfer rate is so low in them that it limits the overall evaporation rate in the body. Heat transfer rates increase by an order of magnitude from the preheat section to the boiling

region within the tubes. Heat exchangers between effects are used to bring liquor temperatures entering an effect to within 10°C of the boiling temperature within that effect.

Vapor and concentrated black liquor exit each body from the top of the tubes into a vapor dome that separates black liquor from vapor. The vapor dome material is normally carbon steel, and is protected with stainless steel in areas where the black liquor concentration exceeds 25 wt-% dry solids content.

The vapor dome contains an umbrella or deflector located directly above the top tube sheet. It acts as the first stage of liquor and vapor separation. It must completely shield the tubesheet from product liquor flowing back into the tubes from above. A second stage of vapor-liquid separation (a demister) in the vapor dome is necessary to separate fine black liquor droplets from the exiting vapor.

The operating pressure for the first effect of most LTV evaporators is from 6 to 7 bar.

Evaporator effects may be equipped with either an internal liquor heater or an afterheater, and sometimes both. These heaters are dedicated sections of tubes designed to heat liquor only. No boiling should take place in them.

Crystallization fouling and high liquor viscosity limit the ability of LTV evaporators to produce concentrated black liquor at dry solids contents above 50 wt-%.ⁱⁱ Liquor recirculation is sometimes used to improve operation in effects operated near or at the onset of sodium salt crystallization from black liquor.

LTV evaporators have poor turndown ratios and that makes it difficult for them to operate with black liquor throughput less than 70% of design capacity. This is because rising vapor within the tubes is necessary to propel the black liquor upward, thinning the liquor film at the tube walls, and generate high heat transfer rates. The ΔT per effect must be at least 7-8°C for them to operate well. At lower throughput rates, the overall ΣT required to achieve the target product dry solids content is too small to meet the minimum 7-8°C ΔT per effect. Two-phase flow is

not sustained and the flow regime becomes single-phase flow, heat transfer rates drop by an order of magnitude, and required evaporation rates are not met.

Table 1. LTV Characteristics

Tube dimensions:

Length: 28 ft industry standard (20-33 ft were used earlier)

OD: 2.0 inches (5.08 cm)

Wall thickness: 18 BWG

Metallurgy: 304 SS

Table 2. Advantages and disadvantages of rising film LTV evaporators.

Advantages of LTVs:

- Simple design
- Minimal control requirement
- Low operating horsepower

Disadvantages of LTVs

- Requires large ΔT (7-8°C) for each effect
- Poor turndown ratio
- Product dry solids content is limited to 48-52 wt% dry solids content because of severe sodium salt scaling
- Cannot process black liquors with unusually high viscosity in the first and second effects

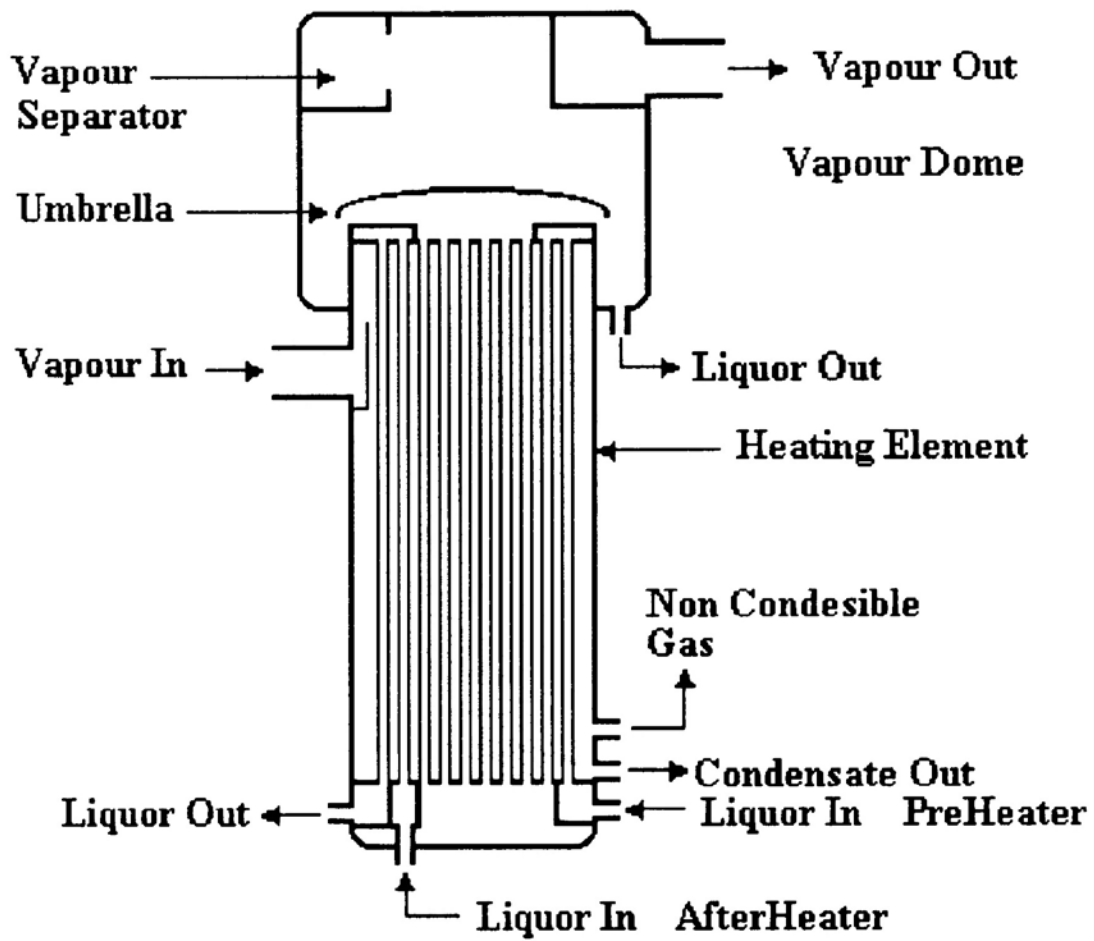


Figure 7. Components of an LTV evaporator

Falling film evaporators, tube types (Figure 8)

Falling film evaporators have replaced LTV evaporators as the industry standard for new installations since the 1970s. Falling film evaporators consist of a heating element, a vapor-liquor separator, a liquor sump, a liquor recirculation line and pump, and a liquor flow distribution device. The heating elements can be either tube bundles (see Figure 1-5) or lamella-type heating units. With tube bundles, black liquor flows downward, either on the inside or outside of the tube. Most suppliers of tube type falling film evaporators design for liquor flow inside the tubes. With lamella-type evaporators, two sheets of stainless steel are welded together to form a heating element with steam condensing in the space between the two sheets. Black liquor flows downward on the outer surfaces of the lamellas. Uniform liquor distribution on the heat transfer surface is critical with either type, to maintain a continuous liquor film along the entire length of the heat transfer surface. This avoids dry areas and deposits of black liquor solids along the heat transfer surface.

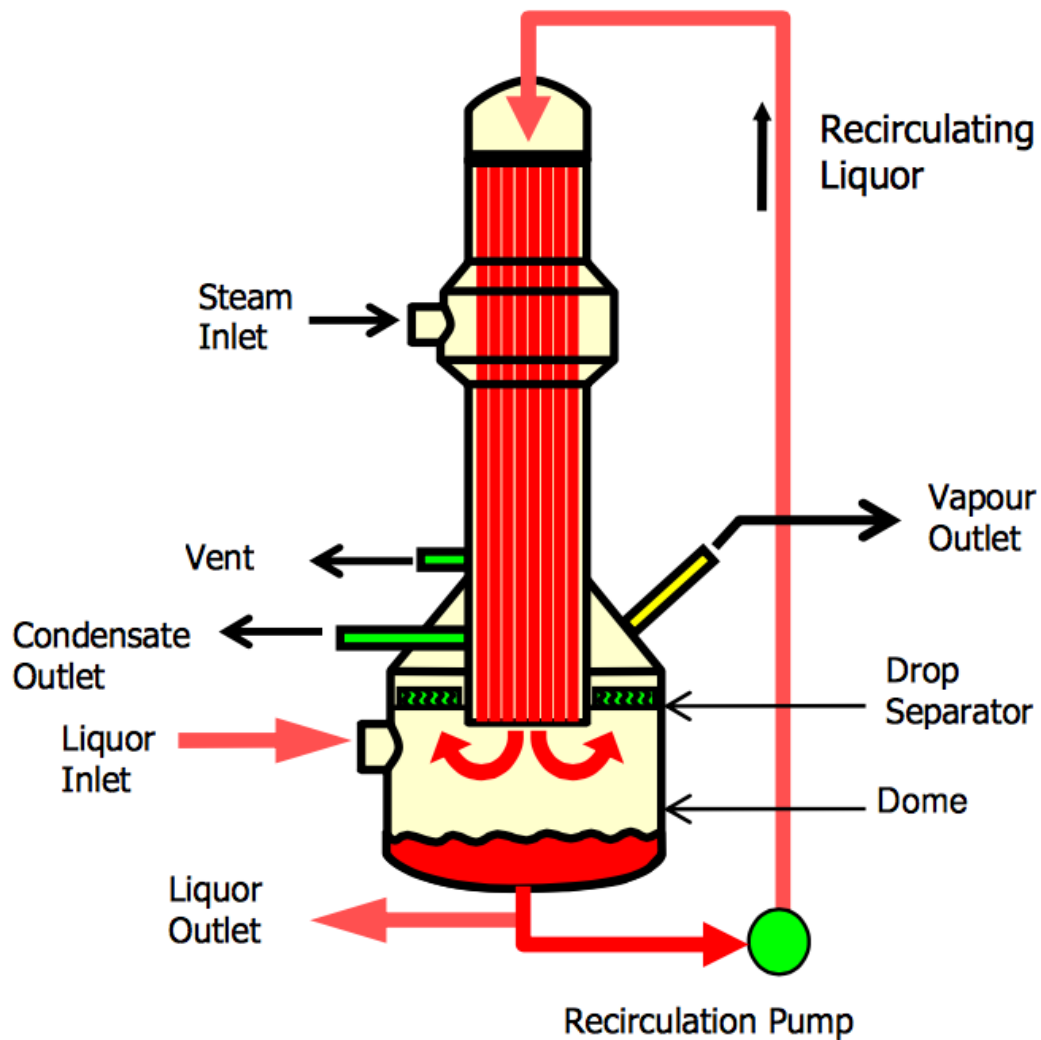


Figure 8. Diagram of a tube type falling film evaporator. (from Patel, TAPPI slides)

Falling film evaporators provide higher heat transfer rates, especially at higher liquor concentrations, than do rising film evaporators. Liquor recirculation helps to minimize crystallization fouling by providing suspended crystals in the liquor as more favorable crystallization sites than heat transfer surfaces. The crystals grow by relieving supersaturation of Na_2CO_3 and Na_2SO_4 in black liquor. This minimizes their tendency to deposit as scale on heat transfer surfaces. Falling film evaporators are suited for handling viscous liquids.ⁱⁱⁱ A high rate of recirculation-to-feed liquor

flow rates, typically about 20, improves the overall rate of heat transfer by eliminating the long subcooled region encountered in LTV evaporators.



Figure 9. Photo of a falling film evaporator installation.

Table 1-3. Heat transfer rates with a pilot tube-type falling film evaporator with black liquor on the outside of the tubes

Evaporator type	Dry BL solids, wt-%	Heat flux, W/m ²	ΔT, °C
Pilot falling film, BL outside of tubes	43.9	10570	5.3
	55.9	9400	6.2
	61.5	8670	6.0
	67.0	8300	6.2
	72.5	7630	7.0
	78.6	4800	8.0
	84.9	3500	25.0

Falling film evaporators with lamella heating surfaces

The heating elements are referred to as dimple lamellas because of their appearance. They are comprised of two sheets of stainless steel welded together at their perimeters, and tack welded together at points across the sheets. Once welded, the lamella unit is inflated with pressurized nitrogen to open up the steam or vapor space between them. Figure 1-9 illustrates their appearance.

The lamellas are arranged in a header as a bank of lamellas. The bank of lamellas is the heating element for a body, and the outer shell acts as a combination vapor dome and liquor sump. Liquor is circulated by pumping from the sump area to a distribution tray located above the lamellas. The liquor flowing over the lamellas is heated by the condensing vapor inside the lamellas. Heat transferred from the condensing vapor to the black liquor evaporates water from it.

The heating elements are arranged in three or four separate sections within a common vapor body. Each section has a separate liquor circulation pump. With this arrangement, one section can be taken offline and washed to remove scale while the remaining sections are in operation.

When employed as concentrators, these falling film units are fed black liquor at about 50 wt% dry solids content. They produce product liquor at 68-75 wt-% dry solids content after flashing of the concentrator product liquor.

Condensate segregation and stripping and is accomplished by using sections of the heating elements as countercurrent strippers for volatile organic species. This is discussed in more detail in Section 2.2.3.

Table 1-4. Characteristics of lamella-type falling film evaporators

- ☐ Lamella dimensions: 4 ft x 20 ft (1.2 m x 6.0 m)
- ☐ Constructed in banks of lamellas
- ☐ Liquor distribution trays above the lamellas ensure that liquor is distributed uniformly on the lamellas and not between them.
- ☐ Steam condenses on the inside of the elements

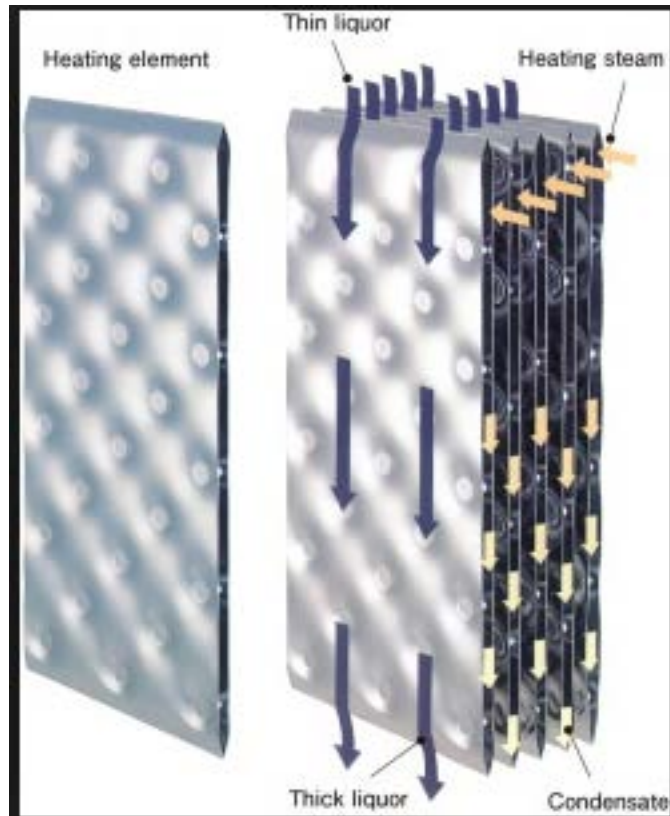


Figure 1-9. Illustration of heat transfer elements for lamella-type evaporators.

Table 1-5. Advantages and disadvantages of lamella-type evaporators.

Advantages of lamella-type evaporators:

- ☐ Suitable for viscous liquids
- ☐ Minimal scaling problems
- ☐ Low pressure drop
- ☐ Excellent turn down capability / can operate at low ΔT
- ☐ Can achieve >75 wt% dry solids content product liquor

Disadvantages of lamella-type evaporators:

- ☐ Lamella repair is costly and difficult
- ☐ Liquor distribution tray is susceptible to plugging with fiber, scale, etc.

- Operation relies on extensive level control of liquids
- Black liquor foams easily at liquor dry solids content of less than 18%

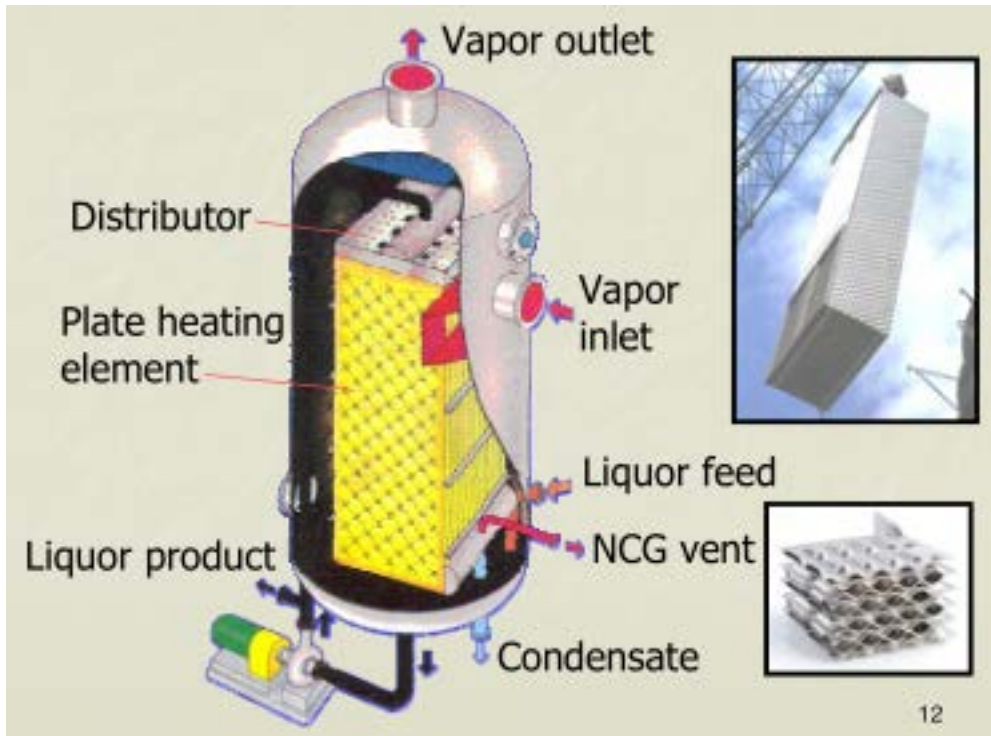


Figure 1-10. Illustration of an evaporator body with lamella-type heating elements.

2.1.2 Concentrator Effects

Black liquor concentrators differ from black liquor evaporators in two important aspects: (a) they process black liquor at higher dry solids content and therefore at higher viscosity, at higher temperatures, and (b) they are de facto crystallizers as well as evaporators as they concentrate black liquor to above the total dry solids content where Na_2CO_3 and Na_2SO_4 crystallize from solution. The viscosity of black liquor at 80wt-% dry solids content is from 10^3 to 10^4 mPa-s at 100°C and 10^2 to 10^3 at 130°C . Sodium salts begin to crystallize from black liquor when it is concentrated to 50% dry solids content or higher. Either burkeite or dicarbonate will crystallize first, depending on the ratio of Na_2CO_3 to Na_2SO_4 dissolved in the black liquor. Dicarbonate is more aggressive in scaling and its scaling is more difficult to control. Black liquors in today's kraft pulp mills are richer in Na_2CO_3 relative to Na_2SO_4 than they were 30 or more years ago. As a result, dicarbonate scaling has become much more prevalent than Burkeite scaling.^{iv} Evaporator fouling and how to minimize it is discussed in detail in Chapter 4.

Lamella-type falling film concentrators

These are similar to the lamella-type falling-film evaporators in most respects. There is no differentiation between “evaporator” effects and “concentrator” effects in single, 7- or 8-effect sets that consist of one or the other type of falling film effects.

Falling film concentrators with tubular heating surfaces

Tube-type falling-film concentrators consist of a bundle of vertical tubes with a liquor distributor at the top and a vapor body and liquor sump below the tubes. They are designed either for black liquor to flow downward inside the tubes with steam condensing on the outside of the tubes, or for black liquor to flow down the outside of the tubes and steam within the tubes. Liquor recirculated from the sump is mixed with feed liquor in a ratio of about 20:1. The black liquor mixture is pumped to the distributor lamella. The distributor lamella delivers the black liquor either to the inside of the tubes where it flows downward as a thin film on the inside

surfaces of the tubes, or downward as a liquor film on the outer tube surfaces. The distributor lamella is designed to provide a uniform flow of liquor to each tube. This is essential to avoid liquor drying on the surface of some of the tubes. Black liquor flows from the bottom of the tubes and into the liquor sump. Condensate is collected from near the bottom of the steam shell.

Black liquor inside tubes

Falling film tube type evaporators look much like an LTV that has been turned upside down, but they operate much differently. Black liquor flows down the inside of the tube surface from the top of the tubes. The liquor is distributed to each tube by passing through a distribution tray, although some use a special splash lamella instead of the distribution lamella. After evaporation, the black liquor is collected in the lower sump area and is circulated by pump to the top tubesheet. The size of the circulation pump is based on the number of tubes in the heating element. A circulation flow of 15 to 27 L/min (4 to 7 gpm) per tube is normal.

The tubes used in the falling film design are normally 5.1 cm (2.0 inch) in outside diameter, 18 BWG, 304 type ERW^v stainless steel. Tube lengths vary by manufacturer from 8.4 to 15m (28 to 50 ft).

Vapor enters the heating element section and condenses on the outside of the tube surface. Energy is transferred through the tube wall to the black liquor. Vapor flows downward, concurrent with the black liquor. It leaves the lower sump and flows to the shell side of the next effect.

Table 1-6. Advantages and disadvantages of tube type evaporators with black liquor inside tubes.

Advantages of tube-type falling film evaporators with black liquor inside of tubes:

- ☐ Suitable for viscous liquids to 300 cP
- ☐ Minimal scaling problems
- ☐ Low pressure drop
- ☐ No head-induced boiling point elevation
- ☐ High heat transfer rates at low ΔT
- ☐ Excellent turn down capability / can operate at very low ΔT
- ☐ Can achieve > 75 wt% dry solids content product liquor
- ☐ Low liquor residence time due to thin liquor film on tube walls
- ☐ No particle accumulation areas

Disadvantages of tube-type evaporators with black liquor inside of tubes:

- ☐ Requires a liquor recirculation pump
- ☐ Performance is sensitive to liquor flow distribution
- ☐ Limited to low ΔT (high ΔT would disrupt film)
- ☐ Liquor distribution tray is susceptible to plugging with fiber, scale, etc.
- ☐ Operation relies on extensive level control of liquids
- ☐ Foams easily at liquor dry solids content of 18% or less

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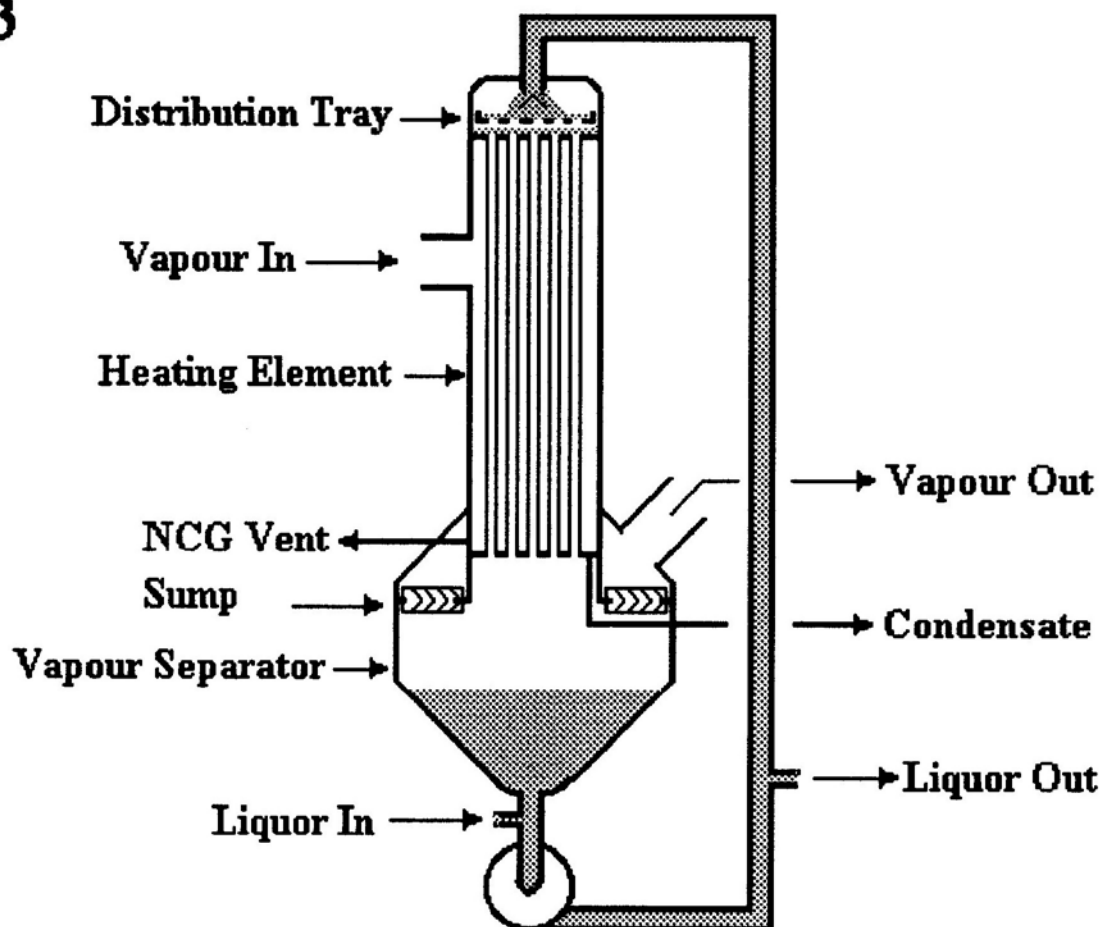


Figure 1-11. Diagram of a black liquor evaporator with liquor on the inside of tubes.

Figure __ shows a tube-type evaporator with liquor flowing on the inside of the heat transfer tubes, and internal vapor segregation to minimize environmental impact. The black liquor flow sequence and the vapor flow sequence are as follows.

1. Black liquor flow sequence:

- a. Black liquor enters at the bottom of the unit, into the liquor sump where it mixes with the more concentrated liquor in the sump.
- b. Black liquor in the sump is circulated to the liquor distribution lamella above the upper tube sheet.
- c. The distribution lamella distributes liquor into the top of each of the evaporator tubes.
- d. Liquor flows downward as a falling film on the inside of the tubes, where it is heated by steam condensing on the outside of the tubes. Water is evaporated from the liquor by convective boiling of the black liquor films within the tubes.
- e. Vapor and liquor are separated primarily in the lower liquor compartment. Final removal of remaining smaller liquor droplets is performed as the vapor exits via horizontal flow type droplet separators and flows to the next (higher number) effect.

2. Vapor and condensate flow sequence:

- a. Vapor from the previous effect enters the near the bottom of the heating shell. It rises and most of it is condensed on the evaporator tubes, heating the liquor and evaporating water from it within the tubes.
- b. As the condensate runs down the outside of the evaporator tubes, it is in contact with the up-flowing vapor, whereas it becomes increasingly depleted in volatile organic species such as methanol, etc., and exits from the bottom of the heating shell as a relatively much cleaner condensate.

- c. A small fraction of the vapor entering the heating shell is not condensed by now. The remaining vapor, enriched in methanol, etc. flows into an inner cylinder at the center of the evaporator, and condenses on separate evaporator tubes (the “foul zone”) inside the inner cylinder, heating the liquor in those tubes and evaporating water from it within the tubes.
- d. The relatively smaller flow of foul condensate collected within the inner cylinder exits the effect in a separate pipe via the bottom of the inner cylinder.

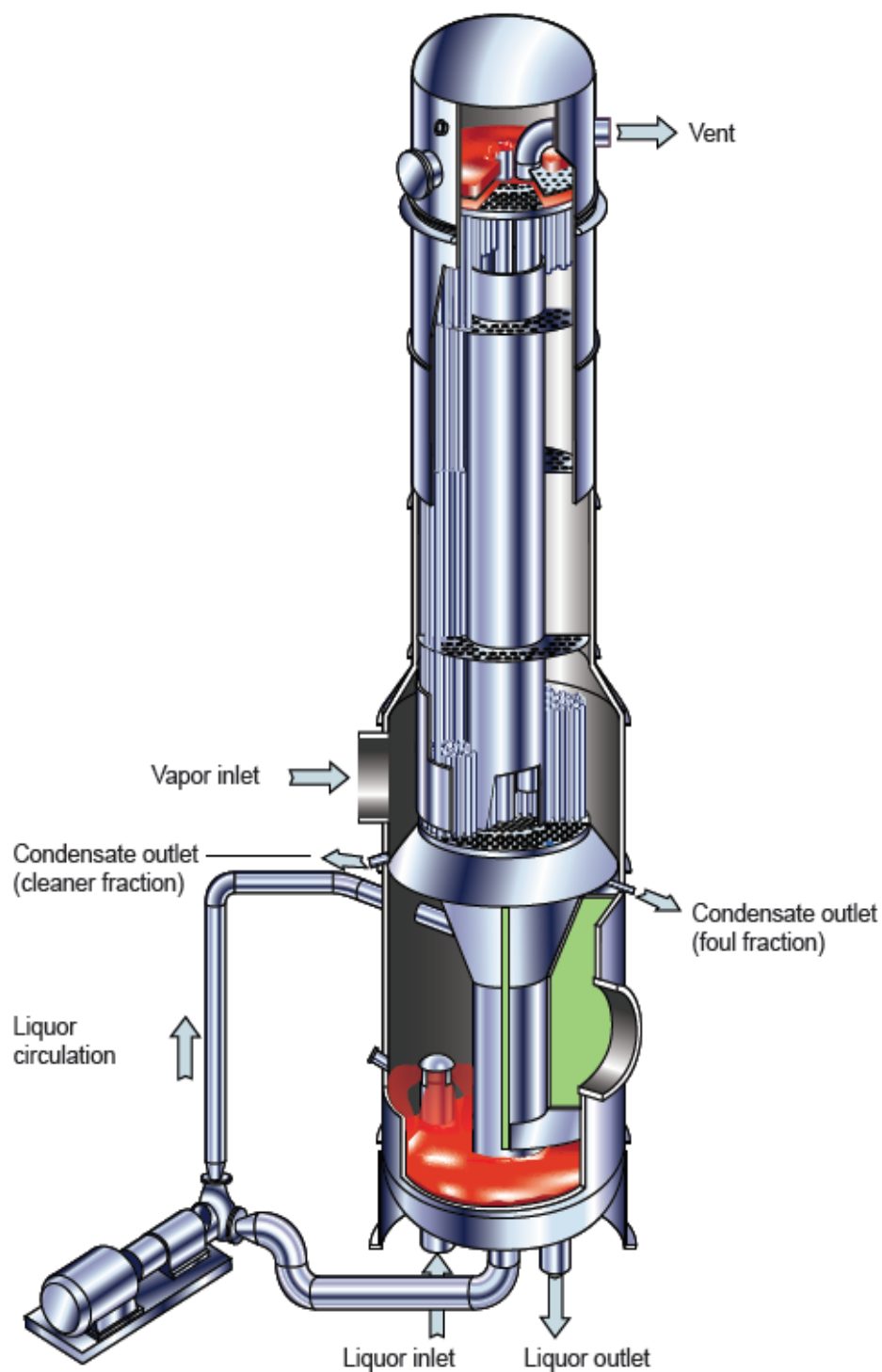
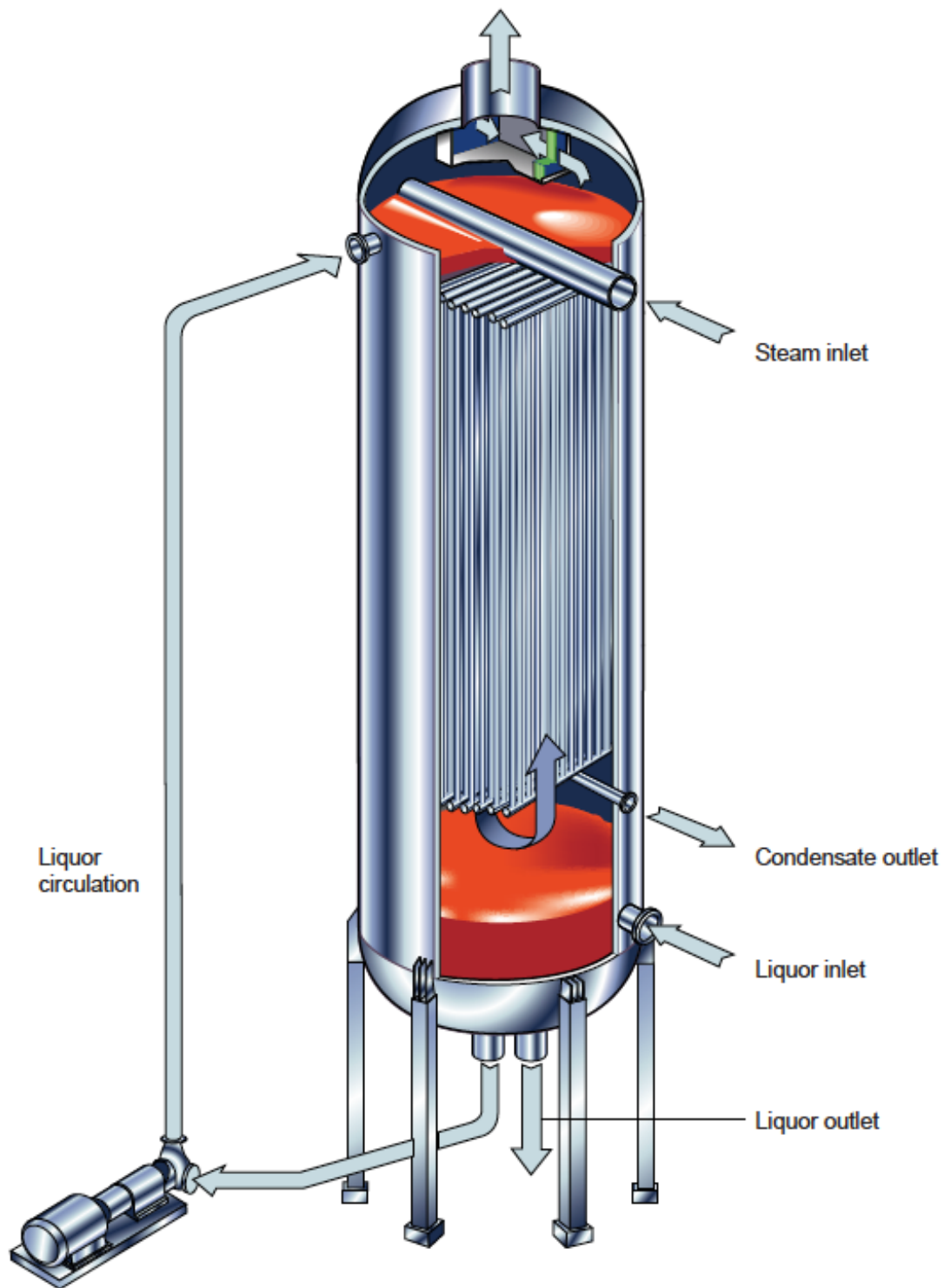


Figure 1-11. Diagram of a tube falling film black liquor evaporator with liquor on the inside of the tubes and with internal vapor separation.

3. TUBEL (Tube-Element) evaporator

- a. Black liquor enters at the bottom of the unit, into the liquor sump, where it mixes with the more concentrated liquor in the sump.
- f. Black liquor from the sump is circulated, via a liquor distributor, to the liquor distribution lamella above the tube elements.
- b. The distribution lamella distributes liquor onto the external surface of the tube elements.
- c. Liquor flows downward as a falling film on the outside of the tubes where it is heated by steam condensing on the inside of the tubes. Water is evaporated from the liquor by convective boiling of the black liquor films on the outer tube surfaces.
- d. Vapor and liquor are separated in the sump. Final removal of remaining smaller liquor droplets is performed as the vapor exits at the top of the evaporator, via horizontal flow type droplet separators, and flows to the next (higher number) effect.



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Figure 1-12. Diagram of a Tubel black liquor evaporator with liquor on the outside of tubes.

Forced Circulation Type Concentrators

Forced circulation concentrators were developed to minimize the crystallization scaling that has plagued LTV evaporators. They consist of heating elements, a flash tank, and large liquor recirculation pumps. The pressure within the heat exchangers is maintained high enough to prevent boiling within them. This minimizes crystallization and scaling of the heat transfer surfaces. Liquor to be concentrated is pumped into a much larger stream of recirculated liquor. The combined liquor stream circulated at high velocity through the concentrator tubes to reduce its viscosity through shear thinning and to enhance heat transfer rates. The hot liquor is flashed in the vapor head (flash tank) of the concentrator. Flashing concentrates the black liquor. Sodium salts crystallize mainly in the vapor head rather in the heat exchangers, and are removed with the concentrated product liquor. The vapor generated passes through mist eliminators before flowing to another effect or separate evaporator unit.

Forced circulation evaporators are designed with two heat exchanger loops connected to a common vapor body as shown in Figures 1-13 and 1-14. This allows for on-line washing of one heat exchanger while the concentrator remains in operation with the other heat exchanger online.

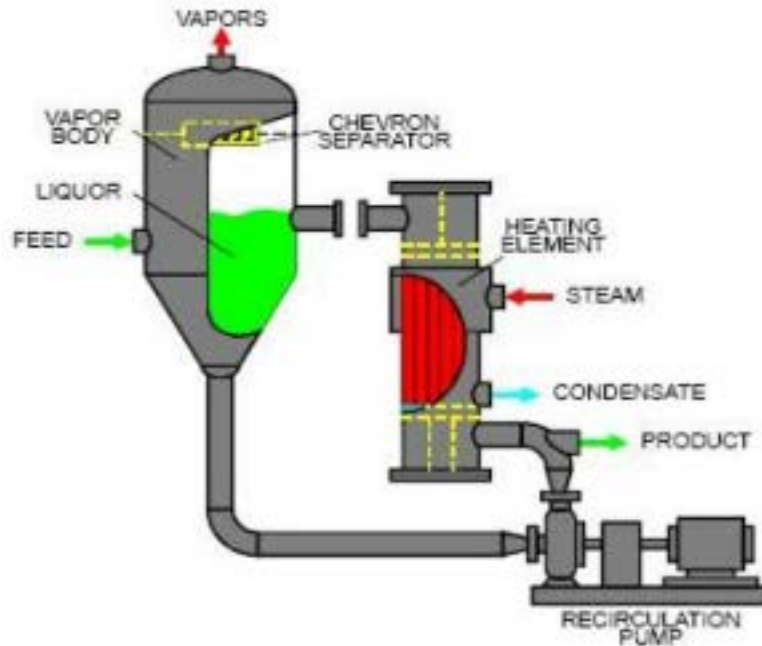


Figure 1-13. Diagram of a forced circulation concentrator with a heat exchange loop.^{vi}



Figure 1-14. Photo of a forced circulation concentrator with two heat exchange loops.^{vii}

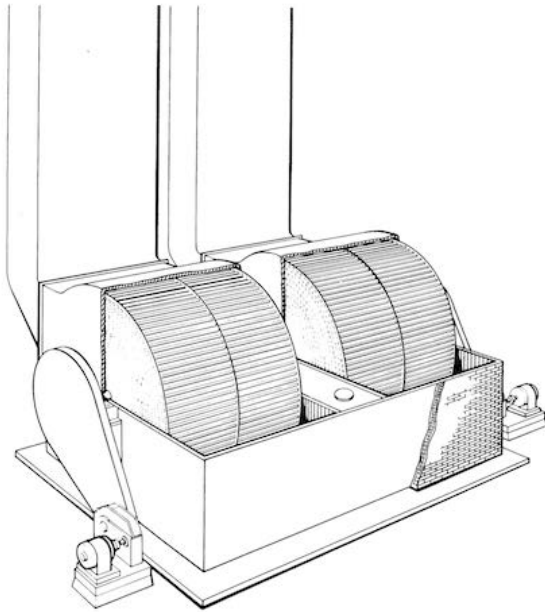
2.1.3 Direct-contact evaporators

This is older technology used to concentrate black liquor prior to the late 1960s, before indirectly heated black liquor concentrators became the standard. Direct-contact evaporators utilize hot flue gas from a recovery boiler as the heat source for evaporation. Black liquor at about 50 wt-% dry solids content is contacted directly by flue gas exiting the recovery boiler, concentrating it to about 65 wt-% dry solids content. Under typical conditions, heat removed from the flue gas to increase the black liquor solids content from 50 wt-% to 65 wt-% dry solids results in a 140°C (250°F) drop in flue gas temperature.^{viii} The upper dry solids content limit for direct contact evaporators is imposed by the viscosity of black liquor above 65 wt-% dry solids content. Direct contact evaporators operate at atmospheric pressure so the temperature of the black liquor cannot exceed boiling point of the black liquor at atmospheric pressure, typically 115°C.

Early on, direct contact evaporators were a major source of TRS emissions from kraft pulp mills. When hot acidic flue gas contacts black liquor, H₂S and other TRS species are released into the flue gas. TRS emissions limits were met by oxidizing the TRS compounds in black liquor to sodium thiosulfate and other non-odorous compounds by installing black liquor oxidation (BLOX) technology. This reduced TRS emissions to less than 100 ppm. With difficulty in meeting decreasing TRS emissions limits, many pulp mills converted from conventional recovery boilers and BLOX technology to modern recovery boilers that can operate with nearly undetectable TRS emissions.

Two types of direct-contact evaporators have been commonly used. One is the cascade evaporator illustrated in Figure 1-15. Liquor is lifted by rotating bundles of tubes held between two circular endlamellas. The tube bundle rotates through the black liquor tank at about 6 rpm. Hot flue gas flows across and between the tubes before exiting to the recovery boiler stack. Black liquor at about 50 wt-% dry solids content flows continuously into the liquor tank through which the tube bundle rotates. Concentrated black liquor at about 65 wt-% dry solids content is removed continuously from the tank through a screen to remove lumps.

Another type of direct contact evaporator is the cyclone evaporator illustrated in Figure 1-16. The cyclone itself is a cylindrical vessel with a conical bottom. Hot flue gas enters the cyclone through a tangential inlet near the bottom of the vessel. It flows in a helical path to the top of the vessel, and exits through a concentric outlet port. Black liquor droplets at about 50 wt-% dry solids content are sprayed across the flue gas inlet. Evaporation occurs as droplets are heated by the hotter flue gas. The cooled flue gas exits through the top of the vessel. The inertia of the black liquor droplets carries them to the vessel wall where they collect as a liquid film and flow downward along the wall to the conical bottom and into a black liquor sump below it. Additional black liquor is recirculated from the sump to maintain a continuously wetted vessel wall and prevent liquor dryout on the vessel walls.



*Figure 1-15. Cascade evaporator (From Grace, T.M., (1985) Chemical Recovery from Concentrated Liquor, Chapter 4 in **Chemical Recovery in the Alkaline Pulping Processes**, G. Hough, Editor, Tappi Press, Atlanta, pp 127.)*

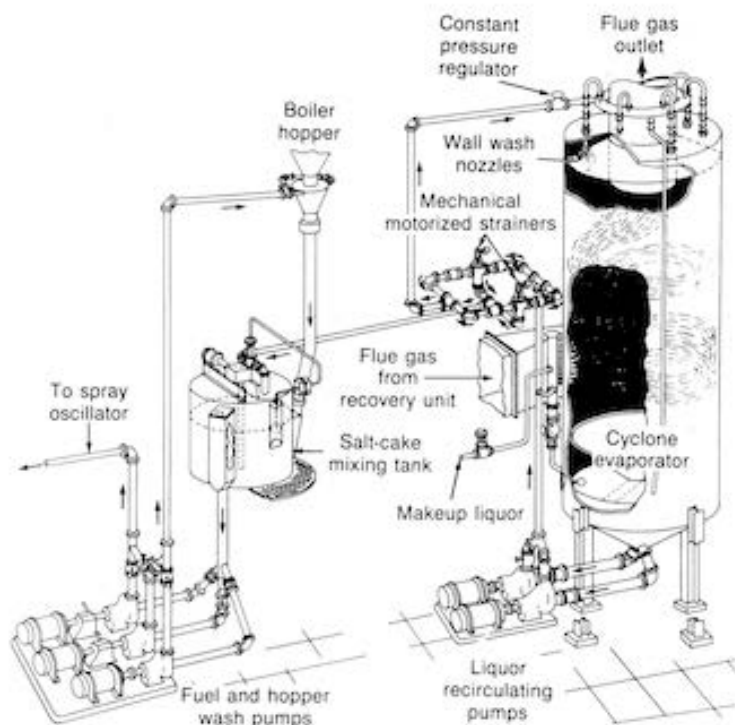


Figure 1-16 Cyclone evaporator (From Grace, T.M., (1985) *Chemical Recovery from Concentrated Liquor*, Chapter 4 in ***Chemical Recovery in the Alkaline Pulping Processes***, G. Hough, Editor, Tappi Press, Atlanta, pp 128).

Multiple-effect evaporator concepts and evaporator configurations

Black liquor evaporators use steam as a heating medium for heat transfer devices to evaporate water from black liquor. The water evaporated has about the same heat content as the steam used to evaporate it. The vapor generated is used as the heating medium for a second, lower pressure effect. When several effects are linked into multiple-effect evaporators, only the first effect receives live steam. The downstream effects are each heated by vapor generated in the previous effect. The pressure of the steam or vapor decreases from the effect where it is generated to the

next effect where it condenses. This cascading of heat continues until the pressure of the vapor generated in the last effect drops to about 75 kPa (22 in Hg vacuum). Vapor exiting the last effect is condensed in a sub-atmospheric pressure condenser. With proper design, evaporator effects are linked together with heat cascading from higher pressure effects to lower pressure ones. Reuse of heat in this way multiplies the total amount of evaporation accomplished by the evaporator set per kg of live steam to the first effect. A six-effect evaporator set may evaporate between 4.5 and 5.0 kg of water per kg of live steam. The ratio of the mass of water evaporated to mass of live steam used is commonly referred to as steam economy. Typically the steam economy for an evaporator set will be between 0.7 and 0.9 times the number of evaporator effects. A 6-effect evaporator set would operate with a steam economy between 4.2 and 5.4. The impact of operating parameters on steam economy is discussed in the material and energy balances section of this chapter.



Figure 1-17. A multiple effect evaporator set.

Many configurations of black liquor evaporators and concentrators have been used with the intent of improving both overall capacity and steam economy. A 6-effect black liquor evaporator set producing 50 wt-% dry solids black liquor and a separate black liquor concentrator producing 65 wt-% dry solids content product liquor was an early configuration. The number of effects in black liquor evaporators evolved to 7 or 8 effects, and now to at least one 9-effect set is operating. Black liquor concentrators became partially integrated with the evaporator set, with vapor from both a concentrator effect and an evaporator effect combined to drive the next lower pressure effect. Eventually, full integration of concentrator and evaporator trains came into use. The first one or two effects became the “concentrator” section of the combined set, concentrating black liquor from 50 wt-% to as high as 80 wt-% dry solids content.

Steam consumption

In an average Swedish non-integrated market pulp mill, the evaporation plant, including the stripper, accounts for one third of the total steam consumption. The total steam consumption in this average pulp mill is 15.5 GJ/ADt, of which 5.1 GJ/ADt is consumed by the evaporation plant.^{ix}

2.3 References

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2.2 Black Liquor Evaporator Mass & Energy Balances

H. Jeff Empie

Mass and energy balances form the basis for designing, evaluating, and troubleshooting multiple effect evaporator sets. They account for all mass and energy entering and leaving an evaporator set. They can also be applied to subunits such as a single body, a single effect, multiple connected effects, flash tanks, the surface condenser, etc. They must be constructed carefully so as to be error-free if they are to be of value. Accurate information on the design and operating conditions of an operating evaporator set or sets must be obtained before undertaking mass and energy balance calculations. Reliable estimates must be made for use in design of a new evaporator set. The balance calculations are relatively straightforward for a single line evaporator set, but become more complicated when evaporator, concentrator, and pre-evaporator units are interconnected.

The data required to perform these calculations are:

- ☐ **Definition of the calculation basis:** this is normally the throughput rate of dry black liquor solids;
- ☐ **Reference conditions:** temperature (normally 0°C/32°F) and pressure (normally 1.013 bar/14.7 psia)
- ☐ Temperature and dry solids content of weak black liquor feed
- ☐ Temperature and dry solids content of evaporator product liquor
- ☐ Pressure and temperature of live steam to the first evaporator effect
- ☐ Pressure drops from effect i vapor head to effect i+1 steam shell
- ☐ Vapor pressure in the condensate side of the surface condenser
- ☐ Heat transfer area in each effect, the surface condenser, and any intermediate liquor heaters

- Heat transfer coefficients for each evaporator body, intermediate liquor heaters, and the surface condenser.
- Input or output flow rates of any process streams such as tall oil soap that are input or removed after the weak liquor feed point and before the product liquor storage tank, and their temperatures and compositions
- Steam and NCG purge rates, temperatures and pressures

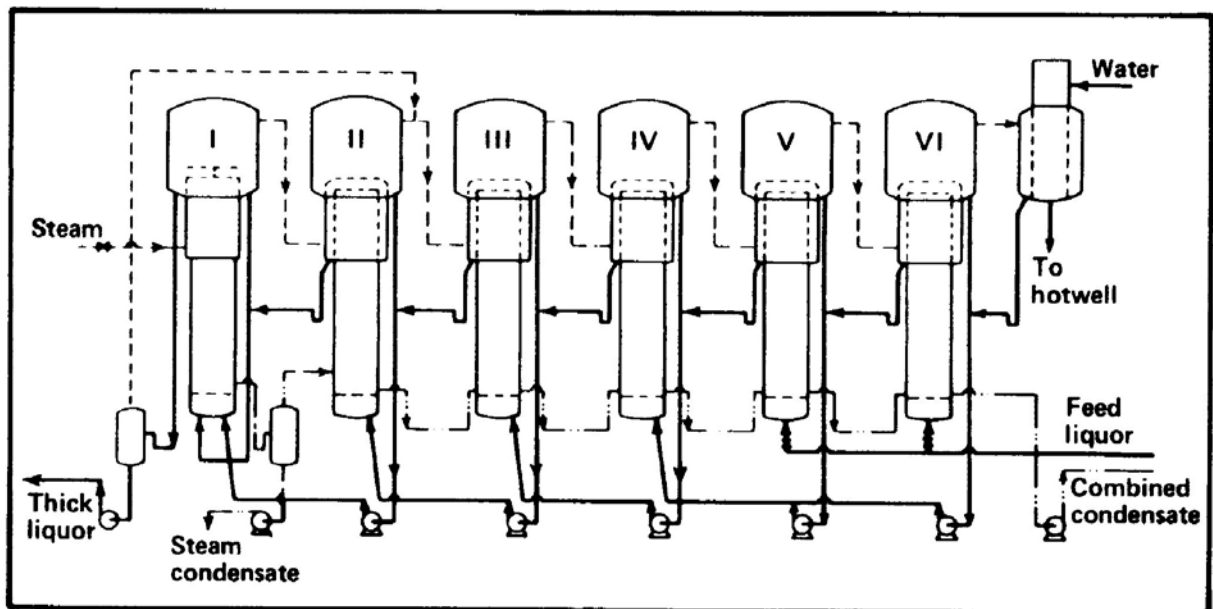


Figure 1. 6-effect evaporator, numbered to correspond to the example presented here.ⁱ

Mass and energy balances are calculated for each evaporator effect and liquor heater, flash tank, etc., and for the overall evaporator set.

Mass and Energy Balance Calculation Procedureⁱⁱ

A. Introduction

Black liquor evaporation is accomplished in both rising film (LTV) and falling film (FF) multiple-effect evaporators whereby sensible heat is transferred to the black liquor, followed by latent heat transfer at boiling. This is accomplished by condensing steam or vapor on one side of a tubular or flat plate heat transfer surface. For saturated steam or vapor, the condensation temperature is fixed by the pressure in the shell or vapor chamber of the heat exchanger. This temperature remains essentially constant over all of the heat transfer surface in the effect. On the other hand, as heat from the condensing steam is absorbed by the liquor, its temperature rises until the boiling point is reached and evaporation commences. In multiple-effect systems, the vapor generated is used in the next effect as the heat source for evaporation in that unit. This can be viewed as a reuse of the latent heat of vaporization of water.

B. Mass and Energy Effect Balances for Single Effect

Calculation of fundamental mass and energy balances for an evaporator effect must recognize that the steam side and liquor side have to be dealt with separately because the two never come into direct contact with each other. Referring to Figure 1 (shown here as a LTV unit), and using the nomenclature below, the mass balances are straightforward (valid for FF unit also). Taking first the shell (steam) side and then the tube (liquor) side:

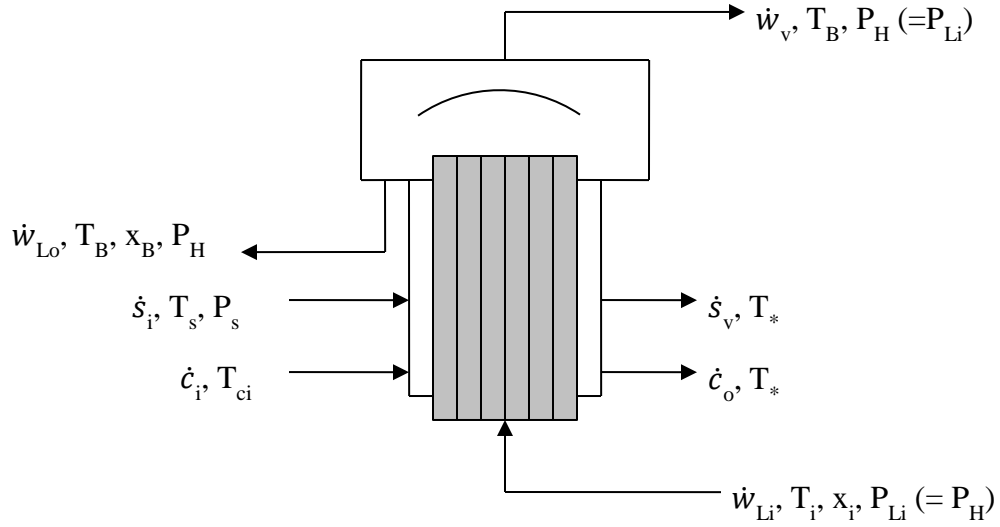


Figure 1. Single Effect

Using the following nomenclature:

P_{Li} = black liquor pressure in

P_H = pump head pressure ($= P_{Li}$)

P_s = steam pressure

P_{ref} = reference pressure

\dot{w}_{Li} = weak BL flow rate in

\dot{w}_{Lo} = concentrated BL flow rate out

\dot{w}_v = vapor flow rate out

\dot{s}_i = steam rate in

\dot{s}_v = vent loss

\dot{c}_i = condensate rate in

\dot{c}_o = condensate rate out

x_i = wt. fr. black liquor solids (BLS) in

x_o = wt. fr. BLS out

T = saturation temperature at P_s

T_B = BL boiling point@ P_H

T_{ci} = condensate temperature in

T_s = steam temperature in

T_i = BL inlet temperature

T_{cond} = temperature of final condenser

c_{pw} = specific heat of condensate

c_{ps} = specific heat of steam

c_{pLi} = specific heat of black liquor in

c_{pLo} = specific heat of black liquor out

λ_B = heat of vaporization @ P_{ref}

λ_s = heat of vaporization @ P_s

\dot{q}_T = heat transfer rate to BL

\dot{q}_L = heat lost

BPR = black liquor boiling point rise

Mass Balances:

$$\text{Shell (Steam) Side:} \quad \dot{s}_i + \dot{c}_i = \dot{c}_o + \dot{s}_v \quad (1)$$

$$\text{Liquor Side: (Total)} \quad \dot{w}_{\text{Li}} = \dot{w}_{\text{Lo}} + \dot{w}_v \quad (2)$$

$$\text{(BL Solids)} \quad \dot{w}_{\text{Li}} x_i = \dot{w}_{\text{Lo}} x_o \quad (3)$$

Energy Balances:

Shell (Steam) Side:

$$\dot{c}_i c_{\text{pw}} (T_{\text{ci}} - T^{\square}) + \dot{s}_i [c_{\text{ps}}(T_s - T^{\square}) + \lambda_s] - \dot{s}_v \lambda_s = \dot{q}_T + \dot{q}_L \quad (4)$$

But \dot{s}_v and $\dot{q}_L \approx 0$, and the sensible heat of vapor is small ($c_{\text{ps}} \approx \frac{1}{2} c_{\text{pw}}$, $T_s \approx T^{\square}$) and can be neglected. With these substitutions, Eq.(4) becomes:

$$\dot{c}_i c_{\text{pw}} (T_{\text{ci}} - T^{\square}) + \dot{s}_i \lambda_s = \dot{q}_T \quad (5)$$

Liquor Side:

$$\begin{aligned} & \dot{W}_{Li} c_{pLi} (T_i - T_{ref}) - \dot{W}_{Lo} c_{pLo} (T_B - T_{ref}) \\ & - \dot{W}_v [c_{ps}(T_B - T_{ref}) + \lambda_B] + \dot{Q}_T = 0 \end{aligned} \quad (6)$$

But: $c_{ps}(T_B - T_{ref}) \ll \lambda_B$ and

$$\dot{W}_{Lo} = \dot{W}_{Li} - \dot{W}_v$$

Hence Eq.(6) becomes:

$$\begin{aligned} & \dot{W}_{Li} c_{pLi} (T_i - T_{ref}) - (\dot{W}_{Li} - \dot{W}_v) c_{pLo} (T_B - T_{ref}) \\ & - \dot{W}_v \lambda_B + \dot{Q}_T = 0 \end{aligned} \quad (7)$$

If our energy balance reference temperature is chosen to be at the boiling point of the black liquor, then $T_B = T_{ref}$ which forces λ_B to be evaluated at the liquor boiling temperature, T_B , at the saturation pressure, $P_B (= P_{ref})$. With these substitutions, Eq.(7) simplifies to:

$$\dot{W}_{Li} c_{pLi} (T_B - T_i) + \dot{W}_v \lambda_B = \dot{Q}_T \quad (8)$$

Balance Eqs. (5) and (8) can be used to help diagnose the operating performance of an individual effect, based on measurable operating variables. These are the steam (vapor) temperature, pressure, and feed rate for an effect, along with the black liquor feed rate, temperature and solids, and liquor product rate, temperature and solids. Black liquor physical properties, namely specific gravity and heat capacity, are calculated from standard correlations included in Chapter 3. Latent heat of vaporization and specific volume for water at saturation conditions are derived from the standard Steam Tables. One practical example of diagnosing operating performance might involve calculating the Steam Economy (weight of water evaporated per weight of steam (vapor) used) of an individual effect. This should normally fall in the range of 0.75 to 0.95. A calculated value greater than 1.0 is generally an indication that the feed liquor temperature to that

effect is too high and flashing is occurring as the liquor enters the effect in question; a value lower than this range would indicate the feed temperature is too low or that significant fouling is occurring. This type of analysis can be carried out using a spreadsheet calculation.

C. Mass and Energy Balance Coupling between Effects

Design of a multiple effect evaporator set requires computing mass and energy balances for all the effects, along with the rates of heat transfer in each. This is difficult because all of the intermediate stream temperatures and pressures are not known apriori. Instead the balances are coupled between the effects and must be solved for by trial and error. The governing balances for the j^{th} and $(j-1)^{\text{th}}$ effects are shown below in Figure 2:

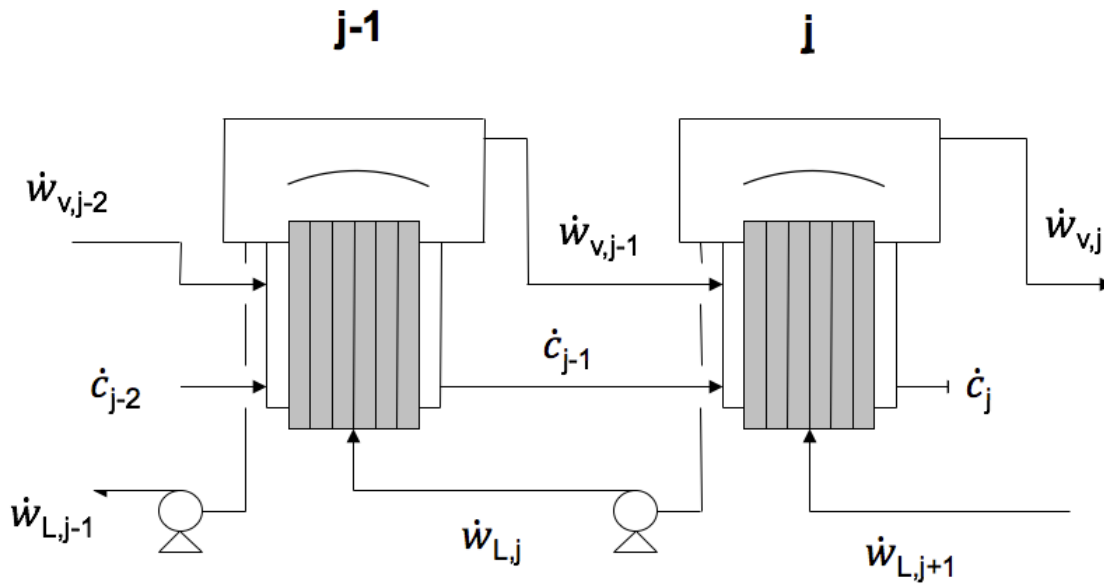


Figure 2 Coupling between Effects

Applying Eq.(8) to the $(j - 1)^{\text{th}}$ effect, BL side:

$$\dot{q}_{j-1} = \dot{w}_{L,j} C_{PLj} (T_{B,j-1} - T_{B,j}) + \dot{w}_{v,j-1} j-1 \quad (9)$$

Applying Eq. (5) to the j^{th} effect, steam side:

$$\dot{q}_j = c_{j-1} c_{PW} (T_{j-1} - T_j) + \dot{w}_{V,j-1,j-1} \quad (10)$$

Eliminating $\dot{w}_{V,j-1,j-1}$ from these two equations:

$$\dot{q}_{j-1} - \dot{w}_{L,j} c_{PLj} (T_{B,j-1} - T_{B,j}) = \dot{q}_j - c_{j-1} c_{PW} (T_{j-1} - T_j)$$

or

$$\dot{q}_j = \dot{q}_{j-1} + c_{j-1} c_{PW} (T_{j-1} - T_j) - \dot{w}_{L,j} c_{PLj} (T_{B,j-1} - T_{B,j}) \quad (11)$$

From Eq. (11), we see that the heat duty in any effect equals the heat duty in the preceding effect with two sensible heat corrections:

- \dot{q}_j reduced by the sensible heat demand of the BL in the j^{th} effect
- \dot{q}_j increased by the sensible heat in the condensate from the $(j-1)^{\text{th}}$ effect

In the common backward flow MEE configuration, BL sensible heat is more important at low number (i.e. high T) effects (because of higher BPR at higher BL concentration), while the condensate term is more important in the high number effects where c_j gets large. Thus Q_j starts out high, then decreases, then increases. In general, $\dot{q}_j = \dot{q}_{\text{avg}} \pm 20\%$.

The \dot{q}_j 's are determined by heat balances which are coupled with heat transfer rates from steam/vapor to liquor. The rate of heat transfer, \dot{q} , is traditionally expressed as:

$$\dot{q} = U A \Delta T_d$$

where U is the overall heat transfer coefficient and A is the surface area for heat transfer. ΔT_d is the temperature driving force for heat transfer calculated, by convention, as the difference between the steam condensation temperature and the product black liquor

boiling temperature. The overall temperature difference between the steam and liquor flows in an effect is variable and difficult to predict. In a multiple effect system, the mass and energy balances, along with the heat transfer rate equations ($\dot{q}_j = U_j A_j T_j$) specify the entire system, subject to the temperature constraint:

$$T_j = T_s - T_{\text{cond}} - \text{BPR}_j \quad (12)$$

This involves a trial and error calculation so that the individual mass and energy balances, which need temperature and pressure specified so that enthalpies and black liquor boiling points can be evaluated, can be rigorously satisfied.

The boiling point rise (BPR) is a key factor because the vapor generated from boiling black liquor at T_B and P_H becomes the heat source for the next effect, condensing at the boiling temperature of water at that pressure. This temperature difference between the vapor generation temperature and water condensation temperature is the BPR [Note the definition of BPR: $\text{BP}_{\text{BL}} - \text{BP}_{\text{water}}$ at a given pressure], and is therefore lost for driving heat transfer in that next effect. This BPR temperature loss is experienced in each effect not fed with fresh steam. Hence, the overall temperature driving force for the multiple effect evaporator system, $T_o (= T_j)$, is diminished by the sum of the individual effect BPRs as given by Eq.(12).

D. Optimizing Operation

Optimum multiple effect evaporator operation involves a tradeoff between capacity and steam economy. Increasing the number of effects raises the steam economy, but a (N+1)-effect system with the same total heat transfer surface area as a N-effect system will have a lower evaporation capacity. This is because the T_o is decreased by the higher BPR losses (c.f.; Eq(12)) with the (N+1)-effect system. This can be shown

quantitatively by calculating the \dot{q}_T for (N+1)- vs. N-effect systems with the same total surface area (A). Assuming constant and equal U-values (U_o),

$$\dot{q}_{N+1} = U_o T_{o,N+1} A_j / (N+1) = (U_o A_j) T_{o,N+1} / (N+1)$$

$$\dot{q}_N = U_o T_{o,N} A_j / (N) = (U_o A_j) T_{o,N} / (N)$$

But

$$T_{o,N+1} = T_s - T_{\text{cond}} - \sum_{j=1}^{N+1} BPR_j$$

Therefore $T_{o,N+1} < T_{o,N}$, which forces $\dot{q}_{N+1} < \dot{q}_N$.

One final concept of importance concerns maintaining an even distribution of heat duties in each effect in a multiple-effect system. To illustrate the impact of this, we define a Heat Duty Distribution Factor, γ_j , as

$$\gamma_j = \dot{q}_j / \dot{q}_{\text{avg}}$$

where $\dot{q}_{\text{avg}} = \dot{q}_{\text{Total}} / N = \text{constant}$

Therefore: $\gamma_j = N \dot{q}_j / \dot{q}_{\text{Total}}$

Note: $\dot{q}_{\text{avg}} = \dot{q}_j / \gamma_j = \text{constant}$

Therefore: $\dot{q}_{\text{avg}} = U_j A_j T_j / \gamma_j = U_1 A_1 T_1 / \gamma_1 = U_2 A_2 T_2 / \gamma_2 = \dots$ (13)

which makes: $T_2 = [U_1 A_1 T_1 / U_2 A_2] [\gamma_2 / \gamma_1]$

But:

$$\begin{aligned}
 T_o &= \sum_{j=1}^N T_j \\
 &= T_1 + [U_1 A_1 T_1 / U_2 A_2] [\gamma_2 / \gamma_1] + [U_1 A_1 T_1 / U_3 A_3] [\gamma_3 / \gamma_1] + \dots \\
 &= [U_1 A_1 T_1 / \gamma_1] [\gamma_1 / U_1 A_1 + \gamma_2 / U_2 A_2 + \gamma_3 / U_3 A_3 + \dots + \gamma_N / U_N A_N]
 \end{aligned}$$

Therefore:

$$\begin{aligned}
 \dot{q}_{\text{Total}} &= N \dot{q}_{\text{avg}} = N \dot{q}_j / \gamma_j \\
 &= [N U_1 A_1 \Delta T_1] / \gamma_1 = N T_o / [\gamma_1 / U_1 A_1 + \dots + \gamma_N / U_N A_N]
 \end{aligned}
 \tag{14}$$

$$\dot{q}_{\text{Total}} = [N (T_s - T_{\text{cond}} - \sum_{j=1}^N BPR_j) / \sum_{j=1}^N (\gamma_j / U_j A_j)]
 \tag{15}$$

Eq.(15) implies that the capacity of the multiple-effect unit determined by \dot{q}_{Total} is a function of not only N , BPR_j , T_s , T_{cond} , U_j , and A_j , but also the Heat Duty Distribution Factors (γ_j).

From Eqs.(13 and 14):

$$\Delta T_j / \Delta T_o = (\gamma_j / U_j A_j) / \sum_{j=1}^N (\gamma_j / U_j A_j)
 \tag{16}$$

The important implication of Eq.(16) is that the ratio of the heat transfer resistance of an individual effect to the total resistance for the multiple effect system is the same ratio of the ΔT for the individual effect (ΔT_j) to the total ΔT for the system (ΔT_o). Hence, a relatively large individual resistance can be totally

controlling for the system. This implies that we should keep the individual resistances approximately equal and that higher surface area should be employed in effects where the U-value is low. This would generally apply to higher solids effects where liquor viscosity is higher and scaling is more prevalent.

D. A Complete Mass and Energy Balance for an LTV Evaporator Set

The following tables contain a set of data required for a complete mass and energy balance for an LTV evaporator set, and results of the balance calculations.ⁱⁱⁱ

Table 1. Input data for mass and energy balance calculations for a 6-effect evaporator set.

Feed rate: 5.823 kg/s dry black liquor solids

Feed dry solids content: 13.9 wt-%

Product liquor dry solids content: 52.0 wt-%

Steam conditions to the first effect: 3.185 bar, 135.6 °C

Surface condenser vapor pressure and saturation temperature: 100.4 mm Hg, 51.7 °C.

Table 2. Heat transfer surface areas and heat transfer coefficients by effect.

Effect #	A, m ²	U, J/m ² s K
IA	409	981
IB	409	1230
II	818	2229
III	818	2186
IV	818	1792
V	818	1363
VI	818	1076

Table 3. Data set for mass and energy balance calculations for a 6-effect LTV black liquor evaporator set; example heat balance. (From Venkatesh and Nguyen, p. 22-23, in *Chemical Recovery in the Alkaline Pulping Process*, R.P. Green and G. Hough Editors, Tappi Press Atlanta, 1992

Effect	Flash tank	IA	IB	II	III	IV	V	VI
Steam pressure, bar		3.19	3.19	1.66	1.22	0.72	0.45	0.25
mm Hg vacuum						211	411	559
Steam temperature, °C		135.6	135.6	114.4	102.2	91.1	79.4	66.7
T, °C		13.3	14.4	7.2	7.2	8.3	9.4	13.3
Liquor temperature, °C	111.1	122.2	121.1	107.2	95.0	82.8	70.0	52.2
Boiling point rise, °C	7.8	7.2	6.1	3.9	2.8	1.7	1.7	1.7
Saturated vapor temperature, °C	103.3	115.0	115.0	103.3	91.1	80.6	68.3	51.7
Vapor pressure, bar		1.69	1.69	1.14	0.75	0.47	0.28	0.12
mm Hg vacuum					188	396	548	660
Heat of vaporization, kJ/kg		2217	2217	2249	2277	2307	2338	2379
Pressure drop between effects, bar		0.028	0.028	0.041	0.030	0.020	0.020	
mm Hg					23	15	15	
Next effect vapor pressure, bar		1.66	1.66	1.10	0.72	0.45	0.25	
mm Hg vacuum					165	381	533	
Feed rate, kg/s	11.42	13.83	17.11	22.54	27.67	31.88	20.95	20.95
Discharge rate, kg/s	11.24	11.42	13.83	17.11	22.54	27.67	16.44	15.44
Total dry solids content in, wt-%	51.2	42.3	34.2	25.9	21.1	18.3	13.9	13.9
Total dry solids content out, wt-%	52.0	51.2	42.3	34.2	25.9	21.1	17.8	18.9
Condensate, kg/s	0.18	2.42	3.28	5.42	5.13	4.21	4.51	5.51

		Vapor q_{vl} ,	Evaporation
	kJ/s	kJ/kg	kg/s
Steam to 1A	5,402		
Heating feed to IA			
(13.98 kg/s)(122.2-121.1°C)(3.31 kJ/kgK)	50.9		
Evaporation 1A	5,352	2,217	2.41
Steam to 1B	8,090		
Heating feed to IB			
(17.11 kg/s)(13.9C)(3.45)	821		
Evaporation IB	7,270	2,217	3.28
From IA	5,352		
Steam condensate flash from IA, IB			
(2.570+3.868)kg/s (20.56C)(4.18kJ/kgK)	553		
Heat to II	13,175		
Heating feed to II			
(22.54 kg/s)(12.2C)(3.53 kJ/kgK)	971		
Evaporation II	12,204	2,249	5.43
Condensate flash from II	318		
Flash liquor from IA	171		
Heat to III	12,694		
Heating feed to III	1,229		
(27.67 kg/s)(12.2C)(3.64 kJ/kgK)			
Evaporation III	11,692	2,277	5.13
Condensate flash from III	537		
Heat to IV	12,229		
Heating feed from V	789		
(16.44 kg/s)(12.8C)(3.75kJ/kgK)			

Heating feed from VI	1,708		
(15.31kg/s)(29.4C)(3.79kJ/kgK)			
Evaporation VI	9,732	2,307	4.22
Condensate flash from V, VI	815		
Heat to V	10,546		
Feed flash to V	89.8		
(20.98 kg/s)(1.1C)(3.89kJ/kgK)			
Evaporation V	10,636	2,338	4.55
Condensate flash	1,117		
Heat to IV	11,754		
Feed flash to VI	1453		
(20.98 kg/s)(17.8C)(3.89kJ/kgK)			
Evaporation VI	13,206	2,379	5.55
Total evaporation			30.57
Live steam consumed			6.44
Steam economy			4.75

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- ⁱⁱⁱ Venkatesh, V., Nguyen, X.N., Evaporation and Concentration of Black Liquor, Chapter 2 in *Chemical Recovery in the Alkaline Pulping Processes, Thir Edition. R.P. Green and G. Hough, Editors, TAPPI Press, Atlanta* (1992).

2.3. Design principles for black liquor evaporation

1. Evaporator Capacity and Steam Economy

The black liquor processing capacity of an evaporator set is equivalent to the rate at which it can remove water from black liquor. The capacity for water removal is equivalent to the rate at which heat is transferred from steam to the black liquor, or

$$\text{CAPACITY} = \text{HEAT TRANSFER RATE}$$

Defining capacity is that simple. Evaporator designers and operators must know this relationship and know how to apply it. Applying it, however, is often more difficult than defining it. The next sections of this chapter deal with how to apply it.

Steam economy and evaporation capacity (rate) are two important factors that are considered when designing black liquor evaporators and concentrators. New evaporator installations are designed to match a pulp mill's requirement for concentrating the volume of black liquor generated to the desired total dry solids content for firing. Meeting the pulp mill's evaporation demand today is a greater challenge with older evaporator sets that were likely designed for different weak black liquor feed rates and feed dry solids contents, total evaporation loads and product dry solids contents, and black liquor with different physical properties. Existing older evaporator sets often need to be optimized to meet new black liquor processing requirements. Optimization alone of an existing evaporation plant may not be sufficient to meet current or future mill needs, so that hardware to increase evaporation capacity may need to be installed. Changes in collection of methanol and turpentine as high value products, and management of NCGs and condensates must be dealt with as well.

The factors that impact steam economy and capacity of multiple effect black liquor evaporators are:ⁱ

Steam economy (in decreasing importance):

- a. number of effects
- b. feed liquor temperature
- c. steam pressure to first effect
- d. dry black liquor solids content of the feed and product liquors from the evaporator set
- e. the number of stages of liquor and condensate flash, and the temperature of the discharged liquor and condensate
- f. the liquor flow sequence within the evaporator set

Capacity (evaporation rate):

- a. total heat transfer surface area
- b. available temperature drop
- c. overall heat transfer coefficients in each effect

The capacity of an evaporator set is determined by a complicated interaction of these variables. This issue is discussed in more detail later in this section.

2. Flow Characteristics in Evaporators and Concentrators

The amount of heat transfer surface required in an evaporator set depends in part on the flow behavior black liquor in contact with the heat transfer surface. Evaporator bodies are designed for one of these three types of flow:

1. climbing (rising) film
2. falling film
3. forced circulation

The first two types operate with vertical, two-phase flow. In these units, water is evaporated mainly by convective boiling as the liquor is heated. Vapor is generated along the liquor flow path, with evaporation taking place at the liquid-vapor interface. Forced circulation evaporators operate with single-phase flow. The

pressure in forced circulation evaporators is kept high enough to prevent boiling within the heating element. Vapor is generated when the hot liquor flashes as it enters a vapor head through a pressure reduction valve.

The rates of heat transfer evaporation are determined in part by the vapor-liquor flow behavior. Vapor and liquid flows are coupled through the momentum of each phase as vapor and liquid move together through the heating element of the evaporator. In climbing film LTV evaporators, for example, the vapor generated during evaporation pushes and drags black liquor upward to the exit of the evaporator tubes, operating against the effect of gravity that pulls it back downward. Similarly, the liquor acts as a drag on rising bubbles and slugs of vapor, shaping and compressing them and creating distinctive flow patterns within the two-phase flow. Boiling increases the total vapor flow rate from zero at the tube entrance. The geometry of the flow as imposed by the flow conduit or surface also plays a role.

The mass ratio of vapor to liquid flowing upward in the heat transfer tubes of a typical climbing film LTV evaporator is between 0.15 and 0.32 (Table 1). The vapor to liquid volume ratios are much higher, from more than 500 in the first effect to more than 20,000 in the 6th effect (Table 1). These values reflect the huge change in specific volume of water vapor from the high-pressure conditions in the first effect to the sub-atmospheric conditions in the 6th effect. These are the ratio of volumetric flow rates for vapor and black liquor exiting the tubes for each of the six effects in a 6-effect climbing-film LTV evaporator:

Table 1. Mass flow and volume flow ratios for vapor and black liquor in each of the six effects of a typical LTV evaporator set.

Effect #	6	5	4	3	2	1
Pressure, bar	0.13	0.27	0.46	0.72	1.08	1.64
Liquor in, kg BLS/kg BL	0.139	0.183	0.211	0.259	0.342	0.423
Liquor out, kg BLS/kg BL	0.183	0.211	0.259	0.342	0.423	0.512
Flow rate ratios at tube exit:						
Vapor to BL volume ratio, m ³ /m ³	21,000	4,230	3,310	2,390	1,020	544
Vapor to BL mass ratio, kg/kg	0.317	0.153	0.227	0.320	0.237	0.210

Flow regime maps for multiphase flow (vapor-liquid flow) are useful representations for flow visualization. They have been used as tools to understand better a variety of 2-phase and multi-phase flow applications.^{ii,iii}

Flow maps are approximate. The boundaries represent transitions between regions. The actual boundary locations change with tube diameter and with variables such as surface roughness, and, for multicomponent fluids, even small differences in composition. Most were developed from data for lower viscosity fluids, although viscosity is included as a variable within the correlations. Even with these limitations, flow maps can be useful in understanding flow behavior in convective heat transfer equipment such as black liquor evaporators.

A flow regime map for vertical flow of black liquor and vapor under one set of conditions is illustrated in Figure 1. The flow regime boundaries are plotted on axes of the superficial vapor momentum flux versus the superficial liquid momentum flux, defined as

$$\text{Superficial vapor momentum flux} = [G_x]^2/\rho_g$$

$$\text{Superficial liquid momentum flux} = [G(1-x)]^2 r_1$$

where G = mass velocity, x = vapor quality, and ρ = density .

The boundaries between regions of the map were calculated using correlations for each for vertical flow in conduits as developed by Hewitt and Roberts⁷ and included in Collier. included in Collier⁵. Figure 4 shows that black liquor flow falls within the churn region in climbing film LTV evaporators. Lavery observed this same slug-churn flow behavior with black liquor in a pilot LTV evaporator.^{iv}

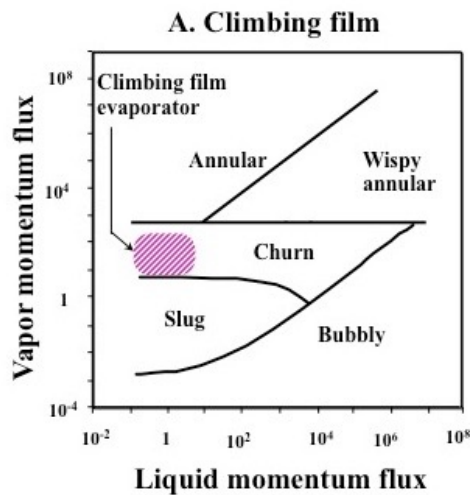


Figure 1. Flow map for black liquor in a climbing film LTV evaporator for black liquor at 50 wt-% dry solids content, 100°C.

3. Descriptions of flow patterns in vertical, co-current vapor-liquid flow

The five flow patterns illustrated in Figure 2 are encountered in two-phase vertical upward co-current flow in LTV evaporators, vertical tubes boilers, and other similar devices. The flow patterns are described here.

Bubbly flow: the vapor bubbles, much smaller than the tube diameter, are distributed as discrete bubbles in a continuous liquid phase.

Slug flow: bubbles are nearly the diameter of the tubes that contain them. A slowly descending film of liquid separates the bubbles from the tube wall.

Churn flow: formed when large bubbles in slug flow begin to break down. The vapor flows in a chaotic manner through the liquid that is mainly displaced to the tube wall. The flow has an oscillatory character.

Wispy annular flow: the liquid forms a thick film on the tube wall, with a considerable amount of liquid entrained as droplets in the center core. The droplets agglomerate into long, irregular filaments.

Annular flow: a liquid film forms at the tube wall, with a central vapor core. Large amplitude coherent waves appear on the liquid film surface. Continuous breakup of the waves generates entrained droplets in the central core flow.

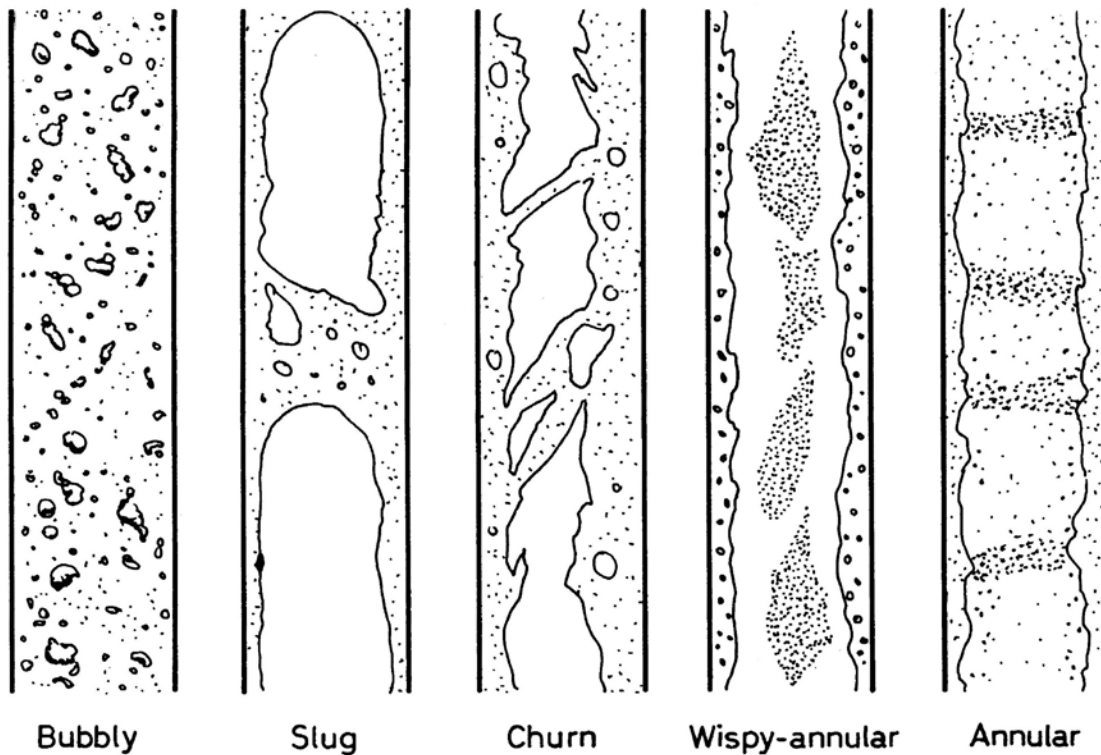


Figure 2. Flow patterns in rising film two phase flow.^v

The range of void fractions, measured experimentally with air-water systems³ are:

Bubbly flow: 0.057-0.20

Slug flow: 0.28-0.54

Froth flow: 0.43-0.7

Falling film flow: 0.80-0.94

Annular flow: 0.81-0.94

Climbing film LTV evaporators. Liquor enters an LTV body from a liquor plenum in the bottom. Flow is single phase liquid for the first meter or more, until the liquor begins to boil and generate vapor. The rising vapor pushes and drags black liquor upward with it. This transports the liquor to the top of the tube and into the vapor head at the top of the tube bundle. There, liquor and vapor are separated.

Two-phase flow in a climbing film evaporator is chaotic. Liquor is transported upward in the center of the tube, and gravity pulls it downward along the tube wall, creating a large back flow. Large slugs of vapor formed from coalescing vapor bubbles rise unsteadily, creating an unstable, chaotic flow. As heat is transferred, the vapor content increases in each tube, and then the liquor and vapor exit the top of the tube as if they are burped out. The tube then refills with liquid, mainly by back flow of liquor collected in the vapor head. Different tubes within the tube bundle eject liquor at different times, and cyclic pressure fluctuations for each tube have been observed. Two important consequences are that the flow pattern within any one tube is unsteady, and the liquor concentration within a tube varies with time. Because of this, tubes that begin to plug with scale will plug faster than adjacent clean tubes because the liquor eject-refill cycle in the scaling tubes is hindered.

Heat transfer in climbing film LTVs is from the tube walls to the liquor adjacent to the walls. The evaporation mechanism is convective boiling, with water evaporating from the liquor-vapor interface of liquor on the tube walls.

The flow map in Figure 1 shows the regions of flow for two-phase vertical up flow as in climbing film LTV black liquor evaporators. The axes represent the superficial momentum fluxes for the vapor and liquid respectively, with superficial velocities calculated for the vapor only or liquid only as their respective volumetric flow rates divided by the tube cross section. The solid lines in the map are boundaries between different flow regions.⁵ The boundaries in Figure 1 were calculated specifically for black liquor at 100°C, 50 wt-% dry solids content.^{vi} Flow maps can be created for other vapor-liquid systems using the method developed by Hewitt and Roberts.^{vii,5}

The flow region for black liquor in climbing film LTVs lies in the churn flow region at low liquor superficial velocities and moderate vapor superficial velocities. This slug-churn flow behavior has been observed in a pilot LTV evaporator.⁴

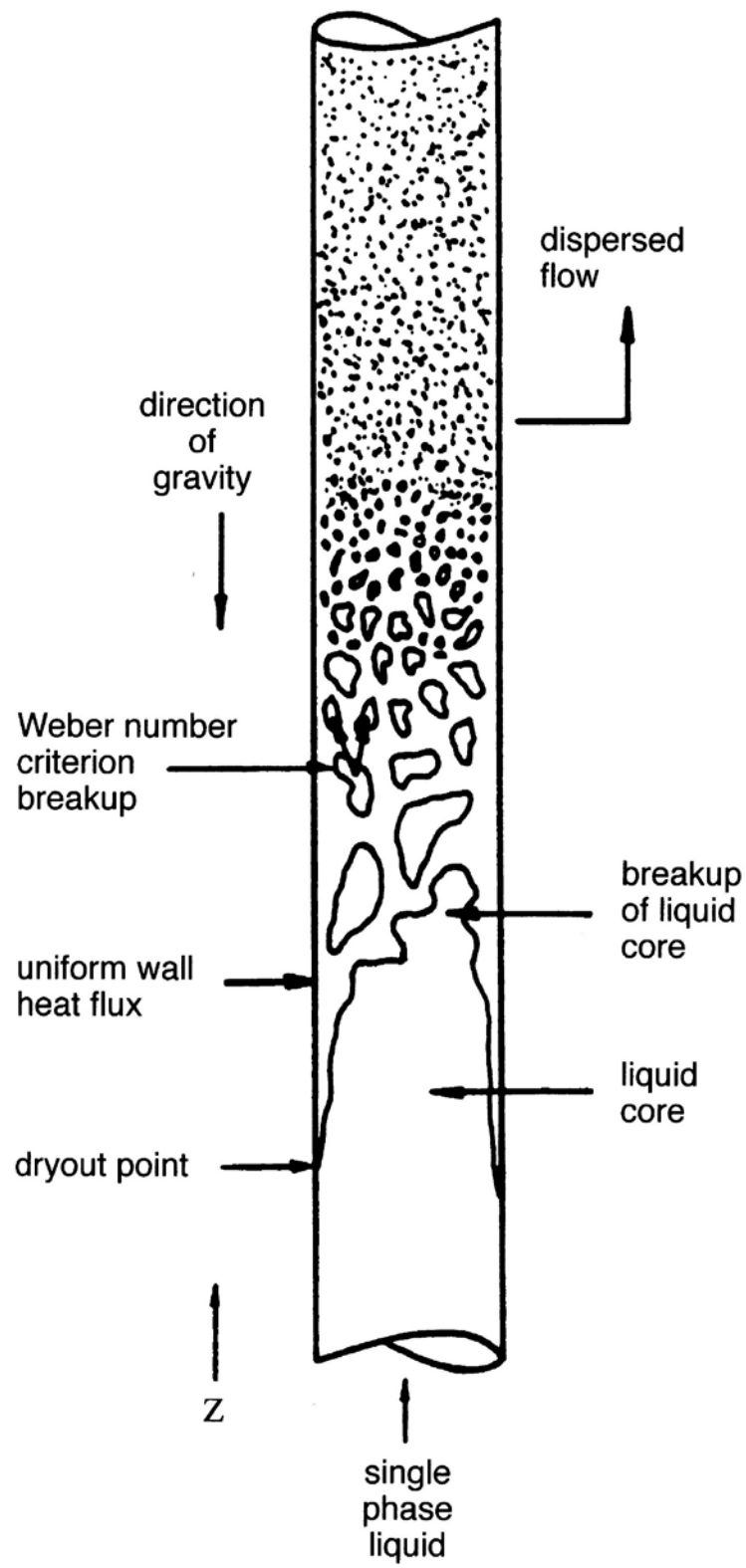


Figure 3. Flow regimes in a rising film evaporator tube.⁵

Vertical down flow inside tubes. Here, gravity acts in the same direction as the black liquor flow. When liquid and vapor both flow downward, flow is not chaotic. Down flow in black liquor evaporators is either falling film or annular, depending on the ratio of vapor to liquid volumetric flow rates, liquor velocity, and viscosity. Higher liquor velocity and vapor-to-liquid volumetric flow rate ratio move the flow pattern more toward annular flow.

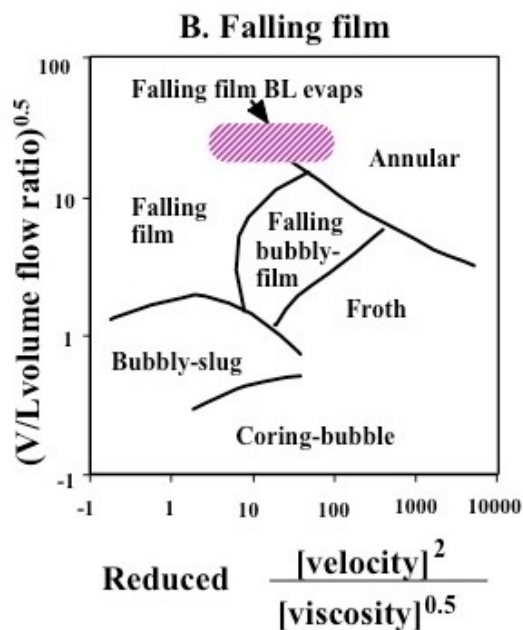


Figure 4. Flow map for black liquor inside a tube in a falling film evaporator.

Vertical down flow over tube heating elements.

The flow of black liquor downward on the outside of vertical tubes is complex. Figure 5 shows the appearance of a black liquor film in down flow near the top of a

tube (left), and near the bottom of the tube (right). These photos were taken for black liquor flowing on the outside of a tubular heat transfer surface in a pilot falling film black liquor evaporator. The range of Reynolds numbers ($Re_f = 4\Gamma_z/\mu_f$) in these measurements was from 2,000 to 56,000, where 5,000 represents the transition from wavy laminar to turbulent flow for viscous liquids in down flow on the outside of tubes.^{viii} This transitional Reynolds is three times higher than the accepted value of 1,600 based on measurements with low viscosity liquids.

The flow pattern of the black liquor falling film in the reported observations was smooth but wavy near the top of the tube. It became increasingly more chaotic along the tube length. One example is shown in [Figure 5](#). The chaotic flow characteristic became more pronounced at the highest viscosities. Bubble formation was not seen in the film, indicating an absence of nucleate boiling.

No dry out of the black liquor film was observed except in runs at the highest black liquor viscosity (570 Pa-s).

Even though the viscosity was very different in the different experiments, from 0.0015 Pa-s to 0.57 Pa-s, the hydrodynamics behavior was similar and did not deviate much from that in [Figure 5](#).

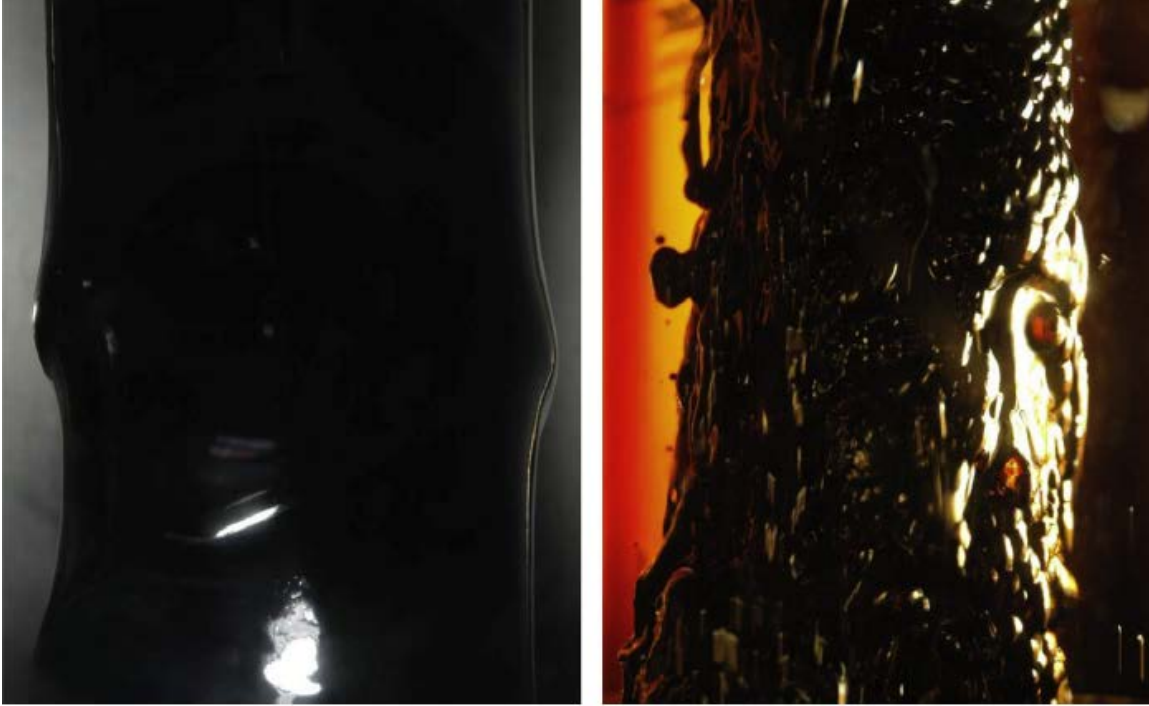


Figure 5. Hydrodynamic behavior of a black liquor falling film at $q = 7700$, $Re = 230$ and $Pr = 209$, (left) near the top of the evaporator tube, 0.3 m below the black liquor inlet, and (right) near the bottom of the tube, 4 m below the black liquor inlet.^{ix}
(From E. Karlsson et al., 2013.)

Vertical downflow over plate heating elements

Evaporation from liquid films destabilizes the films flowing downward over heated, vertical surfaces at all Reynolds numbers.^{x,xi} The instantaneous vaporization rate is greater at the troughs in the film rather than at the crests of surface waves since the film is thinner at the troughs. This creates thin spots in the film within the troughs. The vapor leaving perpendicular to the surface exerts a reactive pressure on the liquid. It is larger at the troughs than at the crests, and this tends to increase the wave amplitude.

The rupture into rivulets of a thin liquid film flowing down a vertical surface is

assumed to occur when both the continuous film and the rivulets carry the same mass flow and total (surface plus kinetic) energy and when, moreover, the latter exhibits in the rivulet configuration a local minimum.

On cooled surfaces, liquid films are not inherently unstable, and only becomes so at a critical Reynolds Number.

Design Procedure

Design basis and input variables

The design basis for an evaporator set is the amount of water to be evaporated per hour from the black liquor processed, i.e

$$\dot{m}_w = \dot{m}_{bls} (1/x_{out} - 1/x_{in}) \quad 0$$

Other values that need to be specified for designing an evaporator set are

Number of effects

Steam pressure to 1st effect (P_{stm})

Steam temperature to first effect, if superheated (T_{stm})

Mass flow rate of black liquor solids (\dot{m}_{bls})

Dry solids mass fraction of black liquor feed (x_{in})

Dry solids mass fraction of black liquor product (x_{out})

Temperature of cooling water to condenser for last effect vapor (T_{cw})

Calculation of the required heat transfer area

q_{tot} , the amount of heat transferred, is calculated from an energy balance around the evaporator unit. It depends upon the amount of water to be evaporated, the entering and exiting black liquor conditions, and exiting condensate conditions.

The overall thermal driving force, ΔT_o , is calculated from the saturation temperature of steam to the first effect ($T_{sat}(P_{stm})$) and the temperature of coolant (T_{cw}) in the vapor condenser following the last effect:

$$\Delta T_o = T_{sat,stm} - T_{cw} \quad 0$$

The overall available thermal driving force is ΔT_o minus the sum of the boiling point rise for each effect, or

$$\Delta T_{avail} = \Delta T_o - \Sigma(BPR_i) \quad 0$$

Heat transfer coefficients for each effect must be provided. Evaporator suppliers have proprietary data that they use in designing evaporator sets. For preliminary design estimates, values of heat transfer coefficients can be estimated using the correlations in Section 2.4 of this book.

ΔT_{avail} apportions itself among the evaporator effects to the evaporation requirements (heat demand) of each effect. The evaporator designer and operator are not able to specify the distribution of ΔT_i between effects.

The heat transfer areas for each effect is calculated as

$$A_i = \dot{q}_i / U_i \Delta T_i \quad 0$$

For initial calculations, $q_i = q_{tot}/n$ and $\Delta T_i = \Delta T_{avail}/n$. \dot{q}_i and ΔT_i will adjust themselves to correct values during iterative calculations.

Liquor recirculation rate

In forced circulation evaporators:

- The tube velocity is determined by a tradeoff between pumping cost and maintenance cost (tube erosion, fouling/scaling).
- The liquor recirculation to feed ratio is selected to maintain the desired temperature in the heat exchangers, control crystallization rate and location, and maintain the desired liquid velocity in the tubes.
- The operating pressure is high enough to prevent vaporization, which minimizes crystallization scaling of the heat exchangers.
- The pressure drop through the heat exchangers is depends upon the flow rate through the tubes and the viscosity of the black liquor. It may be set by optimization of capital vs operating cost for the evaporator set.

Flow Distribution

It is important that the flow of black liquor to each heat transfer tube or plate in a tube type falling film evaporator is nearly equal. This balances the rate of heat transfer for each tube and prevents dryout on the tube surface for those tubes receiving too little liquor. If the liquor loading is too low in some tubes, the flow of black liquor may be insufficient to initially wet completely the tube surface, or to maintain a fully wetted surface during evaporation. Under these conditions, the liquor film breaks down. Rivulets of liquor flow down the tube surface, with dry

regions interspersed between them. Heat transfer to the liquor is reduced, and dry regions are subject to scaling and fouling.

Liquid films flowing down a metal surface can wet the surface completely if certain criteria are met. For a liquid to initially wet the inside of a falling film evaporator tube, the liquor loading must exceed a minimum liquid loading given by

$$\Gamma_{\min} = C_{w,i} (\mu \text{ sg } \sigma^3)^{0.2}$$

where Γ_{\min} = minimum tube loading, kg/s-m, for complete surface wetting

μ = liquor viscosity at process conditions, cP

sg = liquor specific gravity

σ = liquor surface tension, dyne/cm

$C_{w,i}$ = initial wetting constant = 0.00806 kg/s-m (19.5 lb/hr-ft)

The same correlation applies the liquid loading to maintain a wetted surface during evaporator operation except the value of the wetting constant changes. The wetting constant becomes $C_{w,ss}$ = steady state wetting constant = 0.000992 kg/s-m (2.4 lb/hr-ft). The ratio of the initial to steady state wetting constants show that the minimum tube lading during startup needs to be at least 8 times the minimum loading during steady state operation.

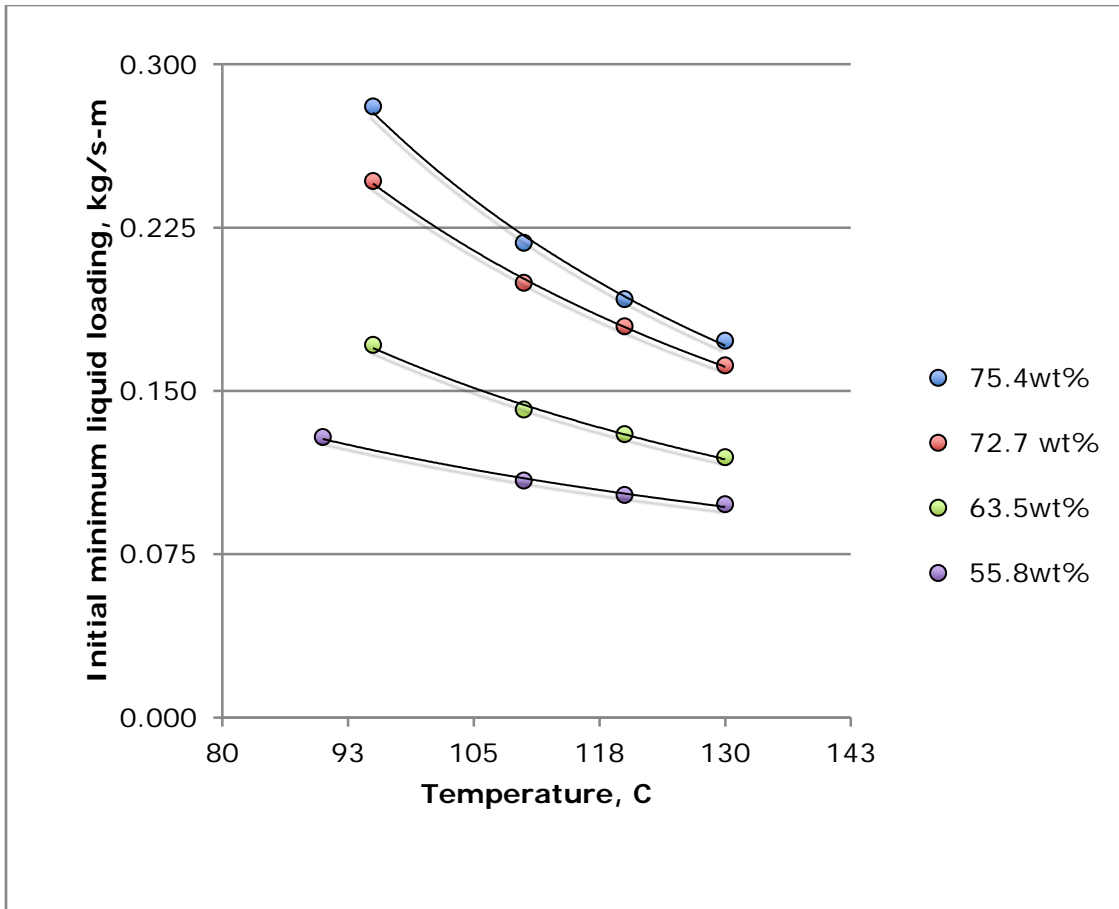


Figure 6. Minimum initial liquid loading (Γ_{min}) to establish complete wetting of tube surface versus dry black liquor solids content and liquor temperature. Data sets are for different black liquor dry solids contents.

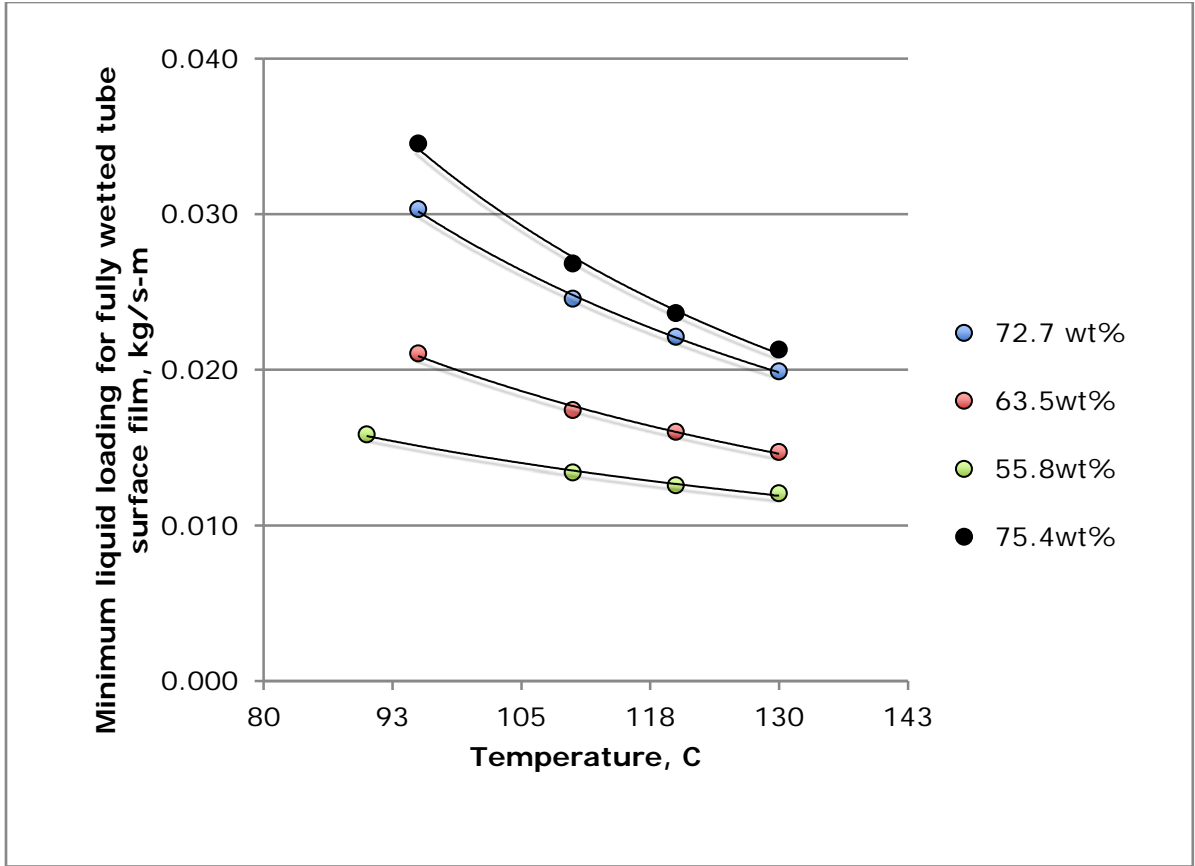


Figure 7. Minimum liquid loading (Γ_{\min}) to maintain complete wetting of tube surface versus dry black liquor solids content and liquor temperature. Data sets are for different black liquor dry solids contents.

Nucleate boiling occurring at the tube surface can cause liquor film rupture and film breakdown. This can be avoided by operating with low ΔT s. Recommended ΔT s for tube type falling film evaporators are $\Delta T < 6-7^{\circ}\text{C}$.

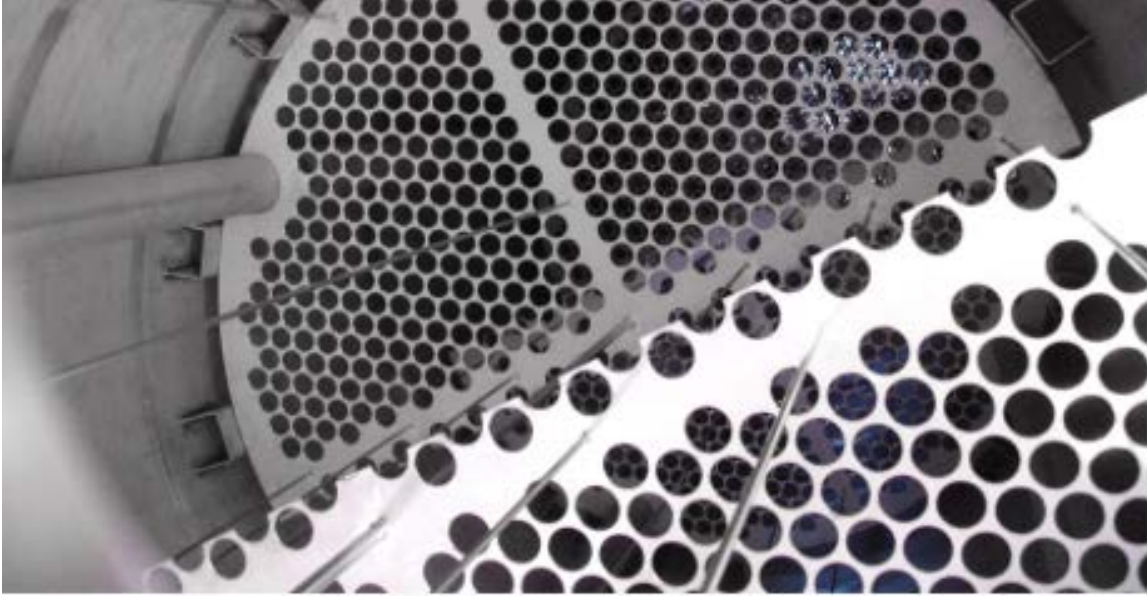


Figure 8. Distributor plate for tube ff evaporators for black liquor and other liquids.^{xii}

Flow distributors in Black Liquor falling film evaporators

The two most common types of flow distributors in black liquor falling film evaporators are drilled distribution plates and spray nozzles. Distribution plates can either be the width of the top liquor box (for tube units) or smaller with a weir wall (for tubes or plates). On a tube unit the liquor is directed through the drilled holes in the distribution plate and down onto the tubesheet between the tubes. It overflows into the tubes to form a black liquor film in each tube. The same concept is used with a plate unit, except the liquor is evenly distributed over the top line of the plate.

Spray nozzles are used in instances where the liquor properties cause issues with distribution plate plugging or preventing even black liquor distribution. The downside of spray nozzles is that some of the liquor is sprayed into the tube and does not adhere to the tube wall until it is part of the way down the tube. This is generally a minor issue when compared to the performance of that body with one with a drilled distribution plate.

Important design considerations for falling film evaporators include

- ☐ Fluid distribution in tubes
- ☐ Fluid distribution across the tubesheet
- ☐ Film breakdown wetting limits
- ☐ Lower ΔT limit for nucleate boiling
- ☐ Pressure drop along the liquor flow path

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- ^{xii} <http://www.gigkarasek.com/en/tube-falling-film-evaporator/> Feb 17, 2016.

2.4 Heat Transfer in Black Liquor Evaporators

The rate of heat transfer in black liquor evaporators determines the rate of evaporation and therefore the capacity to support a mill's pulp production rate. In this section, we consider factors that determine evaporation rate.

The overall rate of heat transfer in a steam-driven heat exchanger or evaporator body is set by $\dot{q} = U A \Delta T$ where U is the overall heat transfer coefficient, A is the heat transfer area, and $\Delta T = T_{\text{sat,stm}} - T_{\text{BL}}$ is the temperature driving force.

To estimate q , we need values for U , $T_{\text{sat,stm}}$, and T_{BL} . Evaluating ΔT for the individual evaporator effects in a multiple effect evaporator set is not straightforward because of the coupling of steam and vapor pressures and black liquor temperatures between effects. Evaluation of the distribution of ΔT s involves iterative calculations for the set. Since U_i can depend on ΔT_i , the calculation procedure is complicated when done manually. For an approximate and rapid manual calculation procedure, refer to Ulrichⁱ. Process simulation software greatly accelerates the calculations.

Evaluating ΔT for individual evaporator effects.

Evaluating ΔT for single evaporator body or a heat exchanger is complicated by the type of hardware, the flow configuration on both sides of the heat transfer device, and any change of phase that occurs on either side of it. Here are ΔT definitions for different cases.

- a.** For single phase flow on both sides of the exchanger, e.g. heat exchange between black liquor and hot water:

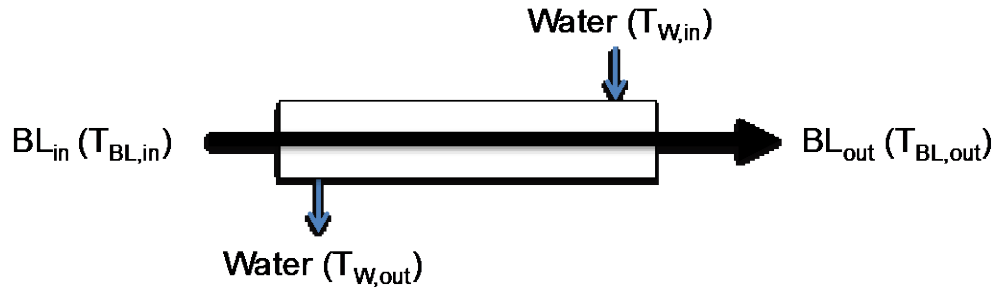


Figure 1. Simplified diagram of a single pass heat exchanger with single passes on both the tube and shell side.

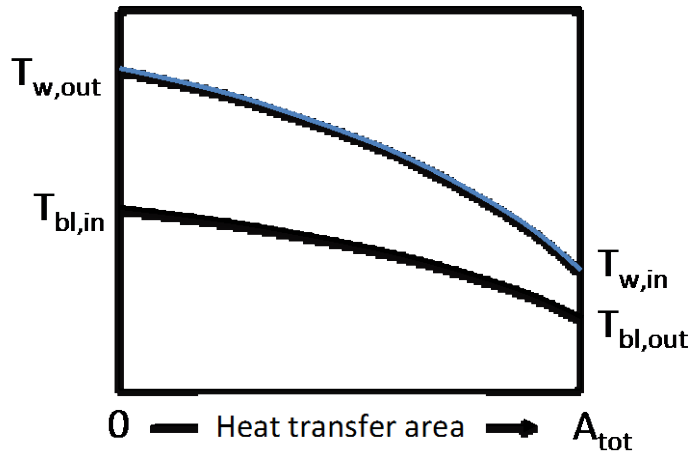


Figure 2. Temperature profiles for both the liquor and coolant (water) sides of the heat exchanger in Figure 1.

In this case, a log-mean temperature difference (LMTD or ΔT_{LM}) is used to calculate the overall rate of heat transfer in the heat exchanger:

$$\Delta T_{LM} = (\Delta T_{out} - \Delta T_{in}) / \ln (\Delta T_{out} / \Delta T_{in}) \quad (1)$$

$$\text{where } \Delta T_{out} = T_{BL,out} - T_{w,in} \text{ and } \Delta T_{in} = T_{BL,in} - T_{w,out} \quad (2)$$

Multi-pass heat exchangers for similar applications require a correction factor (F) be applied to ΔT_{LM} . F is evaluated for configuration-specific equations or published charts.^{ii,iii} Figure 3 contains an F -factor chart for a single shell, double tube pass heat exchanger. Similar charts are available for many shell and tube configurations.

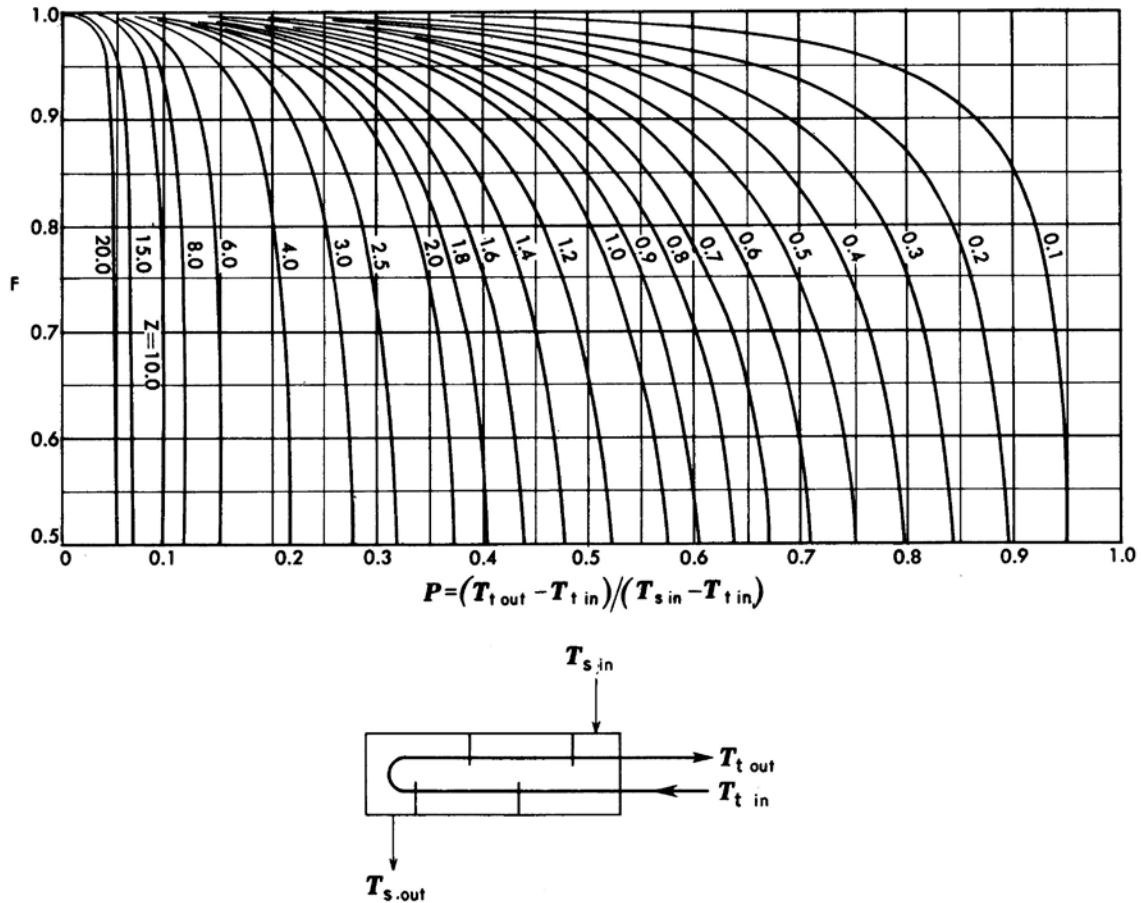


Figure 3. Correction factor (F) to counter-flow ΔT_{LM} for heat exchangers with one shell pass and two or multiple of two, tube passes. From Kreith^{iv}.

- b.** For LTV evaporators with vertical, 2-phase flow, the vertical black liquor temperature is controlled by the pressure in the evaporator body and the changing boiling point rise of the liquor as it moves up the tube. In the subcooled region, the liquor temperature profile is determined by the entering liquor temperature and the ratio of heat transfer rate (\dot{q}) to liquor

thermal mass flow rate ($\dot{m} c_p$). Above the onset of 2-phase flow, the local temperature depends mainly upon cooling by evaporation as well as the local pressure and boiling point elevation of the black liquor. Figure 4 shows two typical black liquor temperature profiles in rising film LTV evaporators. ΔT_{LM} is not applicable to these types of temperature profiles. By convention, the ΔT used is the difference between the saturated steam temperature ($T_{s,stm}$) and the temperature of the black liquor exiting the heat transfer region ($T_{BL,exit}$), so that

$$\Delta T_{LTV} = T_{s,stm} - T_{BL,exit} \quad (3)$$

For falling film and forced circulation evaporators, the ratio of recirculated to feed black liquor is high, typically about 20:1, the boiling point rise change is small through the unit, and the black liquor temperature is controlled mainly by the pressure for the unit. While a log-mean ΔT is applicable for these types of evaporators, a simple temperature difference between the saturated steam and black liquor exiting the unit gives approximately the same result, so

$$\Delta T_{FF} = T_{s,stm} - T_{BL,exit} \quad (4)$$

$$\Delta T_{FC} = T_{s,stm} - T_{BL,exit} \quad (5)$$

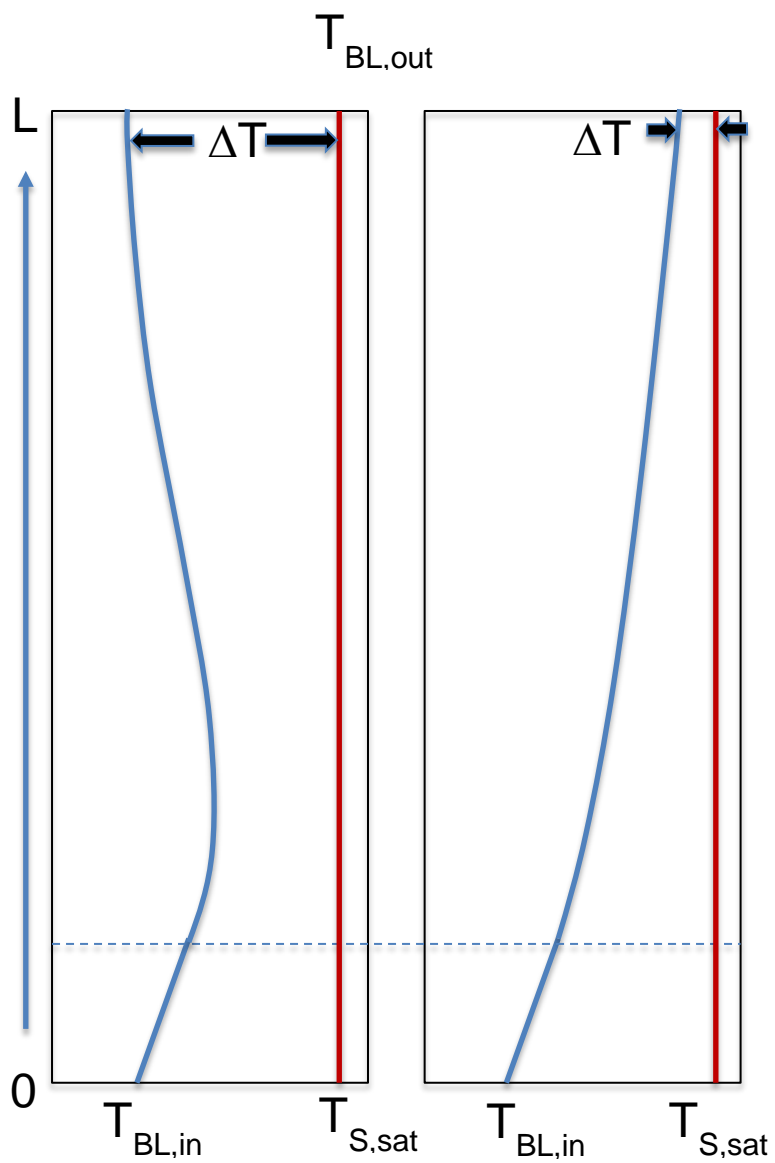


Figure 4. Vertical temperature profiles for black liquor in LTV evaporators.

Different temperature profiles are also possible:

- At a given location, liquor temperature increases with the rate of heat transfer, but decreases with evaporation and as pressure decreases as the liquor moves up in the tube.
- The left figure is for black liquor at low dry solids content, where the BPR does not increase rapidly as water is evaporated.
- The right figure is for high dry solids black liquor, where the BPR increases rapidly as water is removed.

In summary, the definition of ΔT is different between LTV evaporators and other types. The values of heat transfer coefficients (U) calculated for each different type of evaporator will differ from each other because the differences in how ΔT s are defined. U values obtained from one type of evaporator should not be used with other types of evaporators even when they are processing the same black liquor.

Estimating Heat Transfer Coefficients for Black Liquor Evaporators

Methods for estimating heat transfer coefficients in two phase flow have been developed over the past six decades. However, most were based on data for low viscosity liquids such as water, light hydrocarbons, and alcohols. They do not predict accurately heat transfer coefficients with more viscous fluids such as black liquor.^v Only a few correlations have been developed specifically for black liquor or for other viscous liquids. Here we summarize the estimation methods for heat transfer coefficients with black liquor in the heating elements of different types of black liquor evaporators.

Terminology for dimensionless heat transfer correlations

Heat transfer coefficient are normally correlated using dimensional variables for heat transfer coefficients, flow behavior, fluid properties, and equipment

dimensions. These include

1. the Nusselt number for the heat transfer coefficient.
 - a. for single phase flow in a tube: $Nu = hD/k_f$
 - b. for falling liquid film heat transfer on a surface: $Nu_f = h_{TP} \delta / k_f$
2. the Reynolds number for flow behavior
 - a. for single phase flow in a tube: $Re = DV\rho/\mu$
 - b. for falling liquid film on a surface: $Re_f = 4\Gamma/\mu$
3. the Prandtl number for fluid properties: $Pr = c_p \mu/k$

Heat transfer coefficient correlations for convective flow heat transfer normally take the form of

$$Nu = K Re^a Pr^b \quad (6)$$

The values of K, a, and b are obtained by fitting equation 1 to experimental data. These parameter values are not universally constant but change with e.g. flow regime change from laminar to turbulent, and with system geometry.

Heat transfer coefficients depend strongly on fluid viscosity to approximately the power 0.4 ($h \propto 1/\mu^{0.4}$ in equation 1). The viscosity of black liquor increases from 1 cP to 10^4 cP when black liquor is concentrated from a dry solids content of 15wt% to 80wt% dry solids content. With this great a change in viscosity, h decreases by a factor of 40. Because of the strong viscosity dependence, heat transfer correlations developed from low viscosity fluid data do not give accurate estimates of heat transfer coefficients for black liquor.ⁱ

The standard definition for Nu in falling film heat transfer is

$$Nu = (h/k)(v^2/g)^{1/3} \quad (7)$$

Correlations are valid in certain flow regime (laminar, wavy-laminar, or turbulent), as characterized by the Reynolds (Re) number. The Reynolds number for falling film flow is

$$Re = 4G/\mu \quad (8)$$

Nusselt number correlations usually take on a different form in each different flow regime. Table 1 contains examples of some different forms. Various reported values for K, P1, and P2 are included. The parameter values, along with applicable ranges are given in Table 2. In most of the experimentally based correlations were developed from experiments where the evaporated fluid flowed on the outside surface of a vertical tube, while the equation proposed by Nusselt (1916) was derived analytically.

The Re number at the transition from laminar flow with capillary or roll waves to turbulent flow^{vi} is

$$Re_{trans} = 5800Pr^{-1.06} \quad (9)$$

The Kapitza number ($Ka = \sigma/\rho(g \sin\beta)^{1/3}v^{4/3}$), an indicator of the hydrodynamic wave regime in falling liquid films, is included some correlations.

Table 1. Nusselt number expressions for falling film flow on surfaces (from Johansson).

Flow regimes	Nusselt number relationship
Laminar flow with no ripples	$Nu = K Re^{P1}$
Laminar flow with capillary or roll waves	$Nu = K Re^{P1}$
Transition ^a	$Nu = (Nu_{ripples}^n + Nu_{turbulent}^n)^{1/n}$
Turbulent	$Nu = K Re^{P1} Pr^{P2}$

^a Values for n of 2, 5, and 6 have been used in the transition flow correlation.

Three Nu versus Re correlations for the laminar with ripples vertical flow regime from other researchers all give very similar results. Combining three of those correlations into one equation, gives:

$$Nu_{l-r} = 1.20 Re^{-0.298}$$

This correlation is valid for the Prandtl number range $2 < Pr < 8$.

A correlation for turbulent vertical flow, developed using a similar approach, is

$$Nu_t = 0.0123 Re^{0.321} Pr^{0.559}$$

This correlation is valid approximately for the range $10^4 < Re < 10^5$ and $1 < Pr < 50$.

Convective Heat Transfer Correlations for Black liquor

The next three sections present heat transfer correlations for black liquor in both single- and two-phase flow in climbing film evaporators, falling film evaporators, and forced circulation concentrators.^{vii}

For single-phase laminar flow in a circular cross section tube:

$$Nu_{lam} = (6.488^3 + 0.517^3(f Re)(Re Pr D/x)) \quad (2)$$

$$\text{where } f = 1.82 \log_{10} (Re) - 1.64)^{-2}$$

For single-phase turbulent flow in a tube:

$$Nu_{turb} = \{[(f/8(Re-1000)/Pr)/[1 + 12.7 (f/8)^{1/2} Pr^{2/3}-1]]\}^{1/4} \{1+1/3(D/x)^{2/3}\} \quad (3)$$

Replace Re with Re* in equations 2 through 4, where

$$Re^* = [Re^2 + Gr/2.5]^{0.5} \quad (4)$$

and

$$Gr = g \beta (T_s - T_\infty) D^3/\nu^2 \quad (5)$$

where

g = acceleration due to Earth's gravity

β = volumetric thermal expansion coefficient (equal to approximately $1/T$, for ideal gases, where T is absolute temperature)

T_s = surface temperature

T_∞ = bulk liquor temperature

L = characteristic length

D = diameter

ν = kinematic viscosity

Subcooled flow boiling heat transfer

Subcooled flow boiling occurs when the wall temperature is high enough to support bubble growth at the tube wall, but the bulk temperature remains below saturation. For subcooled flow, the two phase heat transfer coefficient (h_{tp}) is the sum of contributions from nucleate boiling (h_{nb}) and forced convection (h_{fc}):

$$h_{tp} = h_{fc} F + h_{nb} S \quad (6)$$

The parameter F accounts for the increase in liquid velocity due to the expansion of liquid to vapor. h_{fc} in Equation 6 is obtained from either equation 2 or 3. The two phase Reynolds number for the liquid phase is

$$Re_{tp} = Re (1-\dot{x}) F^{1.25} \quad (7)$$

Where \dot{x} is the vapor mass quality.

$$\dot{x} = \dot{m}_{vap} / (\dot{m}_{vap} + \dot{m}_{liq}) \quad (8)$$

S accounts for the lower effective superheat in the region of bubble formation, which suppresses nucleate boiling.

F and S are given by

$$F = 1 \text{ for } 1/X_{tt} \leq 0.1 \quad (9)$$

$$F = 2.35 (1/X_{tt} + 0.213)^{0.736} \text{ for } 1/X_{tt} > 0.1 \quad (10)$$

$$S = 1/(1 + 2.53 \times 10^{-6} \text{Re}_{tp}^{1.17}) \quad (11)$$

X_{tt} is the Martinelli parameter, a dimensionless number used in internal two-phase flow calculations. It expresses the liquid fraction of a flowing fluid as $X_{tt} = (m_l/m_g)(r_g/r_l)^{0.5}$. Its main application is in two-phase pressure drop and boiling/condensing heat transfer calculations. It is calculated as

$$X_{tt} = ((1-\dot{X})/\dot{X})^{0.9} (\rho_v/\rho_l)^{0.5} (\mu_l/\mu_v)^{0.1} \quad (12)$$

Determination of flow quality

Flow quality (\dot{X}) is the ratio of vapor mass flow rate to the total mass flow rate. For the subcooled fluid,

$$\dot{X} = Ph - Ph_n \exp\{(Ph/Ph_n)-1\} \quad (13)$$

where Ph , the phase change number at the transition from region B to C in Figure 5.

$$Ph = (h_{tp} - h_1^*)/\Delta h_v \quad (14)$$

$$Ph_n = -Bo/\{(4.55/Pe)^2 + 0.00652\}^{0.5} \quad (15)$$

where Pe = Péclet ($Re Sc$) and Sc is the Schmidt number ($\mu/\rho D$).

$$Bo = \dot{q}/\dot{m} \Delta h_v \quad (16)$$

The phase number and Boiling number (Bo) are related by

Nucleate boiling heat transfer

For saturated nucleate boiling

$$h_{nb}/h_o = (\dot{q}_{nb}/\dot{q}_o)^{N_p} F_p \quad (18)$$

$$N_p = 0.9 - 0.3(P/P_c)^{0.15} \quad (19)$$

$$F_p = 2.55(P/P_c)^{0.27} + [9.0 + \{1/(1-(P/P_c)^2)\}][P/P_c]^2 \quad (20)$$

The reference heat transfer coefficient (h_o) is calculated from Eq. 21:

$$Nu = 0.0871(\dot{q}_o d_B/\lambda_l T_{sat})^{0.674} (\rho_v/\rho_l)^{0.156} (\Delta h_v d_B^2/a_e^2)^{0.371} (a_l^2 \rho_l/\sigma d_B)^{0.350} Pr_l^{-0.162} \quad (21)$$

$$a_l = \lambda_l/\rho_l C_{p,l} \quad (22)$$

The range of the reference heat transfer coefficient is from 3400 to 4400 W/m²K.^{vii}

The Nusselt number in equation 21 is

$$Nu = h_o d_B/\lambda_l \quad (23)$$

The bubble diameter (d_B) is estimated from

$$d_B = 0.0146 \beta [2\sigma/g(\rho_l - \rho_v)]^{0.5} \quad (24)$$

Equation 6 suggests that

$$\dot{q} = \dot{q}_{fc} + \dot{q}_b \quad (25)$$

For equation 25 to be valid, we must redefine the nucleate boiling heat transfer coefficient (h_{nb}) as

$$h_{nb} = \dot{q}_b / (T_s - T_{sat}) \quad (26)$$

Equation 25 must hold for subcooled boiling, so

$$h_{sb} = h_{nb} \{(T_s - T_{sat}) / (T_s - T_b)\} \quad (27)$$

Falling film heat transfer, black liquor on outside of tubes

Many researchers have proposed correlations for heat transfer coefficients in falling film evaporators with the solution being evaporated on the outside of the tubes. Many were developed for turbulent flow, but most were developed for Prandtl numbers less than 20. The Prandtl number for black liquor, which has a higher viscosity than simpler liquids, exceeds 50 as it is concentrated, and can be as high as 500.

The conventional form for correlations for falling film heat transfer correlations is

$$Nu = K Re^a Pr^b \quad (28)$$

where

- Nu is defined as $(h/k) (v^2/g)^{1/3}$
- Re is defined as $4\Gamma/\mu$
- Pr is defined as $c_p \mu/k$

Johansson presented a straightforward correlation for heat transfer coefficients in black liquor falling film evaporators with the liquor on the outside of heat transfer tubes. The correlation is in the form of equation 28. The constants K, a, and b for black liquor were determined from extensive experimental data obtained using a research evaporator described by Johansson and others^{viii}. Their values based on the black liquor used and other operating parameters (q range, ΔT range, etc.) are $K = 201$, $a = 0.26 \pm 0.12$, and $b = -0.41 \pm 0.05$. With these constants, equation 28 becomes

$$Nu = 201 Re^{0.26} Pr^{-0.41} \quad (29)$$

The correlations presented here are for turbulent flow of the black liquor film down the outside surface of the tubes. The Reynolds number at the transition from wavy-laminar flow to turbulent flow is

$$Re_{trans} = 5800 Pr^{-1.06} \quad (30)$$

The range of experimental conditions for data on which the correlation is based are

- ☐ Black liquor temperature in the pilot evaporator: 50-60°C and 110-120°C
- ☐ Black liquor dry solids content: 35 to 51 wt%
- ☐ Black liquor rheology: all of the pilot evaporator data was taken at conditions where black liquor behaves as a Newtonian fluid.

Wetting of heat transfer surfaces by falling black liquor films^{ix}

Falling film evaporators are used extensively in the food industry for their ability to process heat sensitive liquids. A coherent liquid film is required to maintain heat transfer efficiency and minimize fouling. It is likely that most evaporator fouling occurs after film breakdown as the substance within the evaporator dries out. The

minimum flow rate required to maintain a film is known as the minimum wetting rate which is defined as the minimum mass flow rate per unit circumference. In this work, minimum wetting rates were determined in a 1 m long, 48 mm internal diameter, vertical, stainless steel tube. Water and aqueous solutions of glycerol, alcohol and calcium chloride were used. These substances were chosen so as to give a wide range of properties such as viscosity (0.5-39 mPa s), density (950-1410 kg/m³), surface tension (35-90 mN/m) and contact angle (64-98°). In a separate set of experiments, the minimum flow rate required to distribute liquid and completely wet the top of industrial evaporator tubes was measured using a range of sucrose solutions.

The tube wetting results obtained fitted a dimensionless power law relationship well. Surface tension and contact angle had a strong influence on the wetting rate but viscosity and density were found to have very little effect. The minimum flow rates for distribution were found to nearly always exceed the minimum wetting flow rate to maintain rates showing that more attention needs to be given to distributor design.

The flow rate to initiate or maintain a continuous film wetting the heat transfer surface is^{x,xi}

$$\Gamma_{\min} = C_w (\mu_l s_l \sigma^3)^{0.2}$$

For initial surface wetting, $C_w = 19.5$ while to maintain a wet surface, $C_w = 2.4$.

In equation 1, $\Gamma_{\min} = [\text{lb/hr-ft}]$, $\mu_l = [\text{cP}]$, $\sigma = [\text{dyne/cm}]$, and $s = \text{specific gravity}$.

Nucleation Film Destruction

Nucleate boiling that generates vapor bubbles can occur at the surface if the wall ΔT is too high. These bubbles disrupt falling liquid films on heat transfer surfaces. As a rule, the wall ΔT should not exceed 25°C to avoid film disruption. Falling film evaporators for black liquor are designed with much lower wall ΔT s, about 5°C, so that nucleation does not break down black liquor films in falling film evaporators.^{ii,iii}

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2.5 Evaporator and Concentrator Design Considerations for Troubleshooting

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INTRODUCTION

Kraft pulping requires water removal from spent liquor to enable its burning in a chemical recovery boiler. Multiple-effect evaporators and concentrators have been developed for this service. Evaporators and concentrators are typically integrated to maximize steam economy and overall performance. The nominal dry solids content increase in evaporators is from a feed of 15 wt-% to a product of 50 wt-%. The nominal dry solids content of black liquor fed to concentrators is 50 wt-%. The dry solids content of product liquor from older concentrator systems is from 65 wt-% to 70 wt-% while dry product solids content for newer concentrator systems are typically 75 wt-% to 80 wt-% [1].

Higher product solids content increases the recovery boiler steam generation per mass of black liquor solids fired. Dry solids content is therefore a key operating parameter. Temperatures of warm water from surface condensers and clean process condensates from evaporator and concentrator sets are two other key operating parameters. Warm water temperature targets that are sustained year-round enable minimum low pressure steam use for water heating, e.g., in the pulp mill, bleach plant, recausticizing area, and paper machines.

Steam economy is the measure of evaporator and concentrator efficiency of energy use. It is calculated as the mass of water evaporated per mass of steam used.

Evaporator and concentrator designs are based principally on a specific water

removal rate. The dry product solids content and the water temperature from the surface condenser are also specified for the design for black liquor evaporators and concentrators. The number of effects, number of bodies, and their configuration are determined by economic optimization of capital and operating costs. Maximum use is made of clean process condensate, either directly as warm/hot water or indirectly by recovering its useable energy. Best practice designs enable the owner's staff to sustain these design configurations and operating targets over the life of the set. The intent of this section is to present a design approach that enables rapid identification of conditions that cause the above three key energy parameters of the evaporator / concentrator set and its steam use to deviate from their targets.

DESIGN OF EVAPORATION SYSTEMS

Evaporator and concentrator suppliers design and build sets based on specifications developed by owners or their engineering representative. The specifications define expected conditions which the set must meet. Traditionally, once the set has met these specifications as confirmed by performance tests, the supplier's obligations are met. The owner on the other hand is now charged with sustaining the specified performance. It is therefore in the owner's long term interest to ensure that sufficient control and monitoring tools are included in the original design to sustain performance. A recommended evaporator test procedure including process parameters to measure is available for use by owners and suppliers [3].

Control tools are included in evaporator design to sustain short-term performance. Product liquor dry solids content can be controlled by varying the steam supply to the set. There are several different control strategies for dry product solids content that are successful. However, many mills continue to use steam pressure or steam flow as the control parameter and manually adjust the set point to meet the product solids content target. Warm water temperature from the surface condenser can be controlled by varying the cooling water flow rate. This in turn gives stable vacuum control, which is essential for good evaporation system operation. Both percent product solids and warm water temperature can be measured directly , controlled,

and evaluated against their respective targets.

Table I. Basic Operation Instrumentation

	Control &	Monitor &
	Monitor	Alarm
Weak black liquor feed flow rate	Yes	
Percent WBL feed solids content		Yes
Product solids flow rate		Yes
Percent product solids content via BPR	Yes	
Final concentrator effect vapor pressure	Yes	
Product liquor final flash tank pressure	Yes	
Steam pressure and flow rate capability	Yes	
Steam temperature after desuperheater	Yes	
Vacuum in surface condenser		Yes
Surface condenser water temperature in & out	Yes	
Clean steam condensate conductivity		Yes
Combined process condensate(s) flow rate		Yes
Combined process condensate(s) conductivity		Yes
Foul condensate flow rate		Yes
Foul condensate conductivity		Yes
Levels in some vessels, tanks, or stand-pipes	Yes	Yes
Amps used by some pump motors		Yes

Clean process condensate quality and temperature can be measured directly. Target values are included in the set design. They cannot be controlled directly because they depend upon numerous other upstream operating conditions. The overall steam economy of an evaporation system is also not controlled directly. These operating conditions include both parameter changes, e.g., feed liquor flow rate and temperature, but in addition include operating practices, e.g., selection of the flash tank vapor sink and bypassing of liquor heaters.

Significant differences related to instrumentation can be included in original evaporation system designs. Table I shows current minimum instrumentation for evaporator and concentrator systems operation recommended by the author.

These instrumentation and control functions enable the evaporation system to meet typical owner specifications related to black liquor processing. However, these are not sufficient to determine the cause of problems if one of the specifications is not met. Practices and instrumentation recommended for troubleshooting to identify the cause of problems are discussed in the next sections.

Upon commissioning of a new or upgraded evaporator and concentrator system, the mass and energy balances for the design case(s) and the performance test(s) are delivered to the owner. The latter become the commercially-clean desired condition(s) that defines best performance. It has been possible on some projects for owners to obtain both paper and electronic copies of their specific balances. Electronic copies are typically protected with only key input cells accessible.

Evaporator and concentrator designs are based on either measured or assumed physical and chemical characteristics of the black liquor to be processed [2]. The specific properties measured depend on the supplier and the owner's specifications. In either case the physical and chemical characteristics that are included in the design basis are to be understood by both the supplier and the owner.

TROUBLESHOOTING EVAPORATION SYSTEMS

Evaporator / concentrator systems consist of a series of bodies interconnected through numerous vapor ducts and pipes. Overall performance of the set can be assessed by measuring the product dry solids content, warm water outlet temperature, and condensate conductivity and temperature. However, determining why one or more of these parameters is not meeting target requires that performance of individual bodies be determined. Since there can be from five to eleven bodies configured as five to eight effects in a set, the first step is to identify where the problem is located. Design of the set needs to include tools that will rapidly identify which body(s) is not performing as designed.

The fundamental starting point for troubleshooting is to define both the current set of operating conditions and a desired set of operating conditions. The latter can either be an acceptable set of operating conditions, or conditions during the system acceptance tests, or the design specifications. The first troubleshooting objective is to identify what has changed between these two sets of conditions. Definition of the current and prior conditions ideally would include not only available operating data, but also mass and energy balances. In cases where the design and/or desirable condition balances are not available, an evaporator supplier or another capable firm can be used to estimate the original design balances.

Since black liquor is a complex solution of water, inorganic salts, organic salts, and free organics, along with suspended solids, other pulp mill processes will influence its chemical composition. This means that changes in other areas of the mill may contribute to an evaporator and concentrator problem. A comparison of physical and chemical characteristics of the liquor at current and prior conditions may be needed, especially if heat transfer surface fouling is a potential cause of the problem. One example is a significant decrease in causticizing conversion which can lead to increased evaporator and concentrator fouling.

Potential changes in operating procedures or personnel need also to be considered. Potential issues in these areas need to be recognized and the design executed to minimize their likelihood. Differences in boil out procedures and nonuniformity in

operation between shifts are examples where personnel and procedures can contribute to operational problems.

DESIGN PRINCIPLES FOR TROUBLESHOOTING EVAPORATION SYSTEMS

When evaporator and concentrator issues occur there are typically limited knowledgeable resources at the mill, especially at mills that traditionally have not had problems in these areas. Mill operations and technical staffs have been reduced significantly over the last twenty years. Best practice evaporator / concentrator design will include tools for monitoring and troubleshooting that are useful to both operations and technical personnel. Instrumentation in addition to those needed for operation is required. Table II below lists these instruments and the subsequent discussion will highlight their use in troubleshooting operating problems.

Table II. Troubleshooting Instrumentation: Additions to basic operation instrumentation

	Applied in Principle No.
Temperature Monitoring:	
Weak black liquor feed	3&4
Liquor from each bodies	1&3
Liquor to & from internal & external heaters	2&3
Liquor to / from soap skimmer	3&4
Liquor from intermediate flash tanks	3&4
Product liquor from final flash tank	3&4
Combined condensate(s)	4
Foul condensate	4

Steam temperature at flowmeter	3
Steam condensate from final flash tank	3&4
NCG from surface condenser & pre-cooler	4
Pressure Monitoring:	
Over liquor in each body or exiting vapor duct	1 &3
Over product liquor in final flash tank	3&4
Vapor pressure in final steam flash tank	3&4
Vacuum in surface condenser(s)	2&3
Pressure / vacuum in hotwell or outlet of after condenser	3&4
Percent Solids Monitoring with Meter:	
Product liquor from final flash	3&4
Feed weak black liquor	3&4
Flow Monitoring	
Liquor flow from soap skimmer	4
Water flow(s) to surface condenser	4

Data from the recommended additional instruments for troubleshooting are applied in one or more design principles that are noted in Table II.

Principle 1

Evaporator and concentrator systems designed for troubleshooting incorporate measurements and calculations that display, trend, and archive ΔT and BPR values for each body.

The classic individual body performance parameter is the temperature difference (ΔT) between the saturated vapor temperature on one side of the heating surface and the outlet liquor temperature on the other side. The saturated vapor temperature is calculated from the corresponding measured vapor pressure.

$$\Delta T_n = T_{n-1, \text{vapor, sat}} - T_{n, \text{bl, out}}$$

where: $\Delta T_n = \Delta T$ in body n.

n is a body and n-1 is the higher temperature body supplying vapor to body n.

note: If the body contains a forced circulation heater, see Principle 2.

A high ΔT compared to design or a desired condition implies poor heat transfer in a body. This may be the result of heat transfer fouling or several other conditions. High ΔT screening of individual bodies is the first- level approach to identify the body or bodies that have the problem.

The boiling point rise (BPR) of liquor within a body can also be calculated from the difference between the liquor temperature in the body and the saturated vapor temperature over the liquor in the same body.

$$BPR_n = T_{n,bl} - T_{n,vapor, sat}$$

where: BPR_n = boiling point rise of black liquor in body n.

Boiling point rise is mainly a function of dry solids content. It can be used to estimate dry solids content deviations for a specific body. A lower BPR compared to a desired condition, when adjacent bodies are not lower, indicates a possible condensate leak into the black liquor. Since the BPR is relatively low at low dry solids content, it is difficult to use this as a definitive indication of condensate leaks in bodies where the dry solids contents are below 30%.

Both the ΔT and the BPR calculations require saturated vapor temperatures. Accurate pressure measurements in either the vapor space over the liquor or in the exiting vapor duct, coupled with calculations to determine the saturated vapor temperature, are required. Steam tables are sufficiently accurate for black liquor applications. Direct measurement of vapor temperature over the liquor has potential for high temperature errors of superheat and contact with carryover liquor. These errors can be reduced by locating the temperature sensor at the

condensation destination for the vapor, e.g., heat-supply side of the subsequent effect. Any ΔT due to pressure drop through the vapor ducts needs to be accounted for to determine the actual vapor temperature in the upstream effect.

Principle 2

Evaporator and concentrator systems designed for troubleshooting incorporate measurements and calculations that display current values and trends, and they archive the ΔT_{LM} and approach temperatures for each internal heater, external heater, and surface condenser.

The applicable DT in these heat exchangers is the log-mean ΔT , or ΔT_{LM} . These heat exchangers involve a condensing vapor heating a liquid stream, either liquor or water on the other side of the heat transfer surface. It is calculated by:

$$\Delta T_{LM} = (A - B) / \ln(A/B)$$

where: $A = T_{\text{vapor, sat}} - T_{\text{liquid, in}}$

$B = T_{\text{vapor, sat}} - T_{\text{liquid, out}}$

T_v = saturated temperature of condensing inlet vapor

T_l = temperature of liquid

\ln = natural logarithm

This is also the appropriate equation to use for forced circulation concentrator bodies that have fully flooded heat exchangers with recirculating black liquor. Use of ΔT_{LM} corresponds to the use of ΔT in Principle 1. The inlet approach temperature for these heat exchangers is important since internal vapors and liquids are the heating sources and sinks, respectively. Their temperatures vary with process and heat transfer surface conditions. The inlet approach ΔT is calculated by:

$$\text{inlet approach } \Delta T = T_v - T_{l, \text{in}}$$

Inlet approach ΔT is a useful tool to identify if a low outlet liquid temperature is due to a low inlet vapor temperature and not the result of heat transfer surface fouling.

Tracking the inlet approach ΔT for a surface condenser will also give an early alert to fouling, if the value increases continuously.

Principle 3

Evaporator and concentrator systems designed for troubleshooting incorporate measurements of the key energy-related parameters necessary for completing performance mass and energy balances.

Mass and energy balances provide predictions of liquor and vapor flow rates, energy flow rates, and dry black liquor solids content to and from bodies. Heat transfer coefficients, which are measures of heat transfer performance, are also calculated for each body. These reflect more accurately the heat transfer performance condition of a body, since they are less dependent on the evaporation rate, steam flow rate, and liquor flow rate than on the ΔT . The overall heat transfer coefficient (U_i) for body i is calculated by:

$$U_i = \dot{q}_i / (A_i \Delta T_i)$$

where: \dot{q}_i = rate of heat transferred in body i

A_i = heat transfer surface area for body i

ΔT_i = either ΔT or ΔT_{LM} for body i as noted in Principles 1 & 2

The energy transferred between bodies is a result of the mass and energy balance, which enables the heat transfer coefficients (U_i 's) to be determined for each body. Balances are not necessary to calculate an estimate of the heat transfer coefficients (U_i 's) for bodies or heaters supplied with steam for which flow and pressure are measured. These calculations are readily done in the owner's DCS systems.

$$\dot{q}_{s,i} = \dot{m}_{s,i} h_{v,s,i}$$

where: $\dot{q}_{s,i}$ = rate of heat transferred from steam or vapor in body i

$\dot{m}_{s,i}$ = mass flow rate of steam or vapor into body i

$h_{v,i}$ = heat of vaporization of steam in body i

The overall steam economy of the set is calculated from the evaporator set mass and energy balances. It can also be determined by calculations readily done in the owner's DCS systems. Use the steam flow rate and calculate the evaporation rate using the weak liquor inlet flow rate and dry solids content in the feed and product liquor streams.

$$SE = \dot{V}_{wbl} \rho_{wbl} x_{wbl} (1/x_{wbl} - 1/x_{hbl}) / \dot{m}_s$$

where: SE = steam economy

\dot{V}_{wbl} = volumetric flow rate of weak black liquor

ρ_{wbl} = density of weak black liquor

x_{wbl} = fraction of solids in weak black liquor feed

x_{hbl} = fraction of solids in heavy black liquor product

Cross-checks can also be done if the product flow rate is measured and/or if all condensate flow rates are measured.

A valuable function of a mass and energy balance is that the consistency of data for a given test can be checked. This is typically done overall by either comparing the calculated steam flow rate versus the measured flow rate or the calculated weak black liquor flow rate versus the measured flow rate. The author judges an acceptable performance balance as one that predicts the measured value within 5%. This also assumes that the owner's instrumentation values used are sufficiently accurate. U and SE estimates using direct measurements can be cross-checked against mass and energy balance values to determine the accuracy of these rapid estimation methods.

Principle 4

Evaporator and concentrator systems designed for troubleshooting incorporate measurement, display values and trends, and give access to archival feed and product parameters. These can be used to ensure stream characteristics do not

exceed their expected ranges.

Characteristics of inlet streams to evaporator and concentrator bodies, be they external or internal to the evaporator set, have the potential to upset the system if their characteristics are outside of acceptable ranges. For all parameters listed in Table II, these can be measured directly. Examples showing the value of some of these parameters are:

- High weak black liquor temperature that resulted in high condensate conductivity and condensate sewerage.
- Erratic flow from a soap skimmer resulting in high dry product solids content and fouling of the product body.
- Partially flooded NCG line following an after-condenser that resulted in less vacuum at its discharge and lower throughput for an evaporator set.

Principle 5

Evaporator and concentrator systems designed for troubleshooting incorporate sampling stations and/or ports for black liquor and condensate sampling from each body.

Table III. Sampling Station and Port Locations

	Basic Operation	Troubleshooting
Sampling stations for feed and product liquor	Yes	
Sampling ports for liquor from each body		Yes
Sampling ports for condensates from each body		Yes
Level glasses or gauges for condensate in each body	Yes	
Ports for pressure / vacuum measurement in each vessel	Yes	

Ports for vacuum ahead of each ejector or vacuum pump	Yes	
Ports for pressure drop across demisters		Yes
Ports for pressure drop across recirculating liquor orifices		Yes
Ports for discharge pressure from pumps	Yes	
Ports for dP across surface condenser water-side		Yes
Ports for dP across vacuum system condenser(s) water-side		Yes

Manual sampling and testing of process streams is essential for a well operating evaporator and concentrator system. In some cases sampling and testing is used to obtain actual reported parameter values while in others it is used for cross-checks on instrumentation values. In either case safe and effective sampling stations are incorporated into the design. Additionally, sample ports are placed at locations where samples may be needed for consistent operation or to troubleshoot future problems. Table III shows a list of suggested sampling stations and ports.

Examples showing the value of some of these sampling stations and ports are:

- Liquor sampling between bodies to screen for low U's and tube leaks at a facility not having mass and energy balance capability.
- High condensate level in a body that resulted in poor heat transfer performance, contributing to lower black liquor throughput.
- Low water-side pressure drop in a surface condenser due to a baffle door inadvertently being left open that resulted in poor heat transfer, lower vacuum, and lower black liquor throughput.

Practical Considerations

The extent to which owners specify and/or suppliers require additional instrumentation for troubleshooting depends on many factors. These include capital and human resources, instrumentation philosophy, and maintenance philosophy. In

the event that minimal or no instrumentation for troubleshooting is chosen, owners typically rely on outside experts to troubleshoot and resolve evaporator and concentrator problems.

In some case additional instruments for troubleshooting are included in the design, but owner human resources are not available to use them effectively. However, these instruments will aid outside experts in identifying and resolving the problem. A potential problem with this approach is that instrument maintenance and calibration may be insufficient due to infrequent use. Consequently reliable and accurate information may not be readily available when needed.

Several owners, suppliers, or outside experts have the capability to access remotely evaporator and concentrator data information systems. Remote access capability allows those experienced in evaporator and concentrator technology to regularly monitor and/or rapidly assist facilities where technical and operations staff have limited experience and/or resources in this area. Additionally, it is used to spot potential instrumentation or operational issues, which, when corrected minimize the likelihood of performance degradation.

INDUSTRY EXAMPLES

The following industry examples are intended to illustrate use of the instrumentation suggested for inclusion in the design of evaporator and concentrator systems for troubleshooting purposes.

Example 1 Surface Condenser Fouling

Problem

The weak black liquor flow rate through a six-effect multiple-effect evaporator had dropped below the minimum acceptable rate for the set.

Observations

Operators noticed that a significant decrease in surface condenser vacuum had recently occurred. Outside testers found a low heat transfer coefficient for the surface condenser and a high NCG temperature from the surface condenser. Outlet water temperature remained essentially unchanged.

Key Measurements

NCG temperature from the surface condenser

Water temperatures in and out of the surface condenser

Measurements required for a performance mass and energy balance

Resolution

Cause Location. Water-side of tubes was fouled significantly.

Cause Clarification. Higher NCG outlet temperature implied fouling rather than an air leak.

Learning. Monitor NCG outlet temperature as an early warning of surface condenser fouling.

Example 2 Vacuum System Leak

Problem

A six-effect multiple-effect evaporator was periodically venting through a regulated PV-breaker.

Observations

Operations noticed periodic cycling of numerous evaporator process parameters including vacuum, outlet water temperature, body liquor temperatures, percent product solids, and steam flow.

Key Measurements

Surface condenser vacuum trends

Other parameters that were observed cycling with the vacuum

Resolution

Cause Location. Condensate line from main surface condenser to hotwell had a hole in it.

Cause Clarification. Corrosion through the down-leg allowed air to be periodically introduced when the condensate level was below the opening.

Learnings. Compare trends of multiple process parameters to identify patterns and dependencies.

Example 3 Lower Concentrator Product Solids

Problem

A single-effect concentrator integrated into a multiple-effect evaporator was only able to make dry product solids content at 5% solids below the desired target.

Observations

Operations had steam pressures of both the concentrator and the evaporator at maximum values.

Operations noted inlet weak liquor dry solids content was in an acceptable range.

Outside testers measured process parameters required for a performance mass and energy balance.

Key Measurements

Steam pressure

Those parameters required for a performance mass and energy balance

Calculation of heat transfer coefficients for each body

Resolution

Cause Location. Entire evaporator and concentrator system

Cause Clarification. Comparison of measured heat transfer coefficients with those from design showed the system as configured was operating at maximum conditions. That is, to achieve an increase in dry product solids content, the

concentrators needed to operate at lower black liquor throughput or add additional heat transfer area to sustain target dry product solids content.

Learnings. Maximum liquor throughput and energy-related performance of a well maintained and instrumented evaporator and concentrator set can be determined. These can be used to justify improvements to achieve target throughput and performance.

Additional Examples

Troubleshooting approaches to problems in black liquor evaporators that are accompanied with examples and practical solutions are available in the literature [4, 5, 6, 7, 8, and 9].

CONCLUSIONS

Two instrument groups are defined for design of evaporator and concentrator systems. These are 1) those required for basic operation and 2) those used primarily to troubleshoot operating problems. The primary focus of the later group is to improve the effectiveness of troubleshooting operational problems. The main objective of troubleshooting instruments is to define individual body performance. Five design principles were defined for identifying instruments used for troubleshooting. The extent to which troubleshooting instruments are included in the design is based on owner specifications and supplier requirements. Industrial examples are used to illustrate the value of troubleshooting instruments to guide, confirm, and / or assist in resolving operational problems.

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2.6.2 CONDENSATE SEGREGATION

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Introduction

Foul condensate stripping is energy intensive when BOD (biological oxygen demand) removal is required; for high efficiency removal, the steam usage is 20% by weight of the foul condensate to be stripped. For example, a 1000 L/min stripper operated at high efficiency for BOD stripping will use about 12,000 kg/h. Reducing the steam consumption will reduce stripping efficiency.

Methanol is typically used as a surrogate for BOD as it typically makes up a very large fraction (80 to 90%) of the BOD in foul condensate. At 20% steaming rates, methanol removal is greater than 90% and TRS removal is over 99%.

Contaminated condensates are generated in a number of locations in the mill. Two of the major areas are the digester area and the multiple effect evaporator systems. If the mill were to strip all of the contaminated condensate generated in these two locations, the system would be prohibitive from both an equipment and operation cost perspective.

Principles of Condensate Segregation

In order to minimize the amount of condensate that needs to be stripped, condensing equipment is typically partitioned into first and second condensing stages. The result is the formation of two types of condensates. The first type, from the first stage, is a “contaminated” condensate that contains some methanol and very little TRS. The second type of condensate from the second stage, “foul” condensate, is highly contaminated in both methanol and TRS and must be treated by stripping. The process which separates these two types of condensate is called condensate segregation.

The main contaminants, methanol and TRS, have a much higher volatility than water. If a water vapour stream containing these components is allowed to condense in series on two distinct surfaces,

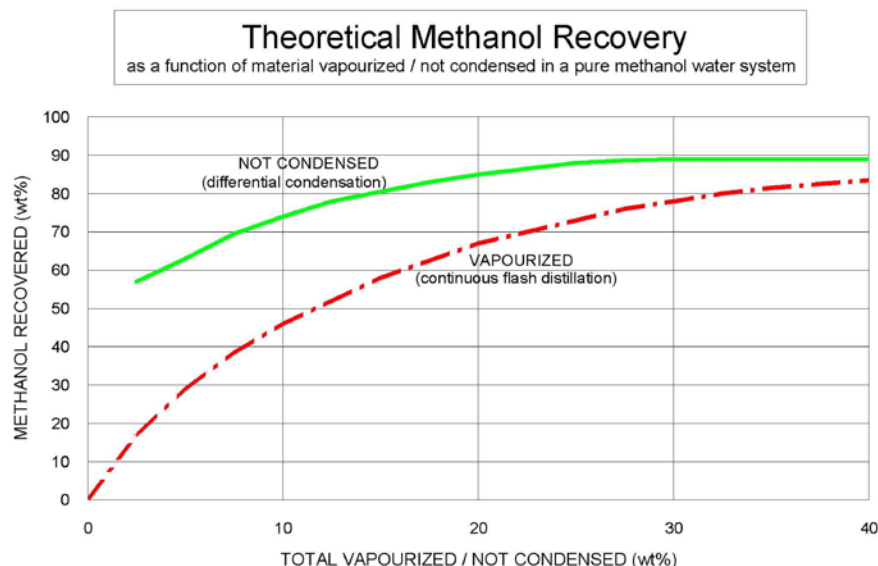
a larger percent of the more volatile components will carry on to the second surface. The technical term for this phenomenon is differential condensation.

The amount of separation is a function of the percent of the vapour that is condensed on the second surface. For example, referring to Figure 1, if 5% is condensed on the second surface, then of the inlet methanol in the vapour, approximately 65% will be carried to the second surface. If the separation is increased to 15% blow through to the second surface, the percent of methanol carried through will be approximately 80%. In this way a large fraction of the methanol can be captured in a relatively small stream.

Because the TRS components are much more volatile than methanol, the amount of separation will be significantly higher. At the 5 and 15% condensed on the secondary surface, depending on the composition of the TRS mixture, about 95 and 99% of the TRS will be contained in the foul condensate portion, respectively.

Separation can also occur when part of the liquid is flashed. For example, referring to Figure 1, if 5% of the liquid is vapourized, then of the inlet methanol in the liquid, approximately 30% will also be vapourized. If the flash is increased to 15% of the liquid, the percent of methanol carried in the vapour will be approximately 55%. The technical term for this phenomenon is continuous flash distillation.

The theoretical amount of methanol separated is based on a pure methanol - water system. In practice, because Kraft mill condensates are multi-component in nature, methanol separation efficiencies may be slightly reduced.



Evaporator Condensate Segregation

It is well documented that a large fraction of the TRS and methanol is flashed off in the first two stages of weak black liquor evaporation. For example, in the first liquor flash of a typical evaporator plant, about 15% of the total liquid is flashed off; the flash vapour will contain about 60% of the total methanol that was in the black liquor. Of the remaining 40% methanol in the black liquor, 60% of this (i.e. 60% of 40% or 25% of the initial total) will be removed in the second liquor flash; therefore, 60% plus 25% or 85% of the initial methanol in the black liquor will be contained in the condensate from the first two liquor flashes.

The condensing vapour from the first two liquor flashes is typically segregated. This means these vessels are partitioned into first and second condensing stages typically containing 85% and 15% of the heat transfer surface, respectively. To continue our example, about 80% of the methanol from the first liquor flash (containing 60% of the methanol) will be recovered in the foul condensate fraction, i.e. about 48% of the total methanol. As well, 80% of the methanol from the second liquor flash containing 25% of the methanol will be recovered in the foul condensate fraction, i.e. about 20%. Therefore, for this segregation system collecting 15% of the condensate from the first two liquor flashes, about 48% plus 20%, or 68% of the methanol contained in the black liquor can be captured in a very small foul condensate stream.

To apply the theory behind condensate segregation, Table 1 lists condensate segregation design features and equipment included for a typical new installation and for a retrofit of an older evaporator set.

Table 1: Evaporator Condensate Segregation Equipment

Equipment	New System	Retrofit Existing System
Weak liquor feed	No split feeding of liquor, all liquor fed to penultimate effect	No split feeding of liquor, all liquor fed to penultimate effect
Last effect	Minimum last 15% of condensing surface segregated for foul condensate	Add external liquor heater, size limited to about 5% condensing
Surface condenser	Minimum last 15% of condensing surface segregated for foul condensate	Add secondary surface condenser in series after existing primary surface condenser; size for minimum 15% of vapour
NCG venting	Last two or more heater bodies vent to foul side of surface condenser	Last two or more heater bodies vent to secondary surface condenser
Vacuum system	Two stage with indirect intercondenser and aftercondenser	Two stage with indirect intercondenser and aftercondenser
Condensate flash tanks	Segregated flash areas inside effects for combined (process) and contaminated condensates for polishing	Add flash tanks on combined (process) and contaminated condensates for polishing

Digester Condensate Segregation

The principles of condensate segregation can be applied by having a primary flash steam condenser perform 85% of the heat duty and using a secondary condenser to condense about 15% of the flash vapours. After segregation, only the foul condensates from the secondary condenser should be stripped.

The benefits of segregating the digester condensates include:

- (a) Reduced foul condensate loading to the stripper.
- (b) Reduced foul condensate loading for turpentine decanting (as turpentine is also very volatile and 99+% will be contained in the secondary condensate).

- (c) Improved heat recovery, i.e. instead of heating cold mill water, 105°F warm water can be heated to 185 to 190°F in the primary condenser.

Kraft Mill Condensates

In general, there are three types of Kraft mill condensates - combined (or process), contaminated, and foul (see Table 2).

TABLE 2: Types of Kraft Mill Condensates

	Combined (Process) Condensates	Contaminated Condensates	Foul Condensates
Typical TRS concentrations	40 ppm	150 ppm	600 ppm
Typical methanol concentrations	100 ppm	500 ppm	3000 ppm
Typical volumes	8000 lpm	4000 lpm	1500 lpm

Combined condensates are the cleanest and the largest volume of the three. Combined condensates are often reused at brown stock washing and/or at recausticizing without any further treatment. Typical combined condensates include:

(a) The front end of the evaporators, i.e. the 2nd effect through to the penultimate effect of the evaporator plant.

(b) Stripped condensate. The cleaned condensate from a foul condensate stripping system.
Requirements: minimum 20% stripping steam.

(c) Primary flash steam condenser. The condensates from the first condenser of a two stage digester condensing system. Requirements: primary condenser must be operated at a vent temperature of no less than 99 °C (210°F); in this case the mill would have a single condenser and send all of the condensate to stripping.

Contaminated condensates are in between combined and foul condensates in contaminant concentration. These condensates are generally too odourous for reuse in the mill and are usually used for level controlled makeup to the stripper feed tank. Typical contaminated condensates include:

(a) The condensates from the primary split of the last effect of the evaporators. Requirements: the liquor feed effect is to penultimate effect; up to 85% is condensed in the primary surface.

(b) The condensates from the primary split of the surface condenser. Requirements: up to 85% of the vapour is condensed in the primary surface.

Foul condensates are the most odourous and BOD concentrated condensates found in the kraft mill. Sending to secondary treatment without in-plant treatment such as steam stripping will result in higher odour and BOD loading at the lagoons. Typical foul condensates include those from the back end of the evaporator including segregated portions from the first two (and sometimes three) liquor flashes:

(a) The condensates from the secondary split of the last effect of the evaporator. Requirements: 15 to 25% of the vapour is condensed in the secondary surface.

(b) If the last effect is not segregated, then the condensates from an external liquor heater off the last evaporator effect. This is not as effective as segregating the last effect surface area as listed in (a) above as liquor heating duty limits the vapour condensed to about 5%.

(c) The condensates from the secondary split of the evaporator surface condenser or a secondary surface condenser. Requirements: 15 to 25% of the vapour is condensed.

(d) Evaporator vacuum system. Condensates from the two stage vacuum system. Requirements: indirect precooler, intercondenser, and aftercondenser.

(e) Secondary flash steam turpentine condenser. The condensates from the second condenser of a two stage condensing system. The condensates are first sent to turpentine decanting first. Requirements: primary condenser must be operated at a vent temperature of no less than 210°F.

(f) NCG/SOG line drains. Line drains from the NCG and SOG headers are collected and pumped to the stripper feed tank.

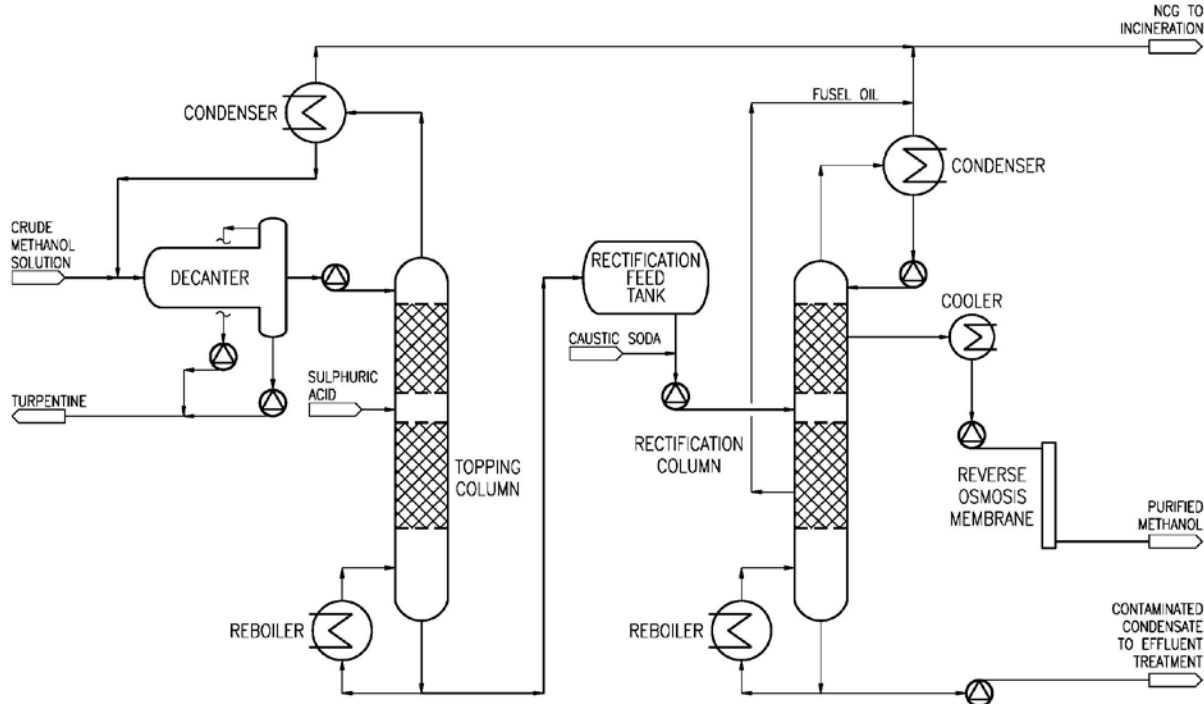


Figure 1: Methanol Purification System Process Flow Diagram

There are numerous methanol purification systems in service around the world and their operation is well understood, but this process had not previously been applied to this feed stream. Recovery of methanol from Kraft pulping condensates has several unique characteristics that inhibit separation by distillation:

- Azeotropes of methanol and organic contaminants
- Immiscibility of terpenes
- Dissociation of hydrogen sulphide and methyl mercaptan
- Ammonia reacting to produce ammonium sulphate (fouling)

A.H. Lundberg has developed and patented features to deal with each of these design issues.

Modular Construction

The Methanol Purification System is supplied in modular form (see Figure 2). All equipment, inter-connecting piping, hand valves, instruments, control valves, insulation and access stairways and ladders are supplied pre-assembled on steel skids.



Figure 2: Methanol Purification System Modules

PROCESS DESCRIPTION

The presence of azeotropes between methanol and some organic contaminants require that the SOG generated in the stripping system be no higher than 40wt% methanol to ensure separation of the contaminants in the topping system. Effective distillation requires a consistent feed stream; therefore, the SOG quantity as well as quality must be stable. In some cases, this requirement will necessitate upgrades to the stripping system.

The SOG is condensed and stored as a crude methanol solution. Typically about one hour of retention is recommended; this gives operational flexibility and provides volume to homogenize the feed flow to the methanol purification system.

Topping System

The crude feed solution is composed mainly of methanol and water, but it also contains over 150 different inorganic and organic “impurities” including various aliphatic alcohols and ketones, and terpenes, along with many sulphur and nitrogen compounds. The topping column is designed to strip out the “low boilers” and noncondensibles, including the odorous total reduced sulphur (TRS) compounds, ammonia, and some ethers, ketones, and aldehydes.

The crude methanol solution is first introduced to the red oils (turpentine) decanter for separation and removal of terpenes. From there it is pumped to the top of the topping column, where it flows down through the packing countercurrent to the stripping steam, thereby driving off the volatile constituents to be removed from the less volatile (methanol, water, etc). Heat is added to the topping column via an integral bayonet type steam reboiler. The overhead vapours from the column are condensed in the topping reflux condenser with the heat recovered into cooling water. The low boilers and noncondensibles from the reflux condenser are vented to the NCG system while the reflux condensate is drained to the decanter.

Contaminants produced during the kraft pulping of wood include the ionizable sulphur compounds, hydrogen sulphide and methyl mercaptan. The removal of these compounds by distillation is directly related to the pH of the liquid phase in the column. For example, at a pH above about 6, hydrogen sulphide begins to dissociate, and at a pH above about 9, this occurs with methyl mercaptan (see Figure 3). In their dissociated form, these compounds do not exert a vapour pressure and therefore cannot be removed by distillation.

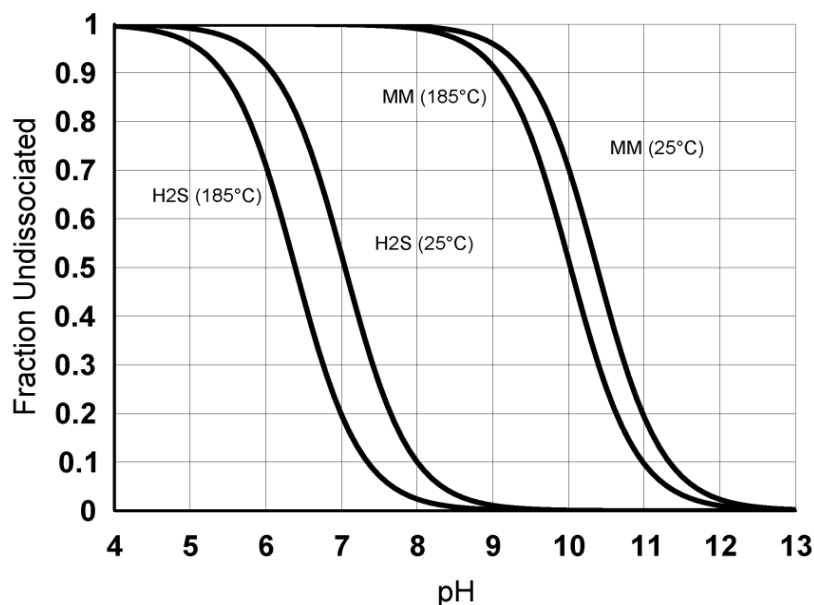


Figure 3: Dissociation of Hydrogen Sulphide and Methyl Mercaptan

The pH of condensed SOG is typically slightly alkaline at about 9 to 10. To ensure that ionizable sulphur compounds are removed, sulphuric acid is added to the topping column. Due to the presence of ammonia, the acid cannot simply be added to the liquid feed to the column as it will react with ammonia producing ammonium sulphate.

Ammonium sulphate is very soluble in water (about 0.7 kg dissolved per kg water at 25°C and 1.0 kg per kg at 100°C), but essentially insoluble in methanol. To ensure that no ammonium sulphate forms, potentially fouling the topping column, the sulphuric acid is added at the mid-point of the column; the top section of the column is therefore used to strip out the highly volatile ammonia before it has a chance to react with the acid.

Sufficient acid is added to lower the pH to between 5 and 6, thereby releasing the newly undissociated hydrogen sulphide and methyl mercaptan. They are stripped out in the bottom section and then must rise up through the top section of the column. These gases, when reabsorbed, are weak acids and will cause a slight decrease in the pH in the top section of the column, but no ammonium sulphate precipitate will form due to lack of any sulphate ions in solution.

Red Oils Decanting

Terpenes, naturally present in wood, are extracted during kraft pulping, and due to their extremely high relative volatility, will be stripped from stock and liquor streams and collect in foul condensate. Those not decanted in the turpentine recovery system will make their way to the stripping system and subsequently into the SOG. The mixture of terpene compounds is referred to as turpentine, or crude sulphate turpentine, or, when heavily contaminated with TRS compounds, "red oils", due to their distinctive red hue. The terpenes are almost totally immiscible in the methanol solution; therefore a red oils decanter is required. If turpentine is disposed of through incineration, then the red oils can be pumped to the turpentine recovery system. If turpentine is sold, the red oils should be processed separately as their high TRS content will degrade the quality of regular turpentine.

Rectification System

The underflow from the topping column flows by gravity to the rectification feed tank. This tank is sized for one hour of retention; therefore, it provides operational flexibility by separating the topping system and rectification system processes. The tank also provides volume to homogenize the methanol solution feed flow to rectification.

The methanol solution from the feed tank is pumped to the rectification column, and flows down through the stripping section countercurrent to the stripping steam, thereby driving off the volatile constituent (methanol) to be removed from the less volatile (water, ethanol, etc). This column is designed to separate the water and other "high boilers" into the underflow, and methanol vapour into the overhead vapour flow. Heat is added to the rectification column via an integral bayonet type steam reboiler.

"Intermediate boilers," those with boiling points between methanol and water, referred to as "fusel oils," are drawn off from the bottoms column below the methanol solution introduction point. The fusel oils include some higher alcohols (primarily ethanol) and higher ketones (primarily methyl ethyl ketone). They are combined with the underflow from the column, but can in future be recovered separately if desired. Typically a fusel oil flow sacrificing about 2 to 3% of the methanol must be bled off to ensure that the methanol product remains free of ethanol contamination. The contaminated condensate from the rectification column is pumped away to effluent treatment.

The vapour from the top of the stripping section passes up to the rectification section. Vapour from the top of the column is condensed in the rectification reflux condenser, with low boilers and noncondensables (not removed in the topping column) vented to the NCG system. The heat introduced by the reboiler steam is recovered into cooling water in the rectification reflux condenser. The reflux condensate is pumped back to the rectification column.

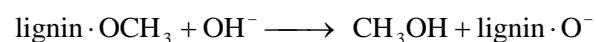
Methanol Product

The methanol product is drawn off slightly below the top of the column. The methanol is withdrawn in sufficient quantity to maintain the temperature (and therefore methanol concentration) profile in the column. The methanol is cooled and then pumped through a reverse osmosis (RO) membrane for final polishing. The 99.85wt% methanol product is then directed to storage. If necessary, the methanol product can also be recycled back to the stripper feed tank, crude methanol tank, rectification feed tank, or disposed of through incineration in a boiler or the lime kiln.

METHANOL RECOVERY AND USAGE

Methanol Formation

Methanol is formed as a by-product of the kraft pulping process; in the digester, the hydroxyl ion reacts with a lignin methoxyl group:



The amount of methanol produced is dependent on:

- Wood species: in general lignin methoxyl groups are more numerous in hardwoods than softwoods. Therefore more methanol is formed from pulping of hardwoods.
- Hydroxide concentration: higher alkali charge will produce more methanol.
- Temperature: higher cooking temperatures will produce more methanol.
- Time: longer cooking times will produce more methanol.

Some methanol is also formed from the acid-catalyzed methanol elimination from hemi-cellulose, while naturally occurring methanol is freed from the biomass at the start of pulping.

Estimates are provided in Table 1 for methanol formation in the digester when producing bleachable grade kraft pulp from various softwoods and hardwoods.

Table 1: Methanol Formation in Kraft Pulping

Wood Species	Methanol Formation (kg/ADT Pulp)
Western Hemlock	6.6
Douglas Fir	7.5
White Spruce	8.2
Southern Pine	8.4
Aspen	8.6
Birch	9.9
Maple	10.2
Oak	11.5
Bass	11.8
Sweetgum	13.5

Methanol Capture

In a continuous mill, the methanol leaves the digester with the extraction liquor or with the brown stock. Methanol is relatively volatile compared to water; therefore, a large fraction will tend to flash-off in the extraction liquor flash tanks and collect in the flash steam condenser foul condensate. In a batch mill, a large fraction of the methanol will leave the digester with the relief steam and collect up

in the relief steam condenser foul condensate. Additional methanol is captured in the foul condensate from the blow heat recovery system.

Some methanol will be carried away by the digester Concentrated Noncondensable Gas (CNCG) collection system and some will be lost to the Dilute NonCondensable Gas (DNCG) system during brown stock washing. The methanol that is captured in the washing filtrate will eventually end up in the weak black liquor feed to the evaporators. Depending on the level of condensate segregation, anywhere from 50 to 90% of the methanol in the weak liquor can be captured in the evaporator foul condensate, with the remainder lost to the combined condensate and the CNCG vent.

Up to 80% of the methanol formed in the digester will end up in the foul condensate. In a well operated stripping system, up to 95% of the methanol will be removed from the foul condensate and captured in the SOG, with up to 90% of this recovered as purified methanol. Overall, up to about 70% of the methanol formed in the digester can be recovered as purified methanol.

Methanol Product

Methanol at a purity level of 99.85wt% can be worth over five times as much as a traded commodity than in crude form as a combustion fuel displacing light fuel oil or natural gas. Typically about 30 to 40% of the methanol produced at a bleached kraft mill can be used internally for production of chlorine dioxide (ClO_2), with the balance for external sale.

2.6.3 THE BASICS OF FOUL CONDENSATE STRIPPING

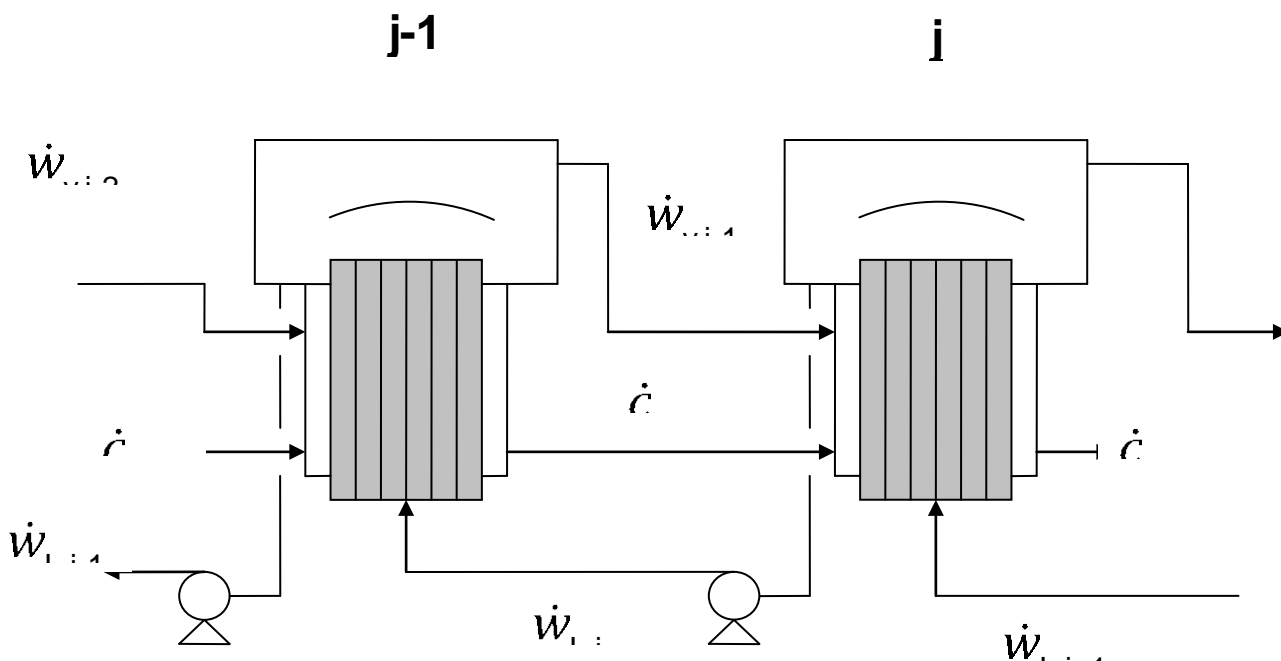
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INTRODUCTION

Foul condensates from the digesters and evaporators contain reduced sulfur gas and organic compounds, such as methanol (MeOH) and turpentine, which contribute greatly to pulp mill water pollution in the form of Biochemical Oxygen Demand (BOD) and toxicity, and air pollution in the form of Volatile Organic Compounds (VOC), and odor.

For this reason, foul condensates are collected and treated by stripping, using either air or steam, to remove the pollutants. In many cases, the pollutants can be used economically as a fossil fuel substitute.



This section covers the condensates that are collected, various stripping methods, types of stripping columns, basic stripping theory, operating problems, the latest operating

Table I. Typical Pollutant Loads in Foul Condensates

Bleached Kraft Mill

Based on Unbleached Digester Production

Source	Total Flow		MeOH		Turpentine		TRS	
	kg/tonne	lb/tonne	kg/t	lb/t	kg/t	lb/t	kg/t	lb/t
Batch Digester Mill (Softwood)								
Digester accumulator overflow	1125	2250	4.0	8.0	0.50	1.0	0.20	0.40
Turpentine decanter underflow	250	500	1.5	3.0	0.50	1.0	0.15	0.30
Total evaporator condensate	7000	14,000	4.2	8.4	0.25	0.5	1.00	2.00
Continuous Digester Mill (Softwood)								
Turpentine decanter underflow	450	900	2.5	5.0	0.50	1.0	0.12	0.24
Total evaporator condensate	8000	16,000	7.5	15.0	0.50	1.0	1.20	2.40

procedures, and disposal of the stripper off gas (SOG) which contains the pollutants removed.

The Cluster Rules resulting from the Clean Air Act Amendments (CAAA) of 1990, will require the collection and treatment of “Kraft process condensates”, better known as foul condensates or combined condensates.

Steam stripping is one of the treatments accepted by the Environmental Protection Agency (EPA). The others are hard piping to biological treatment, reusing the condensates in a process where the vents are collected and incinerated, or any other process that can meet the required removal efficiencies.

Although the rules allow various methods of determining removal efficiencies, bleached mills have to collect selected condensates and treat them to destroy 10.2 pounds of methanol (MeOH) per oven dry ton of pulp (ODTP) (5.1 kilograms per oven dry metric ton of pulp). Unbleached mills will have to destroy 6.6 pounds per ODTP (3.3 kg/metric ton).

Most mills built or upgraded since 1980 have had a condensate stripping system installed as an integral part of the overall mill process. Many older mills have now added a foul condensate stripper to meet the new EPA Cluster Rule.

WHY STRIP FOUL CONDENSATES?

The prime reason for stripping foul condensates is pollution control. Foul condensates can contain 14 to 20 lb (7 to 10 kg) of BOD, 2 to 4 lb (1 to 2 kg) of turpentine and 2 to 4 lb (1 to 2 kg) of TRS per ton (tonne) of pulp (Table I).

If these foul condensates are untreated, they cannot be reused in the mill and are therefore seweraged, putting a high load of BOD and toxicity into the secondary treatment system and frequently causing air pollution problems, as the TRS and MeOH is released to the atmosphere by flashing off from open sewers.

By collecting these foul condensates and stripping them, most of the pollutants can be removed and burned, reducing the pollution load to the air and to the secondary treatment system.

In most cases, the stripped condensates can be reused in the mill for such purposes as brown stock washing and make-up water in the recaust area. Thus, stripping also has the

potential to reduce the total mill water demand. Stripping will also be an integral part of the effluent free mill of the future.

Fortunately, the stripped compounds are easily burned. Heat of combustion values for stripper overheads in the order of 500,000 KJ/tonne (475,000 BTU/Ton) pulp have been reported (Table II).

These stripper overheads can be transported as a gas, or condensed and transported as a liquid and used to replace fossil fuel in lime kilns, boilers and incinerators. In most cases, the net heat released by burning these pollutants is greater than the heat energy required to operate the stripper, and can be used to help economically justify installing the stripping system.

Table II Heat Value of Pollutants

Pollutant	Net Heat of Combustion	
	kJl/kg	Btu/lb
MeOH	21,580	9066
Alpha-pinene	40,890	17,200
H ₂ S	15,620	6565
CH ₃ SH	26,690	11,212
CH ₃ SCH ₃	31,580	13,268
CH ₃ SSCH ₃	24,150	10,148

WHAT CONDENSATES ARE STRIPPED?

The Cluster Rule requires collection and treatment of the condensates from the digester system, the turpentine recovery system, the evaporator systems, the HVLC (dilute NCG) collection system, and the LVHC (concentrated NCG) system.

Batch Digester Blow Steam Condensate

The condensates from condensed blow steam are rich in methanol and TRS. In order for these pollutants to be collected, however, the blow steam system must function correctly. If the blow steam system is undersized or operated incorrectly, much blow steam is vented with much of the methanol and TRS vented as well.

In some mills, fresh water is added to the accumulator to keep the bottom temperature low. This dilution of the blow steam condensate makes collection of these condensates undesirable.

In a modern blow steam condensing system, the direct contact primary condenser is followed by an indirect secondary/tertiary condenser. It has been found that up to 80% of the pollutants can be “segregated” into the 10 - 15% of condensed blow steam that comes from the secondary/tertiary condenser. This concentration of the pollutants in a smaller flow greatly improves the efficiency and economics of stripping.

The Cluster Rule will allow the treatment of only the segregated stream, provided that at least 65% of the methanol in all the blow steam condensates is contained in the segregated stream.

Batch Digester Relief Steam

During the cook, non-condensable gases (NCG) are vented from the digester. The NCG contains considerable amounts of steam, MeOH, and frequently turpentine. This steam must be condensed in order to collect the NCG. The condensates so formed must be collected and treated. As above, this condensation can be done in two stages, and only the rich or

segregated stream collected and treated, providing at least 65% of the methanol in the relief gas is in the segregated stream.

Continuous Digester Flash Steam

During the cook, hot, pressurized liquor is removed from the digester and flashed to atmospheric pressure. The flash steam contains considerable amounts of NCG, MeOH, and frequently turpentine. This steam must be condensed in order to collect the NCG. The condensates so formed must be collected and treated. As above, this condensation can be done in two stages, and only the rich or segregated stream collected and treated, providing at least 65% of the methanol in the flash steam is in the segregated stream.

Turpentine Decanter Underflow

In softwood mills, batch digester relief steam condensates and continuous digester flash steam condensates go to a turpentine recovery system. The underflow from the turpentine decanter is a relatively low flow that is rich in methanol, turpentine and TRS, making this foul condensate a prime candidate for stripping.

Evaporator Condensates

The combined foul condensates from the multiple effect evaporators contain methanol, turpentine and TRS removed from black liquor during evaporation. The evaporators, in effect, act as strippers to remove these pollutants from the black liquor.

It has been found that most of the pollutants (80%+) will collect in the condensates of the effects and condensers following the effects where the weak black liquor is fed. For example, if weak black liquor is fed to effects 5 and 6 of a six-effect evaporator, the pollutants will be found in the condensates from the 6th effect and the surface condenser. The Cluster Rule allows splitting out such condensates for treatment, thus reducing the amount of evaporator condensate to be treated by about two thirds.

In modern multiple effect evaporators, especially those of the falling film design that have been supplied since 1980, the majority of the pollutants can be collected in 15% or less of the total condensate. This is primarily done by using two-stage condensing with internal

condensate segregation. This, as mentioned above, greatly reduces the condensate to be treated, and thus improves the efficiency and economics of stripping.

In general, there will be four condensate streams generated in a modern falling film evaporator. The first is clean steam condensate from the first effect. The second is combined condensates from the middle effects, and these will have methanol in the 100 ppm range. The third stream is contaminated condensates from the lean steam of segregated condensates from the feed effects, and these will have methanol concentrations in the range of 1000 ppm. The final stream is foul condensates from the rich stream of segregated condensates, and the methanol will be in the 5000 ppm range.

The Cluster Rule will allow segregation of condensates, providing that 65% or more of the methanol in all the evaporator condensates is in the segregated stream.

NCG System Condensates

The condensates formed in NCG systems, although small in volume, are very concentrated in methanol and TRS compounds, and contribute to odors in the mill area if drained into open sewers. The Cluster Rules may require collection and treatment of these condensates.

METHODS OF STRIPPING

There are two methods of stripping used in Kraft pulp mills today. One uses air and the other uses steam. Both of these methods can be further subdivided into two types of strippers; stripping for TRS only and stripping for methanol (BOD). TRS is easily stripped, whereas MeOH is more difficult, consequently making the latter systems bigger and more complex. Further, the new Cluster Rules will not accept air stripping, or stripping for TRS only.

Air Stripping for TRS

This is the most basic form of stripping and is generally used to remove TRS from condensates so that they do not create odor problems in the secondary treatment system (Figure 1). In this case, the foul condensates are stripped by air moving counter-currently in a stripper column.

About 3 - 5 weight % of air on condensate is required. Higher air ratios may be necessary if the TRS concentration is high, in order that the stripped gases are well below the Lower Explosive Limit (LEL) in the overheads.

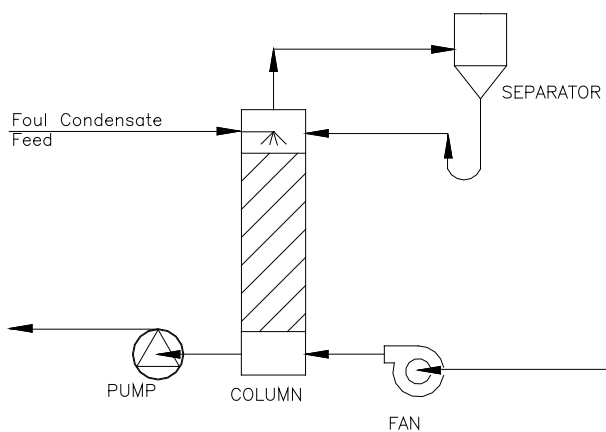


Figure 1. *Air Stripper for TRS*

The two factors that have the greatest effect on air stripper efficiency are temperature and pH (Figure 2). A pH of seven or less and a condensate temperature of 50 °C (120 °F) or more

are required for good stripping efficiency. Disposal of the high volume of warm moist overheads is the major disadvantage of this system.

Disposal is usually done by burning in a boiler, kiln or incinerator. The major advantages of this system are its simplicity and low cost.

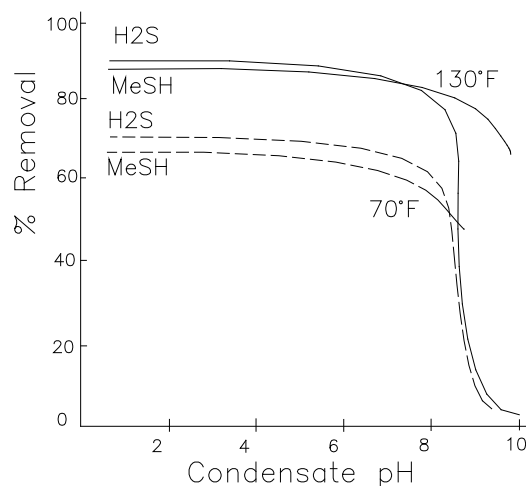


Figure 2. *Air Stripper Efficiency*

Air Stripping for BOD

This system is similar to air stripping for TRS, except that much higher quantities of air are required, in the range of 16 - 20 weight % air on condensate. Temperature is also very important and temperatures in excess of 70°C (160°F) are required for good efficiency. This form of stripping is not practical, and is not practiced.

Steam Stripping for TRS

This system is similar to the air stripper except that two heat exchangers are added to the system (Figure 3). It is necessary to preheat the foul condensate before stripping, otherwise the stripping steam would be condensed by the cold condensate. This preheating is done by heat exchanging with the hot stripped condensate.

The second heat exchanger is used to condense the steam out of the stripped TRS gas before it is incinerated. The condensate is returned to the top of the stripping column. About 3 - 5 weight % of steam is required on condensate.

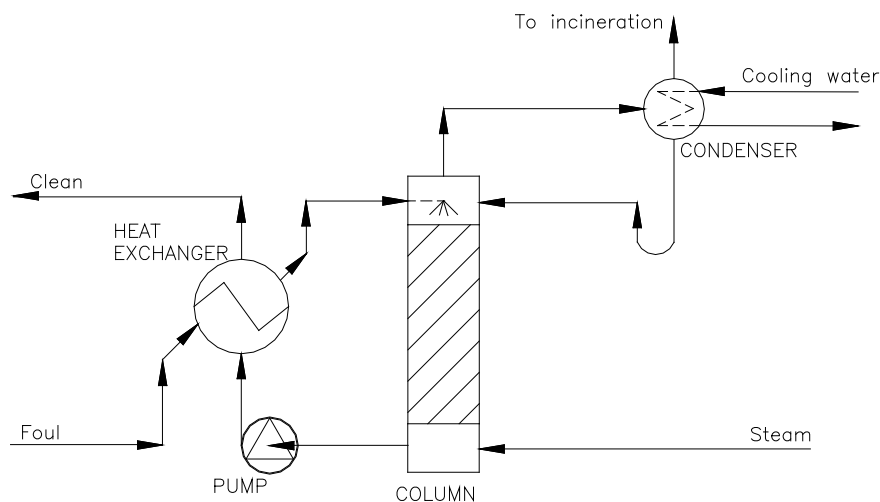


Figure 3. *Steam Stripper for TRS*

As with air stripping, efficiency is pH dependent. Despite the disadvantage of higher capital and operating cost, disposal of the overheads is much simpler. Operating costs can be reduced if some use can be made of the hot water generated in the overhead condenser.

Steam Stripping for BOD

This system is similar to the steam stripper for TRS except that the overhead condenser becomes a reflux condenser and the top of the column becomes a distillation column to concentrate the methanol (Figure 4).

The steam requirement increases to 15 - 20 weight % steam on condensate (Figure 5).

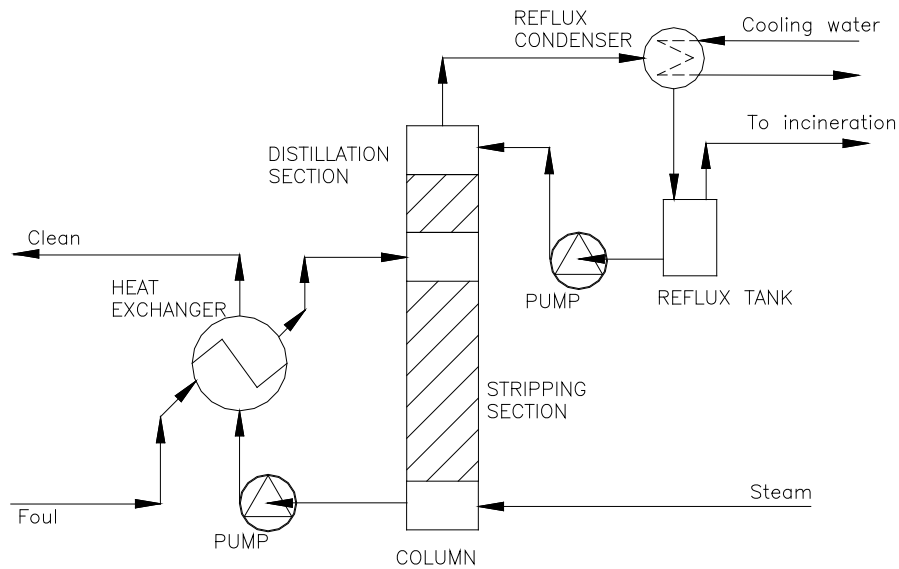


Figure 4. *Steam Stripper for B.O.D.*

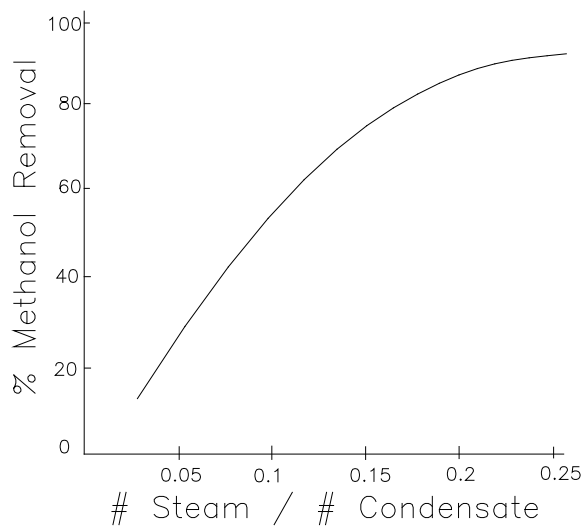


Figure 5. *Distillation Efficiency*

These systems have the advantage of high efficiency, producing an overhead that is usable as a fuel. The major disadvantages are high capital and operating costs. The operating costs can be reduced if some use of the heat from the reflux condenser can be found such as preheating boiler feed water, or making clean hot water for a bleach plant.

The reflux tank shown in Figure 4 is optional, and many systems do not have them. Instead, the reflux condenser is positioned above the column, and reflux flows by gravity back to the column.

There are examples of this type of stripping system at the mills in Palatka, FL., and Brunswick, GA.

TYPES OF COLUMNS

In general, two types of columns are used for stripping; packed columns and valve tray columns.

Packed columns are generally cheaper, especially for small diameter columns. Packing material is either stainless steel pall rings or corrosion resistant plastic saddles. The disadvantages of packed columns are poor turndown ratio and plugging of the packing by fibers carried in the condensate.

Valve tray columns are an advanced form of a bubble cap tray column. They have the advantages of maintaining high efficiency over a wide operating range, a constant pressure drop at varying vapor load rates and self-cleaning of the trays. These advantages normally offset the higher capital cost of the valve tray column.

As a general rule, packed columns are used for TRS strippers and very small BOD strippers, while valve tray columns are used for large BOD strippers. The valve tray column has become the standard under the Cluster Rule.

INTEGRATED COLUMNS

In order to reduce the capital and operating costs of steam type BOD strippers, they are normally integrated into a set of multiple effect evaporators. There are two basic ways to do this.

Fully Integrated

The stripping column is placed between two effects, usually the No. 1 and No. 2. Steam from the No. 1 effect is run through the stripper and then condensed in the No. 2 effect. The No. 2 effect is the reflux condenser, saving this capital cost. The stripping steam is essentially free.

Figure 6 is a schematic depicting a fully integrated stripping system.

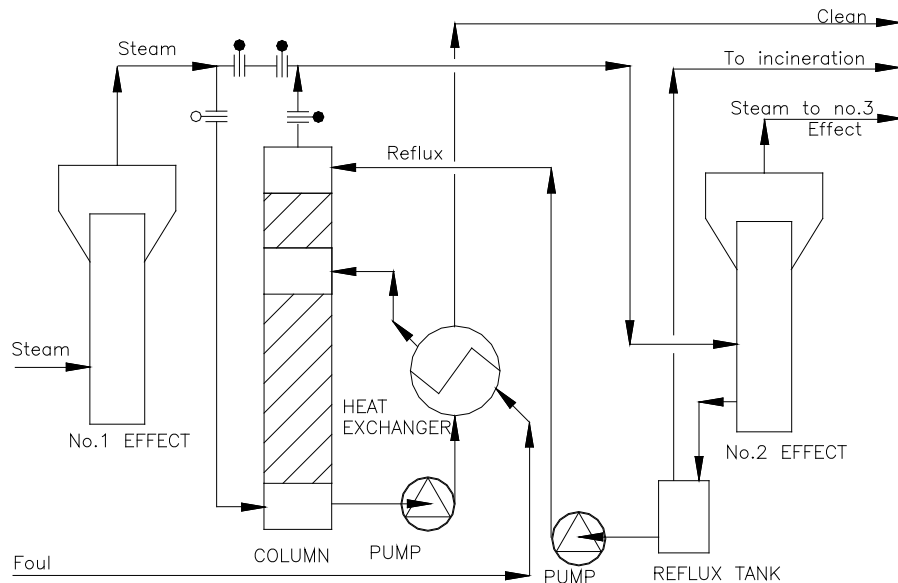


Figure 6. *Fully Integrated Column*

However, there is an efficiency loss of about 12% in the evaporators when the column is fully integrated. In a system of this type, for every kJ or BTU lost by integration, two or more kJ or BTUs are returned by the heat value from burning the stripped BOD.

This type of full integration with respect to the evaporators can be used when a stripper is integrated into an existing mill, and where condensate flows are high compared to the relative size of the evaporators.

Partially Integrated

In the case of a new mill with reduced foul condensate flows, and where the stripper is part of the initial evaporator design, only partial integration is required.

In this case, only part of the evaporator steam flow is used for stripping and the overheads are condensed in either an external preheater or in a dedicated internal section of a subsequent evaporator effect. Such systems have a much better energy return than fully integrated systems.

There is an example of this type of stripper at the mill in Leaf River, MS. Figure 7 is an illustration of a partially integrated stripping system.

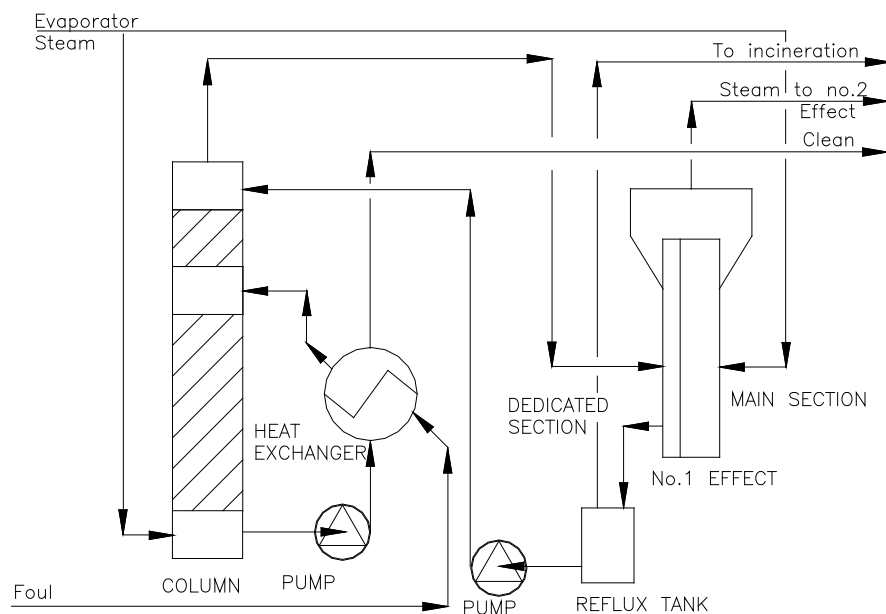


Figure 7. Partially Integrated Column

COLUMN OPERATION

Stripping columns are generally easy to operate although control schemes for strippers integrated into evaporators can be complex due to their interdependence on evaporator operation. Some problems in column operation are as follows:

Foaming

Liquor in the foul condensate can cause the condensate to foam when air or steam is blown through it. This results in a flooded column where the condensate is carried out the top of the column and steam or air flow is stopped.

This problem is handled by measuring the conductivity of the foul condensate. A high conductivity indicates the presence of liquor and the condensate can be dumped before it gets to the column.

This problem will show up as loss of steam flow to the stripper, or high level in the reflux tank.

At worst, the foam will carry through the stripper off gas (SOG) system and into the incineration point, where it may extinguish the flame, or cause other damage.

Unstable Operation

In steam strippers, where steam use is controlled and based on condensate flow, every change in flow upsets the system for several minutes. In a continuous change situation, such as if the condensate feed flow is based on level control of a foul condensate storage tank, the system remains unstable. Therefore, flow changes should be made on a stepwise basis and made as seldom as possible. In the case of an integrated column, the evaporators used should be run as smoothly as possible.

If a mill has more than one set of evaporators, the set with the stripper should carry a constant base load while the other set takes up the production rate changes. Unstable operation can also cause problems if the stripper gas is burned in a kiln or an incinerator. Rapid changes in MeOH load can upset the heat balance in the kiln or incinerator.

Unstable operation will result in low stripping efficiency.

Steam Collapse

Once the column is started up, it is full of steam. If enough cold condensate enters the column, it will condense the steam where it enters the column, causing an almost instantaneous high vacuum in the top of the column.

The hot condensate in the bottom of the column will boil rapidly, almost exploding. This sudden upward rush will buckle trays, pop them out of their hold down clips, or carry packing out of the column.

Such a situation can happen on a start-up, when there is insufficient hot condensate in the bottom of the column to preheat the foul condensate. Great care must be taken to avoid such situations. Start-ups should be done very slowly.

Loss of trays results in loss of stripping efficiency.

Control of Contaminant Removal

Condensing in the reflux condenser must be controlled to maximise contaminant removal while minimising steam loss. At this point in the process, the system is a very complex, two-phase, multi-component system, mainly made up of water, methanol, reduced sulfur gases and turpentine, but with several other compounds such as ethanol and acetone in lesser quantities.

Good control depends on a combination of pressure and temperature control in the reflux condenser. See **REFLUX CONTROL** below.

Fiber

The foul condensates tend to contain pulp fibers and these fibers can plug packed columns and heat exchangers, especially plate-type heat exchangers. Fortunately, valve tray columns are self-cleaning.

The handling of fiber can be done in two ways. The first is to install a good fiber filter before the heat exchanger. The second is to design the system to pass the fiber through, using valve trays and heat exchangers with wide gaps or large diameter tubes.

This problem normally shows up as loss of flow through the heat exchangers, especially plate type heat exchangers.

Plugging of Heat Exchangers

Several mills have experienced a scale build-up on the stripped condensate side of the heat exchanger. The reason for this is still not clear, and the scale is very difficult to remove. For this reason, the stripped condensate should go through the tube side of the heat exchanger, to facilitate mechanical cleaning.

Attempts to chemically remove the scale have been tried, but with mixed success. Chemical cleaning should be done before the tubes become too badly scaled.

This problem usually shows up as loss of flow of the stripper bottoms, or high level in the bottom of the stripper.

Turpentine in Storage Tank

As there is turpentine in the condensates, there is a tendency for the turpentine to decant in the storage tank, and collect on top of the condensates. If this happens, and the storage tank is pulled down, it is possible to send a slug of turpentine to the stripper.

The turpentine will easily strip, and the turpentine will go through the SOG system, and cause problems at the incineration point, usually a high temperature trip.

There are several means to avoid the turpentine build up in the storage tank. The first is to send the turpentine decanter underflow, which can be rich in turpentine, directly to the suction of the stripper feed pump. The second method is to make sure there is agitation in the tank, to prevent the turpentine decanting. This can be done by making the incoming condensate lines tangential, recirculating feed condensate back to the tank, or by adding an internal agitator.

Skimming the turpentine can also be done. This can either be done periodically, returning the turpentine to the turpentine recovery system, or by continuously skimming the turpentine into the stripper feed condensate.

REFLUX CONTROL

As mentioned above, good control of the reflux cycle is required to operate the stripper efficiently, both with respect to contaminant removal and heat recovery.

For any given operating pressure, and any desired stripper off gas (SOG) concentration, there is fixed equilibrium temperature. Operating at the proper equilibrium temperature and pressure is necessary in order to control the overhead composition.

Further, operating at the proper equilibrium conditions will ensure removal of the turpenes and red oils with the SOG, where they can be burned. If the reflux condensate is allowed to sub cool, the red oils will separate out, and build up in the reflux cycle until they give control problems.

The concentration of methanol in the SOG is a compromise between stripping efficiency and heat recovery. At higher methanol concentrations, heat recovery improves, but stripping efficiency declines. At lower methanol concentrations, stripping efficiency improves, but heat

recovery declines. The generally accepted optimum methanol concentration in the overheads is 50% by weight.

In a non-integrated stripper, it is possible to control both the pressure and the temperature. Two methods of doing this are shown in Figures 8A and 8B.

With an integrated stripper, the evaporator operation will fix the stripping temperature. Thus it is necessary to control the operating pressure based on this temperature.

Temperature control is done by controlling the cooling water flow to the reflux condenser. Pressure control is done by controlling the back pressure on the SOG line.

In the scheme shown in Figure 8A, the temperature of the reflux condensate is measured rather than the gas, which gives faster response to temperature changes. This assumes that the condensate is not sub-cooled, and is at the equilibrium temperature. The condenser is mounted horizontally in order to minimize sub-cooling of the condensate. This scheme works well at design conditions, but sub-cooling tends to occur at reduced operating rates.

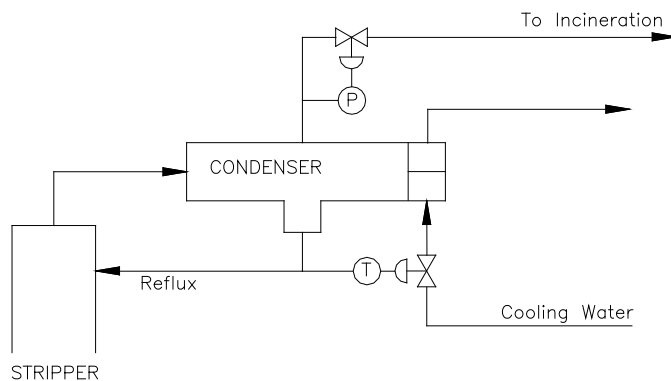


Figure 8A. *Reflux Control*

In the scheme shown in figure 8B, gas temperature is measured directly. Vapor from the reflux condenser is bubbled through the condensate in the reflux tank, which ensures that liquid and gas phases are in equilibrium. This gives better control, but at a capital cost penalty.

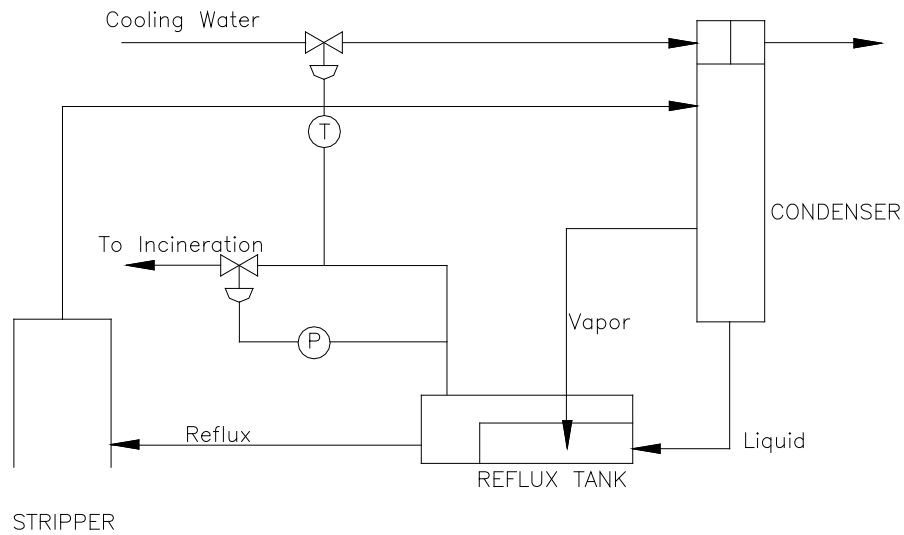


Figure 8B. *Reflux Control*

TRANSPORT AND DISPOSAL OF CONTAMINANTS

Once the contaminants have been stripped out of the condensate and removed from the system, they must be disposed of. The most common way to do this is to carry them in gaseous form and burn them directly in a kiln, boiler or separate incinerator. The system for transport is almost identical to a concentrated or, low volume, high concentration (LVHC) non-condensable gas (NCG) system. See Figure 9.

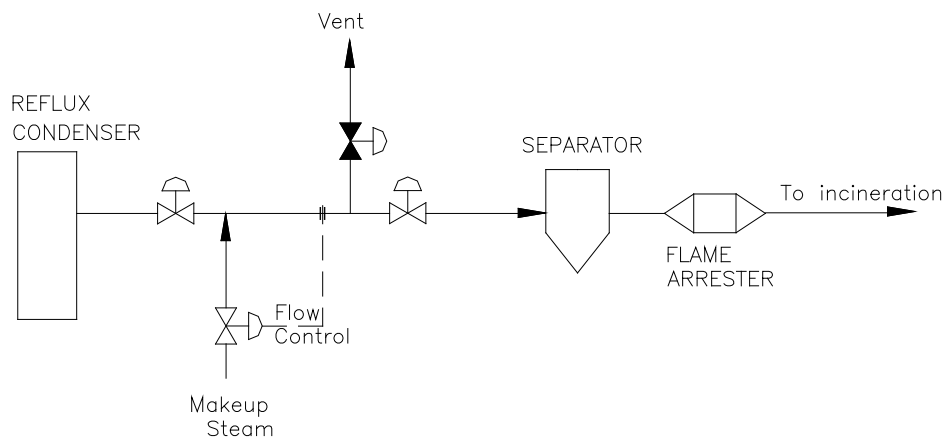


Figure 9. *Stripper Off Gas System*

The major difference with a concentrated NCG system is that the stripper gas system requires makeup steam to ensure purging of the entire system on start-up and to ensure a minimum velocity in the entire line at all times. This is done by measuring the line velocity near the incineration point and adding makeup steam, as necessary, near the pressure control valve at the stripper. The stripper gas system should be operating, with SOG going to incineration, before any foul condensate is sent to the stripper.

The advantages of burning stripper overheads are low cost, simple operation, and recovery of the high fuel value of the contaminants.

The major disadvantage is that if this stream is ever vented, it will create a severe odor problem in the local area very quickly.

It is also possible to collect the contaminants as a liquid by use of an extra condenser. This allows for storage of the contaminants as a liquid, which can be used for fuel or be further refined for recovery of the chemicals.

In the past, the liquid MeOH so collected fell under Resource Conservation and Recovery Act (RCRA) rules classifying it as a hazardous waste, making this form of collection virtually illegal. The Cluster Rule reclassifies this liquid methanol as a “clean fuel”, and thus allow it to be collected and stored as a liquid.

In most cases, the SOG will be further rectified to increase the methanol concentration to 85 to 90% by weight.

PROPER OPERATING PROCEDURES

The following procedures have been recently developed to reduce the possibility of steam collapse, and to eliminate venting of SOG during start-up and shutdown.

The stripper should be started up with steam. Once the stripper is fully heated and all air purged out of the system by cracking open the back pressure control valve, feeding of condensate can begin. At a minimum steam flow of about 20% of design, start with a condensate flow of about 10% of design.

After five minutes, increase condensate flow to 20% of design. Continue to increase both condensate and steam flow in 10% increments every five minutes until the full operating rate is achieved.

The slow start-up will prevent damage to the trays due to steam collapse, as well as give the incineration point time to adjust to the changing fuel load from the methanol in the SOG.

As mentioned before, any changes in operating rate should be made in small increments (maximum 10%) and as seldom as possible.

For planned shutdowns, condensate and steam flow should be reduced slowly, until 20% of design is reached. At this point stop the condensate flow, but leave the steam flow on.

The SOG should continue to be sent to incineration until the back-pressure control valve goes closed. At this point steam flow to the stripper can be stopped, and the SOG system (which now contains no pollutants), can be shut down.

In the emergency shutdown situation, such as when the fire goes out at the kiln or incinerator, it is necessary to vent the gases coming off the column.

To virtually eliminate this venting, condensate flow to the column should be stopped immediately and at the same time the pressure control valve at the column should be closed. However, steam flow should remain unchanged.

This will keep the system in thermal balance, which is especially important if integrated into the evaporator. It will also speed up the restart of the system. As before, on the restart, the condensate flow must be brought on slowly.

CONCLUSION

The Cluster Rule requires collection and treatment of foul condensates to reduce Kraft Pulp Mill pollution. The stripping of foul condensates is a good in-plant method of doing this. BOD, toxicity and odor can be reduced by stripping. In most cases, the pollutants can be converted to a usable fuel, giving an economic return on investment.

ACKNOWLEDGEMENT

This paper was originally authored by Mr. Tom Burgess. He had been the instructor for the TAPPI Kraft Recovery Short course since 1986, before retiring in 2005.

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2.6.4 COLLECTING AND BURNING NONCONDENSIBLE GASES

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INTRODUCTION

In order to meet stringent air quality requirements in kraft pulp mills, it is necessary to collect NCG, NonCondensable Gases (NCG), containing VOC, Volatile Organic Compounds, and TRS, Total Reduced Sulphur, from various emission sources into one or several systems for disposal by chemical modification or incineration.

The VOCs typically in the highest concentrations include methanol and the terpenes, while the TRS compounds most commonly encountered include hydrogen sulfide (H_2S), methyl mercaptan (CH_3SH), dimethyl sulfide (CH_3SCH_3) and dimethyl disulfide (CH_3SSCH_3). These latter compounds are noxious and have very low thresholds of odor detectability.

TRS are generated in pulp digesters when wood is cooked with Kraft liquor. TRS can also be generated in direct contact evaporators, turpentine systems, strippers, brown stock washers, liquor storage tanks, in recovery boilers, and in lime kilns. TRS gases that are emitted are contained in gases referred to as Noncondensable Gases.

NCG collection and treatment systems successfully contain and chemically convert the malodorous gases released during the operation of a kraft pulp mill. A well designed system will collect all odorous streams from the mill and deliver the gases to the point of treatment under controlled temperature and pressure conditions.

Attempts to collect and burn NCG were first tried in the late 1950's. The initial systems collected the gases in pipelines, using fans as motivators to transport the gases. These systems usually diluted the gases with air to bring the TRS concentrations below their lower explosive limits. This was not always successful, especially with concentrated gases coming from digesters and evaporators, and many early systems experienced fires or explosions.

In the early 1970's, a system was developed in Sweden that kept the NCG undiluted and that used steam ejectors as motivators. This system has virtually eliminated explosions in NCG systems, and is now the accepted method for handling NCG.

NCG also contains other pollutants, such as turpentine and methanol, which are classified as Hazardous Air Pollutants (HAPs). Environmental Regulations also require the collection and destruction of HAPs.

The steam ejector based NCG systems have been so successful and reliable, that the Environmental Regulations in the United States require all Kraft mills to have NCG systems, and to operate them at 99% uptime.

COMPOSITION OF NCG

NCG can be broken down into four separate categories, depending upon composition. These are: Concentrated NCG (low volume, high concentration (LVHC)); Dilute NCG (high volume low concentration, (HVLC)); chip bin gases; and stripper off gases (SOG).

Table I shows a typical analysis for concentrated NCG. These gases come from blow heat recovery systems, turpentine recovery systems, continuous digester flash steam condensers, evaporator vacuum systems, and hotwells. The actual composition will vary widely from system to system, and from time to time within the same system.

TABLE I Concentrated NCG Typical Analysis
Normal Operation

Source	TRS (vol%)			O ₂ (vol%)		
	Min	Avg	Max	Min	Avg	Max
Batch Digester	20	50	70	0.5	2.0	5.0
Continuous Digester	12	60	80	1.0	3.0	10.0
Turpentine (Batch)	0.1	2.0	10	4.0	12.0	20.0
Evaporator	1.0	60	70	1.0	5.0	8.0
Combined	10	50	65	2.0	3.0	6.0

Note: The balance of the gas is nitrogen

It should be noted that in concentrated NCG, the TRS gases and HAPs make up only about 50% of the volume. The largest component of NCG is air which has been depleted of 50% or more of its oxygen.

Although some air enters the system by dissolving in white and black liquors and then stripping back out of them, most air enters through leaks in the system. The oxygen is depleted by reacting with reducing agents, such as Na₂S, in the liquors that it contacts. The data shown in Table I are for a “tight” system.

PROPERTIES OF NCG

Corrosivity

NCG, especially concentrated NCG, are highly corrosive to carbon steel. These gases are normally saturated with water vapor, such that condensation occurs in the collection system. Some of the TRS gases, especially H₂S and CH₃SH are acidic, and will absorb in the condensate formed. The combination of this acidic condensate, along with the oxygen present in the NCG, can be very corrosive to carbon steel. Therefore, carbon steel should be avoided in NCG collection systems.

Other components of NCG, especially turpentine and methanol, are very strong solvents and can dissolve or soften plastics or resin in Fiberglass Reinforced Plastic (FRP) piping. Therefore, plastics or FRP should also be avoided in NCG collection systems. Further, FRP should be avoided because of potential failure during a fire started by NCG ignition.

Stainless steel of the 300 series, typically 304 or 316, has proven to be corrosion resistant to NCG and is the preferred material of construction for NCG systems.

Toxicity

NCG are highly toxic and are responsible for deaths and injuries in the pulp and paper industry. The toxicity of hydrogen sulfide is well known. At 20 ppm it causes irritation of the eyes and respiratory tract. Thirty minutes of exposure at 500 ppm causes severe sickness. Exposure at 1000 ppm for 30 minutes is fatal. The other components of NCG are similarly toxic. TRS concentration in concentrated NCG is 100,000 ppm or greater.

Because these gases are toxic, great care must be taken in the design and construction of systems to handle these gases. Gas leaks, especially in enclosed areas, must be avoided. Adequate venting must also be provided, such that when there is an upset condition causing the gases to vent, the gases are released in a safe area and in a safe manner. Vent stacks should be as high as possible and clear of any buildings, platforms and ladders where personnel may travel, and well clear of any ventilation air intakes.

Explosivity

TRS, methanol and turpentine are flammable in the presence of sufficient oxygen, and if contained in a pipeline or vessel, can be explosive. Table II shows the combustion properties of the main components of NCG.

TABLE II Combustion Properties of NCG in Air

	Explosive Limits		Flame Speed ft/s (m/s)	Auto Ignition Temp °F (°C)
	Lower (Vol %)	Upper (Vol %)		
H ₂ S	4.3	45.0		500 (260)
CH ₃ SH	3.9	21.8	1.8 (0.55)	
CH ₃ SCH ₃	2.2	19.7		400 (206)
CH ₃ SSCH ₃	1.1	8.0		572 (300)
Alpha-pinene	0.8	6.0	500 (154)?	487 (253)
Methanol	6.7	36.5	1.5 (0.50)	867 (464)

The Lower Explosive Limit (LEL) is the lowest concentration of gas, by percent volume, that, when mixed with air, will burn. Below the LEL, there is an insufficient supply of combustible material to sustain combustion.

Similarly, the Upper Explosive Limit (UEL) is the highest concentration of gas, that, when mixed with air, will burn. At higher concentrations there is insufficient oxygen to sustain combustion.

Unfortunately, the terms LEL and UEL only explain the situation in theoretical terms. In reality, TRS gases, as emitted, are mixed with air that has most of its oxygen depleted, such that the gases are not explosive. Figure 1 illustrates this situation.

It is based on some test data and the assumption that mixed TRS gases, which also contain other combustibles such as MeOH and turpentine, are flammable over the range of 2% to 50% for all combustibles. The exact shape of this curve has not been determined, and will vary depending on the TRS components present.

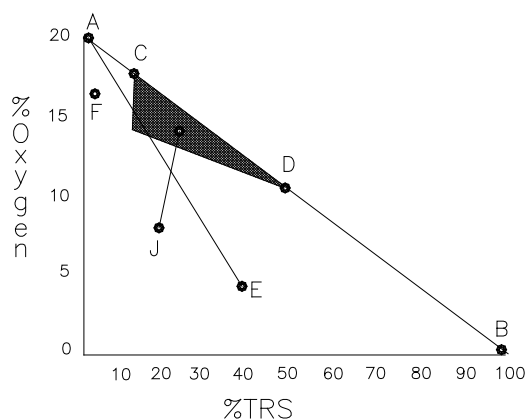


Figure 1. *Explosive Range of NCG*

Point A on Figure 1 is the pure air case with 21% oxygen, 0% combustible. Point B is the pure combustible case with 100% combustible and 0% oxygen. The two points, C and D, on Line AB represent the LEL and UEL respectively.

The shaded area under CD represents the range of mixtures that are explosive. Point E represents a typical concentrated NCG mixture, which is well outside the explosive range. However, as air is added, the mixture moves along Line AE towards Point A. At some point, if enough air is added, the gases will enter the explosive range.

In the past, attempts were made to reduce concentrated NCG below the explosive limit by diluting by a factor of 20 to 1 with air. As a result the mixture ended up at Point F, which is outside the explosive range.

However, if the system was under-designed, or if an upset occurred which reduced this dilution ratio, the gases would end up in the explosive range. At this point, if an ignition source were present, such as a spark from a fan, then an explosion would occur.

The explosive range for the mixture of combustibles that make up NCG has been estimated to be 2% to 50%, but this will vary greatly from mill to mill, depending on the composition of the waste gases.

Another characteristic of flammable gases is flame propagation speed. This is a measure of how fast the flame will travel through the pipeline or vessel once a fire is started. See Table II.

The flame propagation speed for sulfur gases is relatively slow. However, the flame propagation speed for turpentine is extremely fast. The value reported in Table II for turpentine is currently being disputed by some experts in the field, who claim the speed is much slower.

Regardless, explosions caused by TRS are usually minor, with minimum damage, while explosions caused by turpentine can be catastrophic.

It should be noted that the flame propagation speeds listed in Table II are based on pure compounds in pure air.

It should also be noted that while NCG systems are designed to handle the flame propagation speed of TRS, it is not practical to design against the flame propagation speed of turpentine. For this reason, it is very important to minimize the amount of turpentine entering the NCG system.

The auto ignition temperatures for the combustibles are as low as 206 °C, so care must be taken to ensure the NCG does not contact any surfaces above this temperature.

IGNITION SOURCES

Before an explosion can occur, three things must be present. The first two, namely a combustible material (TRS), and sufficient oxygen, have already been discussed. The final item is an ignition source.

Fans have traditionally provided an ignition source for NCG system fires. This could be from static sparks, hot spots on the casing if rubbed by the impeller, by a hot impeller shaft due to a bearing failure, or by sparks created by foreign material hitting the impeller.

Another potential cause of fire is welding. Welding torches have ignited many NCG fires. Lines and vessels containing NCG should be clearly labeled so that they are not accidentally touched by welding flames or arcs.

If welding is necessary, all lines or vessels thought to contain NCG should be thoroughly purged and then checked for combustibles before welding is permitted. Care must also be taken to ensure that welding sparks are not drawn in through vacuum relief devices.

As TRS is known to adsorb onto, and then desorb from metal walls, welding should be done immediately after purging and testing for combustibles.

Static electricity can also provide a spark to ignite NCG. All lines and vessels containing NCG must be properly grounded. Otherwise, a static charge may build up, eventually leading to a spark or static discharge. Several explosions have been traced to this mechanism.

Turpentine can provide a second mechanism for static discharge. If sufficient turpentine vapor enters an NCG system, such as during loss of water flow to a turpentine condenser, then some of that turpentine will condense in the piping along with the water vapor also present.

As water and turpentine are immiscible, they will decant in the pipeline. At this point, if the interface between the two immiscible liquids is subjected to a shear force, the friction between the two liquids can generate a static spark, which will ignite the turpentine.

Such a shear force can occur if these liquids enter a fan, or if they cascade from a horizontal pipe run down a vertical pipe run. As before, several explosions have been credited to this mechanism.

CONCENTRATED NCG (LOW VOLUME, HIGH CONCENTRATION) SYSTEMS

Concentrated NCG come from two types of sources, continuous and batch. Continuous sources are:

- Continuous digesters
- Flash steam condensers
- Turpentine recovery systems

- Evaporator seal tank and vacuum systems
- Pressurized heavy liquor storage tanks
- Foul condensate storage tanks
- Stripping systems when operated in TRS mode only

Batch sources are:

- Digester relief condensing systems
- Blow heat recovery systems

The volumes of gases to be collected vary greatly from mill to mill, and from time to time within each mill. Table III shows the expected ranges of concentrated NCG flow from various sources. It is always best to design on actual test data. If this is not available, then conservative values should be used. Line sizes should be chosen to give a low pressure drop (0.4 to 1.0 inches w.c. per 100 ft. of pipe) at peak flow conditions.

TABLE III Concentrated NCG Volumes

Source	Ft ³ /Ton Pulp	m ³ /Tonne Pulp
Batch Digester	100 - 200	2.6 - 5.2
Continuous Digester	150 - 300	3.9 - 7.7
Turpentine System (Batch)	40 - 80	1.0 - 2.0
Evaporators	50 - 200	1.3 - 5.2

Note: Volumes are actual at 60 °C (140 °F) and saturated with H₂O.

Collected NCG are typically at 60 °C (140 °F) or cooler. If the gases are hotter than this, then consideration should be given to cooling them before transporting them.

The concentrated NCG, Point E on Figure 1, are normally outside the explosive range due to lack of oxygen. Therefore, the system should be designed to prevent ingress of air into the system. This is done by sealing all parts of the system to make them airtight.

However, a sealed system can be exposed to high pressure or vacuum under upset conditions. Most storage tanks or evaporator hotwells are not designed to withstand pressure or vacuum. Therefore they must be protected by both pressure and vacuum relieving devices in order to prevent damage to the vessel during upset conditions.

Keeping the NCG outside the explosive range will insure a safe system at most times. However, during upset conditions, especially during start-ups and shutdowns, it is possible for air to enter the system, creating a potentially explosive mixture.

For this reason, it is necessary to eliminate all possible ignition sources during the design and operation of the system. Hence, steam ejectors, rather than fans, should be used to motivate the gases. The CNCG system collects the most concentrated emitters into a relatively small system. The design of the CNCG header incorporates a number of safety design features including use of steam ejectors, flame arresters, rupture disks, high gas velocities, sloping of lines in the direction of flow, and low point drains.

Since the vacuum that an ejector pulls varies inversely with gas flow, it is possible to pull a high vacuum under low flow conditions. In that case, a vacuum breaker could open, allowing air into the system. To overcome this problem, a pressure controller is used on the ejector suction.

Steam ejectors have other advantages. It is possible to size the ejector and the piping downstream of the ejector, such that the steam flow to the ejector insures a line velocity greater than the flame propagation speed of TRS gases, even under low NCG flow conditions.

It should be noted again that it is impractical to design against the flame propagation speed of turpentine. However, steam is an inert gas, and it is possible to design the ejector such that the steam flow will dilute the NCG to a point outside the explosive range.

Refer to Figure 1. If, for some reason the NCG coming to the ejector is in the explosive range, Point I, the steam flow will dilute the resulting steam - NCG mixture to Point J, which is outside the explosive range.

For the reasons discussed above, it is necessary to keep the steam in the system from condensing. This requires that all steam lines, and all NCG lines after the ejector be properly insulated to insure the steam does not condense.

If a condenser or scrubber is used after the ejector, and the steam does condense, then some other means of ensuring line velocity and dilution should be provided. If not, then the NCG will burn back into the transport line whenever the gases get into the explosive range.

Even if care is taken to keep the gases outside the explosive range, and care is taken to remove ignition sources, there is still a remote possibility that a fire may occur.

To protect against this, flame arresters should be used at several points throughout the system in order to prevent the spread of a fire and to minimize pipeline and equipment damage. As a general rule, flame arresters should be placed at each NCG source, and at each incineration point.

To prevent damage from an explosion in the NCG line, line size rupture discs have been used on the NCG line. Typically, rupture discs have been placed at approximately 100 ft. (30 m) intervals on extended NCG lines. Care must be used when placing the rupture discs to insure that the gases are vented in a safe location if a disc ruptures.

However, because the rupture discs are subject to mechanical failure due to fatigue rather than explosions, some mills are removing them, and some new installations are built without rupture discs on the transport lines.

The collection of NCG from continuous sources is relatively straightforward. See Figure 2. Care must be taken to insure that the vessels from which the gases are being collected are airtight and properly sealed. Each vessel should be protected by a pressure/vacuum breaker and by a flame arrester.

Special attention must be given to overflow lines on these vessels. Overflow lines must be sealed, to insure that during an upset condition, pressure or vacuum is relieved through the pressure/vacuum breaker, rather than through the overflow line.

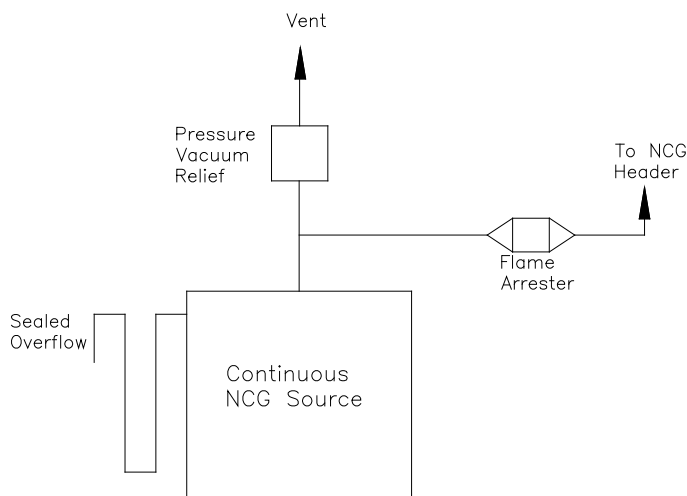


Figure 2. *NCG Collection - Continuous Source*

The collection of NCG from batch digesters is much more difficult, due to the cyclical nature of the gas flow. Between blows, there is virtually no NCG flow. At the start of a blow, there is a momentary high flow. As the blow progresses, this flow drops off until it reaches zero flow at the end of the blow. In many cases, there is another small peak flow at the end of the blow when the digester blows clean.

In the past, these variations in flow were smoothed out using a gas holder of either a diaphragm or inverted-bell type. These gas holders were expensive to build and troublesome to maintain. In modern NCG systems, gas holders are no longer required, except in rare situations.

Blow steam condensing systems should be operated under positive pressure at all times, to prevent ingress of air into the system. If air gets into the accumulator and condensers, it becomes part of the NCG to be collected. The result is a very high NCG flow, and a decrease in

condenser efficiency at the start of the blow, invariably leading to venting of steam and NCG. See Figure 3.

Pressure is maintained in the system by use of pressure control on the NCG discharge line from the condensers, or by pressure control between the primary and secondary condensers. Padding steam is also added to the accumulator, blow tank or condensers between blows to keep the pressure and temperature up.

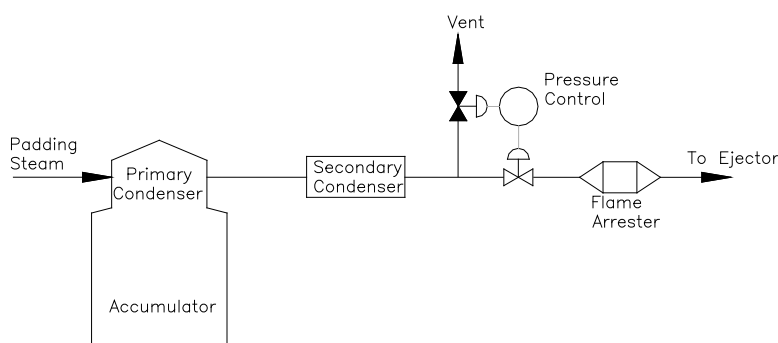


Figure 3. *NCG Collection - Batch Gases*

It is essential that the blow steam condensing system be properly sized and operated. If this is not done, very little NCG from the batch digesters will be collected.

Pressure relief is also required on the NCG line. This will prevent large quantities of steam from entering the NCG system and overloading the ejector system if the blow steam condensing system fails.

Batch gases must be kept separate from continuous NCG sources until at least downstream of the ejectors. If this is not done, the pressure swings in the batch gas line will upset the smooth flow of the continuous gases, causing frequent venting of the continuous gases.

Figure 4 shows a concentrated NCG transport system.

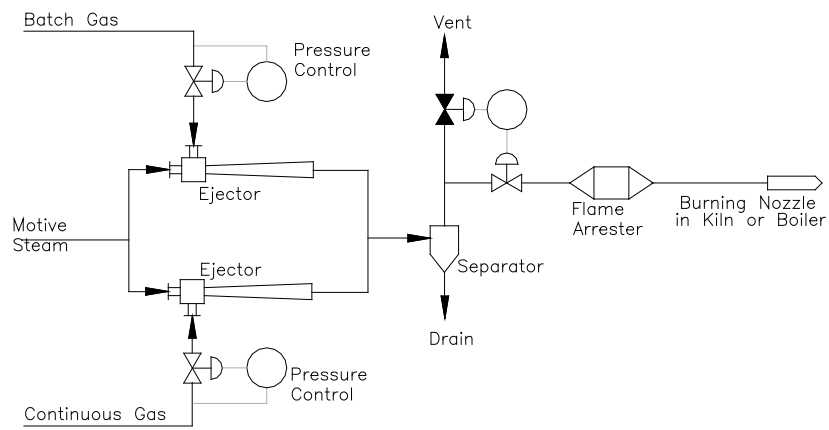


Figure 4. *LVHC NCG Transport*

PIPING DESIGN AND LAYOUT

The flame speed of the combustibles is around 0.5 m/s; all NCG transport piping is designed for velocities greater than 5 m/s to ensure the flame at the incineration point cannot travel back through the piping. NCG piping design takes into account the volume of gases at each point of the system to ensure gas velocities will be above the flame propagation speed. Proper selection of gas velocities combined with mechanical safety devices and an extensive interlock system provide for safe operation of these systems.

Consideration in the design of NCG systems must be given to the condensates that form in the lines. The NCG are normally saturated with water vapor, and some water will condense in the lines. Therefore, NCG piping is designed with a slope in the direction of flow to facilitate the removal of condensate, and all low points have sealed drains to prevent plugging, foul condensate accumulation and two phase bidirectional flow.

Sloping of NCG lines in the direction of flow is important to prevent condensate hang up, "black tar" accumulation, and pressure fluctuations in the system. Black tar is commonly found in CNCG lines. It is the result of the condensation and accumulation of pitches, resins, terpenes, and other organic material carried with the NCG.

Low point drains and separators are also required. The condensates formed are very foul and should be collected and treated.

Special attention needs to be paid to the layout and positioning of block valves at the incineration point on both the NCG lines and the steam purge lines.

Any condensate that collects behind these valves will be injected into the incineration device when the valve is opened. This condensate will be instantly vaporized when it gets into the incineration device, and could result in a minor explosion, damaging the equipment.

Attention must also be paid to the condensate collection system. It must be designed so that it can never be pressurized. If it does become pressurized, condensate can be pumped back up the collection lines, and into the incineration device, possibly leading to an explosion.

The motivating steam ejectors should be as close to the point of incineration as possible. A separator or mist eliminator is used after the ejector to remove any water droplets or condensate before the gases are injected into the incineration point.

The NCG then goes through a final flame arrester and into the kiln, boiler or incinerator for destruction. If the kiln, boiler or incinerator is not available for incineration, the gases should be safely vented upstream of the incineration point.

The final consideration is vent lines. From time to time it will be necessary to vent the gases to atmosphere. All vent lines should be made as high as possible, and should release the gases straight up and away from buildings and platforms.

Vent lines should not terminate in elbows, or have caps on the discharge. Water build-up in the vent line, due to rainfall or condensate, can be removed through sealed drain lines.

SCRUBBING NCG

Some mills have found it advantageous to scrub the TRS from the NCG before the gases are burned. Scrubbing the TRS may be necessary to reduce SO_2 emissions if the NCG are burned in a boiler, or an incinerator. In mills where NCG are burned in lime kilns, the SO_2 formed may contribute to ring formation in the kiln. Scrubbing the NCG may alleviate this problem.

Scrubbing is normally done in packed columns, or spray columns, using either a caustic solution, or white liquor for scrubbing. The polar sulfur gases, H_2S and CH_3SH , are easily and almost totally removed from the NCG by a chemical reaction, while the non-polar gases, CH_3SCH_3 and CH_3SSCH_3 are not removed.

In most cases, hot white liquor is used for scrubbing. However, the NCG is heated up, and must be sent to a gas cooler to reduce the gas volume and reduce condensation in the line following the scrubber.

To avoid the cooler, some mills have used cooled white liquor for scrubbing. However, the heat exchanger used to cool the white liquor is often subject to severe scaling.

There are also specialized chemicals on the market that are designed to scavenge the TRS from NCG. These have proven very effective. However, their high cost excludes their use on a

continuous basis, but they have been successfully used as a backup for TRS reduction when incineration is not available.

Depending on the scrubbing media used, and the NCG streams scrubbed, overall TRS removal efficiencies from as low as 40% to as high as 99% have been reported, although overall TRS removal of about 65% is typical.

DILUTE NCG (HIGH VOLUME, LOW CONCENTRATION) SYSTEMS

DNCG sources generally include:

- Diffusion washer
- Brown stock washer
- Knotter hood
- Screen hood
- Decker washer
- Black liquor oxidizer
- Soap skimmer
- Sewer vent
- Fiberline tankage: filtrate, stock, knot, screen feed, screen rejects, refiner feed
- Evaporation tankage: black liquor, spill, soap, tall oil
- Recovery tankage: precipitator mix and chemical ash mix tanks

Historically, sources from the lime kiln and causticizing areas were not collected due to low TRS concentrations, although it is becoming more common to include these in the DNCG system as there is typically some hydrogen sulphide present.

The recovery boiler smelt dissolving tank vent gases are a special source due to the high particulate loadings and they are usually dealt with in a separate system. New or rebuilt recovery boilers usually include a venturi and cyclone, followed by a cooler, scrubber, fan and heater to treat these gases so they can be injected into the recovery boiler as combustion air.

As with concentrated NCG, the total volume of dilute NCG to be handled varies greatly. Flows of between 10,000 to 30,000 ft³ per ton (300 to 900 m³ /tonne) of pulp are typical.

Brown stock washer hoods on vacuum drum washers are the largest single source of dilute NCG. It is necessary to properly seal the washer hoods to the vats, and to train operators to always keep the inspection doors closed. Suppliers are developing better fitting hoods to keep the washer hood gases to a minimum. Design volumes are now about 1000 ft³/min (cfm) per drum.

Recirculating seal tank vents back to the washer hoods in order to reduce volumes has been successfully done. This reduces the volume to be collected here by about half.

Similarly, air for air doctors should be taken from the hood, thus recirculating this air and further reducing the volume.

Pressure washers and diffusion washers have much lower volumes of gas to handle.

Large, flat top storage tanks are another problem. In most cases, they were never designed to take any pressure or vacuum. When they are tied into a collection system, they will be subjected to either a pressure or a vacuum (or both from time to time). Safe design here is a real challenge.

Because NCG collection systems are designed to provide up to about 10 kPa (1.5 psia) draft, it is possible to expose source tankage to vacuums that are considerably higher than design; as well, the age and condition of the tankage must be considered. Tanks that were previously free to vent to atmosphere will be very confined. Confined vapour spaces in tanks are susceptible to vapour collapse either slowly through radiation loss or quickly through liquid spray. Vapour collapse will result in tank implosion.

To safeguard against tank collapse, high volume mechanical vacuum breakers are required at each tank not designed for full vacuum. Water sealed vacuum breakers have been used in these applications but are slower to respond due to their higher inertia. In addition, pressure relief devices are used to protect tankage from over pressurization where there is a possibility of freezing, thus plugging the overflow.

Gases in this category are normally outside the explosive range due to low concentration of TRS (see point F on Fig. 1). Dilute NCG systems are normally designed to run at 25% of the LEL or less. Frequently they are diluted with air to ensure they are well below the LEL. Dilute NCG are normally collected and transported in a common pipeline, and motivated with a fan.

As with concentrated NCG, these gases are normally collected at 60 °C (140 °F) or cooler. In instances where dilute NCG are hot and saturated with moisture, coolers are used to reduce their volume and moisture before the fan.

Heaters can be used to raise the dilute NCG above the saturation temperature, and thus dry them out. Generally, they are heated to a point where they are at 50% relative humidity. Having a dry gas is necessary if the gas will eventually go through mild steel equipment, such as forced draft fans, at the incineration point.

Under no circumstances should concentrated NCG be added to dilute NCG, especially a concentrated NCG stream that may contain considerable quantities of turpentine vapor.

As in the case of concentrated NCG systems, the pipelines for dilute NCG systems should be properly sloped and drained. A moisture separator should be installed upstream of the fan suction.

Many mills monitor the concentration of combustibles in the dilute NCG system to ensure that the gases are always well below the LEL. If the combustible concentration rises above 50% of the LEL during system upsets, then the system is vented or shut down until the problem is corrected.

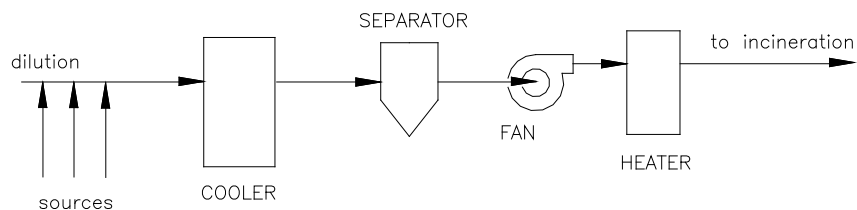


Figure 5. HVLC System

A system for handling dilute NCG is depicted in Figure 5.

CHIP BIN GAS SYSTEMS

Chip bin NCG (and more recently ImpBin™ NCG) has historically been defined as a high volume low concentration source and collected with other DNCG sources. The majority of the time, these gases are emitted as a dilute DNCG source, but it can at times, reach combustibles levels (turpentine and TRS) that are in the explosive range. Therefore, it is common to install a dedicated system for this source.

In the United States, the Environmental Regulations require the NCG from chip bins to be collected if flash steam is used for steaming. Collection is not required if fresh steam is used for steaming.

Collection of chip bin NCG is illustrated in Figure 6. Chips are steamed in the chip bin to remove air before they enter the steaming vessel. This steam can drive volatile compounds, such as turpenes, out of the chips. This situation is aggravated when too much steam is used, or if the steam breaks through the chips due to low chip bin level.

In any case, large quantities of turpentine vapor may be present in chip bin gas. One kraft mill in the southern United States that pulps pine, has reported recovering up to 2 lb. of turpentine per ton (1kg/tonne) of pulp from chip bin gas.

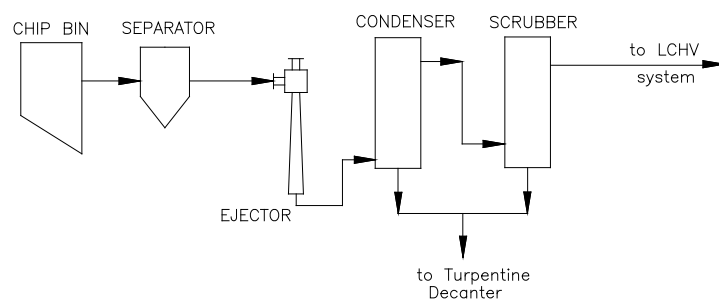


Figure 6. *Chip Bin System*

The majority of time, the chip bin gases are emitted as a dilute NCG source, i.e. essentially air with trace amounts of combustibles (TRS, methanol, etc); the combustibles originate from the low pressure feeder which is typically vented back to the chip bin. However, chip bin vents may also have momentary situations with high combustible concentrations; this may occur due to pockets of foul gas forming in the bin which then break through the chip pile causing a surge in flow and combustibles in the vent gases. This situation is exacerbated when contaminated steam from the extraction liquor flash tanks (a CNCG source) is used for pre-steaming.

In the case of softwood mills, the vent stream may also contain high turpentine vapour loadings due to stripping from the raw chips. In locations where a high proportion of softwood is used in the wood supply, turpentine liberated in the chip bin and carried out with the NCG becomes a safety concern if introduced into the main DNCG collection header.

The chip bin typically requires pressure and vacuum protection; fast acting mechanical type devices are the preferred technology. The vent gases from the chip bin are first passed through a cyclone separator to remove any entrained fibre; the cone of the cyclone is flushed with hot water or filtrate to prevent fibre hang-up. The gases are then passed through an indirect contact cooler, similar in design to the DNCG cooler, to cool and dehumidify the gases ahead of collection. For softwood mills, due to potentially high turpentine levels, a direct contact turpentine scrubber is included after the cooler.

Due to the transitory nature of chip bin NCG, and because the concentration of combustibles in these gases can exceed the lower explosive limit (LEL), the design philosophy for collecting this source has been changing and the chip bin is now typically collected in a separate dedicated NCG system designed very similar to a CNCG system as described above; a steam ejector, moisture separator, flame arrester and rupture disk are supplied for each incineration point.

The concerns over the transitory nature of chip bin NCG are exacerbated with the recent change in design where the chip bin and impregnation vessel have been combined into what is referred to as an ImpBin™. In this design, the hot extraction black liquor is flashed directly into the chips; therefore, the chance of introducing what is CNCG source (extraction liquor flash) into the ImpBin™ vent gases is exacerbated. For mills that include an ImpBin™, the installation of a dedicated ImpBin™ NCG system is mandatory.

Proper treatment of chip bin gas includes cooling, condensing and scrubbing to remove as much turpentine as possible. A typical system will include a separator to remove chip fines, followed by an indirect cooler/condenser, followed by a packed column scrubber using cold water.

At temperatures as low as 87 °F (30 °C) and atmospheric pressure, the vapor pressure of the alpha-pinene fraction of turpentine is high enough to create an explosive mixture. Therefore, dilution air must be added to ensure a safe mixture is being transported.

The use of cold water to recover turpentine can cause the temperature of the condensate going to the decanter to be too low for good decanting. In this case, it may be necessary to reheat the condensate before it goes to the decanter.

STRIPPER OFF (SOG) GAS SYSTEMS

Some mills treat their odorous foul condensate in a steam stripping column, producing SOG, Stripper Off Gas. This effectively reduces the odour originating from downstream effluent treatment systems where the odorous compounds may be stripped out during aeration. In

the case of a stripping system designed to target TRS removal only, the SOG consists mainly of noncondensables and is added to the CNCG collection system.

The steam stripping system can also be designed for removal of volatile condensables, mainly methanol, but also turpentine and other organics, to reduce the BOD, Biological Oxygen Demand, loading to the effluent treatment system. The stripping column will then include a rectification system and a reflux condenser and a higher steaming rate is used.

In most cases, the stripper off gas is removed as a mixture of 50% methanol and 50% water, by weight, at about 210 °F (99 °C). This stream, due to its high temperature and water content, must be transported separately in a concentrated NCG system (see Figure 7). For a BOD stripper, the SOG cannot be added to the CNCG system and is transported to the incineration point in a separate dedicated system.

The stripper typically runs under pressure, so an ejector is not required for motivation. It is necessary to insulate and heat trace the gas line in order to prevent condensation.

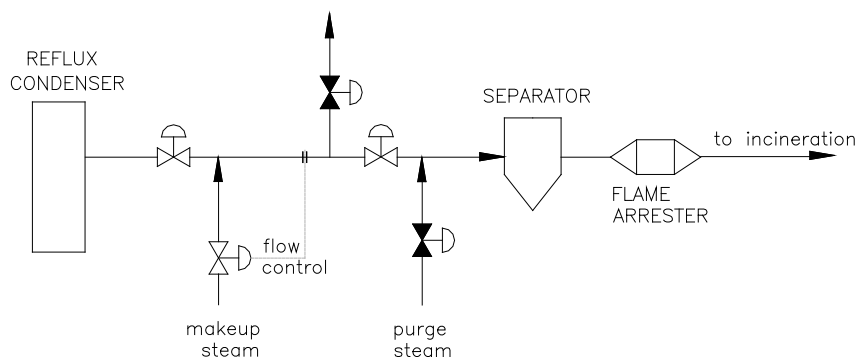


Figure 7. *Stripper Gas System*

It is necessary to steam purge the entire line to remove all air before any methanol rich gas is introduced. This is done by a steam makeup line.

Steam is added at the stripper pressure control valve on flow control by measuring the flow at the incineration end of the line. This ensures a minimum velocity in the line during start-up, shutdown and low flow conditions.

As an alternative to incineration of SOG as a gaseous stream, it can be passed through a condenser to produce a liquid methanol solution, referred to as SOL, Stripper Off Liquid. The SOL can then either be fired directly, or it can be passed through a second steam stripping column to rectify it up to 90+wt% methanol.

The SOL can then be used in the lime kiln displacing fossil fuel, or as a support fuel when firing CNCG in the recovery boiler or in a dedicated incinerator. With installation of a methanol purification system, it is possible to produce grade "AA" (99.85%) methanol from SOG for use in ClO_2 production or for external sale.

BURNING NCG

Several options are available to treat the NCG including incineration, scrubbing, biofiltration, gas phase oxidation, and adsorption. Unfortunately, other than incineration, no other alternatives have been shown to have high enough efficiency or reliability. That said, CNCG scrubbing is commonly used for pre-treatment ahead of incineration.

The most effective means of treating waste gases is by thermal oxidation. Minimum incineration temperatures in the combustion zone are required to achieve and maintain this oxidation, for kraft mills this temperature is typically 850°C. The incineration system thermally converts the organic vapours and TRS contained in the NCG to water vapour, carbon dioxide, and sulphur dioxide. Thermal oxidation of some of the substances requires turbulent mixing, which is accomplished through relatively high pressure drops, gas inlet design, and the overall geometry of the incineration device.

In order for the TRS and HAPs in NCG to be properly destroyed by combustion, three conditions must be met. These are:

- Temperature of 1600 °F (871 °C)
- Residence time of 0.75 sec.

- Excess oxygen (O_2) content of 3-4%

These should be considered basic conditions, and if any of them are exceeded, the others can be reduced. That is, if the temperature is higher than 1600 °F (871 °C), then reducing retention time, and/or lowering excess oxygen content, will still provide good destruction. For example, some new, high efficiency kilns are capable of meeting TRS emissions at excess oxygen content as low as 2%.

Options for thermal oxidation in a kraft mill include the recovery boiler, power boiler, lime kiln, or a waste gas incinerator. The most popular disposal point for CNCG, SOG and SOL is incineration in the lime kiln, with incineration in the recovery boiler, power boiler and dedicated incinerator also practiced. DNCG has traditionally been fired in the power boiler, although recently the recovery boiler appears to have become the number one choice. Dedicated incinerators have also been used for DNCG treatment but because these can be high in operating costs they are more often used as a secondary disposal point.

Lime Kiln

The lime kiln has traditionally been the first choice for burning NCG. It has the advantage that most of the SO_2 formed is absorbed on the lime mud, and returned to the liquor cycle.

In many mills, the lime kilns are overloaded, and have low levels of excess O_2 . An overloaded kiln is not very effective at TRS destruction.

Modern high efficiency kilns are designed to run at about 0.5% excess oxygen. This is normally not enough O_2 to completely destroy TRS.

CNCG can be fired in the lime kiln in a dedicated port on the main burner or preferably in a dedicated injection nozzle. This nozzle must be cooled in order to prevent preignition of the gases while still in the nozzle. In addition, during NCG system shut down, with the kiln operating, the cooling jacket prevents heat damage to the nozzle.

Air, steam and water have all been successfully used as cooling media. NCG can also be injected through a dedicated port in the main fuel burner of the kiln.

The dedicated firing gun has many advantages over burning in a main burner. Avoiding burning the CNCG in a main burner allows the main fuel to burn more efficiently, as the waste gases do not interfere by "grabbing" oxygen and starving the main flame. Also important is that any impurities contained in the waste gases do not have a chance to foul the main burner, resulting in fewer unscheduled shutdowns and less maintenance on the main burner.

SOL is burned in the lime kiln using a dedicated air or steam atomized injection nozzle. SOL is a heat contributor and will result in primary fuel savings.

When incinerating NCG in the kiln, a large fraction of the SO_2 formed in the burning of NCG is expected to be absorbed in the reburned lime as calcium sulphate. The remaining SO_2 passes to the primary dust chamber and scrubber where SO_2 can also be absorbed.

Kiln ringing, balling, and dusting can be minimized by maintaining high mud solids of at least 70% and minimizing soda content in the mud. Removal of TRS by upstream alkaline absorption, such as scrubbing with white liquor, is also helpful. Since SO_2 will acidify the lime mud at the back of the kiln, it is important to ensure that the lime mud is well washed and the sulphides removed; acidification will result in the formation of H_2S and this will be seen in the lime kiln vent flue gases.

The dilute NCG are not normally burned in the lime kiln due to their large volume. If burned in the kiln, they are used as primary air and introduced either on the suction or discharge of the F.D. fan. However, in most cases, the volume of dilute NCG is larger than the kiln primary air flow, and thus the kiln cannot be used.

Power Boilers

Firing of CNCG, DNCG, Chip Bin NCG, and SOG or SOL in the power boiler is being done at many mills around the world. In the case of DNCG, the flow added to the power boiler effectively displaces normal combustion air requirements. Similar to the recovery boiler and unlike kiln or incinerator burning however, the heat generated during firing is not wasted but is recovered as steam.

Boiler incineration of NCG will probably not reduce boiler natural gas requirements. Because of the additional moisture introduced with the NCG, fuel consumption may in fact increase.

However, those NCG sources with relatively high TRS/organic contents will be net heat contributors.

In order to avoid TRS leakage into the boiler area, the point of NCG entry must be carefully selected. CNCG, Chip Bin NCG, and SOG is typically fired through dedicated injection nozzles located near the fossil fuel burners while possible admission points for DNCG include the natural gas burners, under/over grate plenums, or directly through the boiler wall. Admission points may be a combination of one or more of these alternatives.

The NCG flow may be split into two or more parallel lines before entering the boiler. This is done to balance the flow of NCG into the boiler. However, if this is done, great care must be taken to ensure a balance of flow to each nozzle, in order to prevent flame backs due to low gas velocity on one side.

Ideally, the preferred entry point should be into an area of high temperature and turbulence in the boiler. Because of the huge volumes of boilers, residence times are not an issue and are expected to be far in excess of the standard 0.75 to one second residence time.

For DNCG, the use of natural gas burners as an admission point will typically require either adding one or more new units as required or replacing the existing natural gas burners. The new gas burners would be equal to the existing units except they would use DNCG for combustion air. Since this method would require continuous consumption of natural gas, operating costs may be high, unless the boiler is operated with one burner on natural gas at all times.

The use of grate plenums for DNCG entry is practiced by most mills. Since the existing air ducting are not air tight and are therefore susceptible to leakage, modifications must be made to ducting to seal all joints tight. DNCG leakage of even small quantities will produce an odour nuisance in the local area.

The small quantities of TRS gases are converted to sulphur dioxide as byproduct. The SO₂ is lost in the flue gas and cannot be economically recovered. Provided the prevailing flue gas temperatures are kept above the "acid condensation" temperature of about 150 °C, the additional sulphur dioxide loading in the flue gas is not expected to affect corrosion rates in

the boiler. However, in situations where flue gas scrubbing or heat recovery is practiced, the additional sulphur dioxide will cause a drop in the prevailing pH. In these specific cases, accelerated corrosion in the stack may be quite pronounced.

Studies have shown that considerable amounts of SO₂ are absorbed on the ash from coal or bark, if either of these fuels is used in the boiler. This can reduce the SO₂ emitted from the burning of NCG.

Recovery Boilers

The recovery boiler is theoretically the best place to destroy NCG, as the sulfur gases are destroyed and recovered as Na₂S in the smelt.

However, due to the nature of the recovery boiler, its importance in the recovery cycle, and the potential for catastrophic explosion due to water entering the unit during operation, or explosive gas build-up during shut down, the recovery boiler is generally not used as a place to burn NCG.

The Black Liquor Recovery Boiler Advisory Committee (BLRBAC) has recently published “Recommended Good Practice for the Thermal Oxidation of Waste Streams in a Black Liquor Recovery Boiler”. It is available on the BLRBAC Website, www.blrbac.com.

New mills are burning concentrated NCG in recovery boilers successfully. With respect to dilute NCG, several mills are now doing this, and there is a trend to accepting this practice.

DNCG consists of 99+% air and therefore simply displaces combustion air in a boiler. It is typically introduced to a recovery boiler at the secondary air level, displacing combustion air volume supplied by the secondary air fan. It can also be introduced at the tertiary air level but the available volume to be displaced is lower and the potential to recover the sulphur into the smelt bed is reduced, instead venting with the flue gases as sulphur dioxide.

For DNCG incineration in recovery or power boilers, the DNCG must not simply be added into an existing air manifold and mixed with combustion air; instead it must be introduced

directly into the combustion chamber; this typically takes the form of multiple nozzles at dedicated ports or penetrating the manifold.

Incinerators

A special dedicated incinerator is a proven and reliable method for disposal of NCG gases. Despite its potentially high capital and operating costs, the incinerator has become more popular, primarily because it is independent of other operating equipment. The incinerator has a virtually unlimited turndown capability, being able to burn various waste sources in any flow range from almost zero to one hundred percent.

The dedicated incinerator consists of a cylindrically shaped, horizontal or vertical vessel made of refractory lined steel. The design of the vessel is to provide efficient gas mixing with sufficient retention time.

The incinerator is typically sized to handle malodorous gases from the CNCG, Chip Bin NCG and DNCG collection systems, SOG or SOL from the foul condensate stripping system, and crude sulphate turpentine (CST) from the turpentine recovery system. CNCG and SOG are admitted through dedicated ports in the auxiliary burner, while SOL and turpentine are introduced through dedicated air or steam atomizing nozzles. To maximize efficiency, DNCG is used as primary combustion air for the auxiliary burner, with the balance introduced into the main chamber.

Waste Heat Boiler

If DNCG is burned alone, fossil fuel requirements can be extremely high and heat recovery can be practiced to minimize usage. If the incinerator is to be used for a backup incineration point then heat recovery equipment is typically not installed. The two main types of heat recovery are DNCG preheating using the flue gases or steam production via a waste heat boiler.

For DNCG preheating, the DNCG is passed through a heat exchanger in counterflow to the hot flue gases. Care must be taken to ensure the flue gases are not cooled to below the acid dew point or corrosion will become a concern. The acid dew point is typically about 150 °C, so the

flue gases are cooled to no less than 200 °C. For an incinerator operating at 850 °C, the DNCG can be preheated up to about 600°C.

For steam production the hot flue gases are directed to a firetube waste heat boiler. The boiler is typically a horizontal shell and tube heat exchanger with the hot flue gases passing through the tubes. Again it is imperative that the flue gases from the heat exchanger remain above the acid dew point temperature. Boiler feed water is fed to the shellside and the steam produced rises to a steam drum located above the boiler. Heat recovery is limited to about 80% with the recovered heat in the form of 400 kPa(g) (58 psig) steam.

Other mills have installed incinerators without heat recovery and use this system as a backup to the power or recovery boiler. Fossil fuel requirements are high when the system is in use, but with good availability of the main incineration point the occasional use of the incinerator with its high fuel consumption is not a concern.

Incinerator Flue Gas Conditioning

Once the flue gas leaves the incinerator or heat recovery equipment it is either vented or cooled and scrubbed for SO₂. Available methods of flue gas conditioning include air quenching, water quenching, alkaline absorption and water absorption.

Typically, the vent gases exiting the incinerator are scrubbed for removal of SO₂. Before scrubbing, the temperature of the vent gases is lowered by direct contact of water with the gases in a spray type cooler, or quench vessel. The vent gases first pass through a venturi throat section which promotes turbulence and mixing with water spray from several nozzles. The gases then flow to the main section of the quench vessel where they are sprayed with water once again. From here, the cooled gases flow to the absorption tower for scrubbing of the SO₂.

The absorption tower can be a spray type or a packed type tower using an alkaline solution for scrubbing of SO₂. Alkaline solutions successfully used include caustic soda, sodium carbonate, oxidized white liquor and oxidized green liquor. Scrubbing of SO₂ with caustic provides 99+ percent reaction efficiency of SO₂ producing sodium sulphite and bisulphite, Na₂SO₃ and NaHSO₃.

Another byproduct of incineration is sulfuric acid mist (SAM), which can give high opacity. Some mills have added devices, such as Candle Mist Eliminators, to reduce SAM.

Stripper off gas, due to its high MeOH content, has a very high fuel value, and can be used as a primary fuel for incinerators, reducing the incinerator operating cost, by supplying most of the fuel requirement.

However, SOG also contains ammonia, which can give high NO_x emissions from incinerators.

In some cases, if the incinerator combustion air flow is greater than the dilute NCG flow, it is possible to burn the dilute NCG in the incinerator, although this is not normally done.

Regenerative Thermal Oxidizers

Regenerative Thermal Oxidizers (RTO) are viable devices for incinerating dilute NCG, as they generally have much lower installation costs than other incineration alternatives. They also have very low operating costs, and very high uptime.

Several mills are successfully using RTOs to incinerate dilute NCG. The RTOs provide 95% thermal efficiency and also provides the added benefit of locating the unit in the same area as the DNCG sources, thereby saving long runs of large diameter ducting and overall installation cost.

Other Considerations

Several other factors affect the choice of an incineration point for NCG. These are usually site specific. The age, size and operating condition of the locations available are important.

As a general rule, the higher the capacity of the location selected, the better, as the NCG will have less impact on the operation of the larger equipment.

The uptime of the equipment used to burn NCG is very important. A boiler or kiln that is off line frequently, or for long periods of time, is a poor choice. The Environmental Regulations in the United States require an uptime of 99% for burning concentrated NCG. It is virtually impossible to do this unless there are two points for NCG incineration.

The relative locations of the NCG sources and the point of incineration is also important, as this affects the length, and hence the cost, of the pipe line required to carry the gases. Generally, shorter piping runs are advantageous.

A final factor to consider is human nature. Few operating superintendents are willing to accept the responsibility of handling and burning NCG.

IMPACT

Installation and operation of NCG systems has an impact on the rest of the mill operation. The largest impact usually comes from the digester blow heat recovery system.

It is necessary for the blow heat system to collect and condense all the blow steam. Consequently, steam and methanol that was previously vented, are now recovered as hot foul condensates. The recovered heat must be reused or discarded.

Many mills are now using recovered blow heat to pre-evaporate black liquor, while others install cooling towers to get rid of it. Also, the methanol collected can increase the B.O.D. load to the effluent treatment system, forcing mills to add aeration capacity, or install foul condensate strippers.

Capturing and burning the reduced sulfur gases in the kiln, or scrubbing the SO₂ formed with white liquor, will increase the liquor sulfidity, and that may upset the mill sulfur balance, necessitating a change in mill makeup chemicals, away from saltcake, and toward caustic soda or sodium carbonate.

There is a new process available that will convert the TRS in NCG directly to sulfuric acid, which can be used to cut tall oil. This will help the overall mill sulfur balance.

It may be necessary to update and improve the operations of the turpentine recovery system, and multiple effect evaporators in order to make them compatible for NCG collection. For instance, the practice of bleeding air into the evaporator vacuum system in order to control vacuum, is not acceptable with NCG collection.

Finally, places where NCG are burned can no longer be looked upon as production devices only. They must also be considered pollution control devices. It is not uncommon for a mill to

run a kiln or a boiler strictly for the purpose of burning NCG, even when the production capacity is not needed.

CONCLUSION

In order to reduce odorous and hazardous air pollutants from kraft pulp mills, environmental regulations require that noncondensable gases containing TRS and HAPs be collected and incinerated. These gases are very corrosive, highly toxic, and if mixed with air, potentially explosive.

Consequently, great care must be taken in the design, construction, operation and maintenance of these systems. Fortunately, the technology exists to build NCG systems that are safe, efficient and reliable.

ACKNOWLEDGEMENT

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3. Black Liquor Properties Relevant to Evaporation

3.1 Introduction

The chemical, thermal, and transport properties of black liquor directly affect the design and operation of black liquor evaporators and concentrators.¹ The properties of interest include viscosity, boiling point rise, heat capacity, surface tension, inorganic solubility, density, and thermal conductivity. Knowledge of these properties is needed to assess accurately the flow and heat transfer characteristics of black liquor. The chemical characteristics of black liquor that can influence these properties include its elemental composition, the composition of its organic and inorganic components, its residual alkali content, and the molecular weight of the organic matter that it contains.

Black liquor is a complex mixture of water, inorganic chemicals, and organic matter. Each of these components plays an important role in determining its chemical characteristics and physical properties. One of the challenges in developing broadly applicable descriptions of the properties of black liquor is that a wide range of wood species and pulping conditions are used to produce paper making fiber. Both the wood species and pulping conditions influence the relative amounts of organic and inorganic constituents in black liquor, and the composition and molecular weight of the organic matter. Some properties of black liquor (e.g., heat capacity and density) depend on the nominal composition of the black liquor solids. These properties differ from mill to mill, but they are related directly to the charge of chemicals used in pulping or the yield and Kappa number of the pulp. Others, such as viscosity, depend on the wood species and the pulping conditions. They cannot be predicted as easily.

Another factor that impacts the chemistry and properties of black liquor in bleached kraft mills is the use of bleach plant filtrates in brownstock washing. The dissolved organic solids from

oxygen delignification or alkaline extraction stages are partially oxidized compared with the organic matter in kraft black liquor. The inorganic matter contains little sulfur and may contain more chloride than kraft black liquor solids. The use of these filtrates in brownstock washing impacts the composition of the resulting black liquor and its properties.

One purpose of this chapter is to provide an understanding of how the chemical characteristics and physical properties of black liquor depend upon wood species and process conditions. A second is to present the best available methods currently available for estimating these properties for process calculations.

3.2 Chemical Composition of Black Liquor

Kraft black liquor is the spent liquor from pulping of wood or other biomass with aqueous solutions of NaOH and Na₂S. It contains dissolved components from biomass, along with the spent inorganic chemicals from pulping, and water. It is the chemical composition of black liquor and the nature of the dissolved organic matter from wood that determines its physical and transport properties.

3.2.1 Black Liquor Composition

Wood contains mainly polysaccharides (cellulose and hemicellulose) and lignin, but also extractives and inorganic species. It is the larger molecules of lignin and hemicellulose extracted during pulping that play a dominant role in black liquor viscosity. The data in Table 1 are typical compositions for softwood and hardwood species in different regions of the world. Polysaccharides are the largest group of components in all wood species. Softwood species always contain more lignin than do hardwood species. Extractives, while present as minor constituents, account for 2-5% of most wood species.

Table 1. Lignin, cellulose, hemicellulose, and extractives content of wood species used for pulp manufacture. The values are normalized to 100% on a dry, ash-free basis.²

Northern Softwoods

Wood species	Lignin wt-%	Cellulose wt-%	Hemicellulose wt-%	Extractives wt-%
Balsam fir	29.4	39.2	28.8	2.7
Douglas fir	29.4	38.9	26.4	5.3
Eastern hemlock	30.7	37.9	28.0	3.4
Monterey pine	27.3	37.6	33.3	1.8
Scots pine	27.8	40.1	28.6	3.5
Norway spruce	27.6	42.1	28.6	1.7
White spruce	27.6	39.6	30.7	2.1

Northern Hardwoods

Wood species	Lignin wt-%	Cellulose wt-%	Hemicellulose wt-%	Extractives wt-%
Red maple	25.5	42.2	29.0	3.2
Sugar maple	25.4	41.0	31.0	2.5
Common beech	25.1	39.9	33.7	1.2
Silver birch	22.3	41.6	32.9	3.2
Paper birch	21.9	40.2	35.2	2.7
Gray alder	25.2	38.8	31.3	4.7

Eucalyptus and Acacia

Wood species	Lignin wt-%	Cellulose wt-%	Hemicellulose wt-%	Extractives wt-%
Eucalyptus camaldulensis (River red gum)	31.8	45.8	19.5	2.8
Eucalyptus globulus (Blue gum)	22.0	51.5	25.3	1.3
Gmelina arborea (Yemane)	26.3	47.7	21.3	4.6
Acacia Mollissima (Black wattle)	21.0	43.3	33.9	1.8

Lignin and cellulose are natural polymers, or macromolecules. In wood, lignin consists of a 3-dimensional, cross-linked network comprised mainly of phenylpropane monomer units connected by ether linkages or carbon-carbon bonds. By contrast, cellulose is a linear, crystalline polymer composed of cellobiose units. Hemicelluloses are more diverse in composition. They are composed of building blocks of five- or six-membered rings, with each ring containing an oxygen atom.

During kraft pulping, the alkali and sulfide present in white liquor attack the lignin and polysaccharides in wood, reducing their molecular size and rendering them soluble. Fragmentation of lignin is by cleavage of both α - and β - aryl ether linkages, the predominant linkages in both hardwood and softwood lignins.³ Both the hydroxide and hydrosulfide ions in kraft white liquor contribute to lignin degradation. Polysaccharides, on the other hand, are attacked only by the hydroxide ions. The main reactions of the polysaccharides are saponification of the acetyl groups in hemicelluloses, “peeling” or sequential loss of sugar units, random cleavage of the main polysaccharide chain, and removal of methoxyl and other groups

from xylans. Tall oil and other extractives may be saponified, but are extracted without reacting further with alkali or hydrosulfide ion.

Most of the alkali in white liquor consumed in pulping is consumed in neutralizing the acidic phenolic groups in lignin and its degradation products, the organic acids formed from the polysaccharides, and the resin acids. Black liquor consists of the dissolved organic matter, dissolved cooking chemicals, and dissolved inorganic matter from the wood or impurities in the white liquor.

White liquor contains NaOH and Na₂S and smaller amounts of sodium sulfate (Na₂SO₄), sodium thiosulfate (Na₂S₂O₃), sodium carbonate (Na₂CO₃), and sodium chloride (NaCl). During pulping, much of the Na₂S is oxidized to Na₂S₂O₃, and most of the NaOH is consumed in neutralization of organic acids. The resulting inorganic salts and the sodium associated with the organic anions comprise most of the inorganic part of black liquor.

The range of heating values and elemental composition of kraft black liquors from northern hardwood and softwood species and Brazilian eucalyptus are shown in Table 2.

Table 2. Heating values and elemental composition of black liquor from pulping of five different groups of wood species, presented as averages and standard deviations.⁴ The numbers in parentheses for each different group of wood species indicate the number of black liquors analyzed.

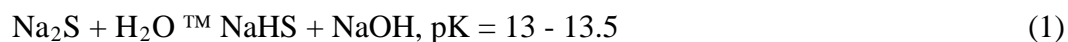
		HHV	S	C	Na	K	Cl	H	N
		MJ/kg ds	wt-% dry solids						
Northern softwood (149)*	Average	14.09	4.65	34.36	19.45	2.0	0.38	3.46	0.102
	St. Dev.	0.69	0.91	1.91	1.57	0.92	0.23	0.23	0.041
Northern hardwood (25)	Average	13.4	4.02	33.48	19.12	2.2	0.43	3.45	0.135
	St. Dev.	0.68	0.81	1.48	1.79	0.93	0.19	0.34	0.043
Birch (36)	Average	13.38	5.23	32.3	20.47	2.17	0.38	3.35	0.131
	St. Dev.	0.64	0.95	1.43	1.43	0.63	0.24	0.18	0.047
Eucalyptus (91)	Average	14.17	4.16	34.61	19.04	2.09	0.82	3.27	0.171
	St. Dev.	0.83	1.04	1.98	1.85	1.5	0.68	0.23	0.092
Mixed tropical woods (21)	Average	14.02	3.63	35.08	19.75	2.62	1.04	3.51	0.153
	St. Dev.	0.86	0.62	2.07	2.2	1.72	0.74	0.25	0.048

*Number of black liquors analyzed for each wood category.

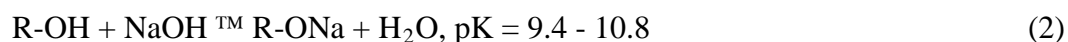
Acid-Base Behavior of Black Liquor

Black liquor contains several chemical species that act as buffers and influence its pH. These include:

- sodium sulfide, which buffers the pH according to the equilibrium



- phenolate groups on lignin, which buffer the pH according to the equilibrium



- sodium carbonate, which buffers the pH according to the equilibrium



Because of these buffers, the residual alkali content of black liquor cannot be measured by direct acid-base titration. SCAN test procedures SCAN-N 33:94 and SCAN-N 31:94 are used for measuring the residual active and effective alkali content in black liquor. The residual active alkali is defined as the sum of the concentrations of hydroxide and hydrosulfide ions in black liquor, while the residual effective alkali is the hydroxide ion concentration.

3.2.2 Molecular Size and Conformation of Kraft Lignin and Polysaccharides

The size and shape of large molecules in black liquor play a dominant role in its viscosity. The weight average molecular weight of alkali lignin in black liquor is typically 10,000 to 50,000, while the number average molecular weight range is 1000 to 5000. These values reflect the broad distribution of molecular weights, from a few hundred to over 100,000. Over 20% of the alkali lignin in kraft black liquor can have a number average molecular weight greater than 5,000; up to

10% can be greater than 10,000. Softwood kraft black liquors contain more of the high molecular weight alkali lignin than do hardwood liquors.

The conformation and apparent size of kraft lignin molecules changes with pH. At a pH of 12.5 or higher, the phenolate moieties of alkali lignin are completely ionized, and the alkali lignin molecules are soluble. In solution, they assume a compact, spherical shape. At a pH below 12.5, the phenolate groups are not completely ionized and the alkali lignin molecules are less soluble. The lignin molecules associate with each other, forming larger alkali lignin structures that behave as molecules of higher molecular weight. At pHs substantially below 11.5, alkali lignin precipitates from solution.

Polysaccharides behave quite differently from alkali lignin in black liquor. They tend to form loose, expanded random coils. The volume encompassed by polysaccharide molecules is much greater than that of lignin molecules of the same molecular weight. In the pH range normally encountered with black liquor, the polysaccharide molecules can entangle with a number of lignin and other polysaccharide molecules. By doing so, they can have a greater influence on the viscosity of black liquor than would be expected based on their molecular weight alone. This effect is more pronounced with hardwood kraft liquors, which have higher polysaccharide content than do softwood kraft liquors.

3.2.3 *Extractives*

The extractives in black liquor fall into three categories: resin acids, fatty acids, and neutral species. Resin acids are aromatic molecules such as abietic acid. Fatty acids are long-chain hydrocarbons with a carboxylic acid group on one end; two examples are oleic or linoleic acid. Neutral species consist of high molecular alcohols that do not contain carboxylic acid groups and therefore cannot be saponified.

Fatty and resin acids exist as their sodium salts in black liquor. These relatively long molecules have a polar, hydrophilic end where the carboxylate ion is located, and a neutral, hydrophobic (hydrocarbon) end. These molecules clump together to form micelles with the hydrophobic ends together in the center and stabilized by the ionized hydrophilic groups at the outer surface. The

micelles are small, 2 to 20 nm, and form a stable suspension within black liquor. The neutral species are soluble in the hydrocarbon-rich regions within the micelles.

As black liquor is concentrated, the increasing concentration of sodium ions tends to neutralize the negative charge that keeps the micelles dispersed. This causes the micelles to coalesce as a soap phase. The solubility of the soap is normally at a minimum when the dry solids content of black liquor is 25-28%.

3.2.4 Turpentine

Softwoods contain a number of small, neutral, ringed structures called terpenes. They contain mainly α - and β -pinenes. They are not very soluble in black liquor and are relatively volatile. Most of the terpenes are flashed when the digester is blown and/or stripped from black liquor during evaporation. They are collected from digester and evaporator condensates as water-insoluble turpentine.

3.3 THERMAL AND TRANSPORT PROPERTIES

3.3.1 Viscosity

Viscosity is the property of a fluid that describes how easily it flows. It is formally defined as the ratio of the shear stress to the shear rate for a fluid when subjected to a shearing force.

Weak black liquor, at a dry solids content of about 15-18%, is quite thin, having a viscosity two or three times that of water. Its viscosity increases rapidly with dry solids content, and it decreases with temperature. The nature and amount of the organics molecules, their molecular weight and colloidal state, are important in determining the viscosity of specific black liquors. These variables are affected by wood species pulped, the pulping conditions, and the subsequent processing conditions to which the black liquor is subjected. They vary considerably, so that

black liquors from two different mills, or from one mill at two different times, can have viscosities that differ by almost an order of magnitude when measured at the same dry solids content and temperature.

The shear rate to which black liquor is subjected can affect its viscosity. Most common fluids such as water have viscosities that are not affected by shear rate. These fluids are referred to as Newtonian fluids. For Newtonian fluids, the shear force is directly proportional to the shear rate. The constant of proportionality is the Newtonian viscosity.

$$\tau = \mu \, dv/dy \quad (4)$$

where:

τ = shear stress, Pa

μ = Newtonian viscosity, Pa-s

v = velocity, m/s

y = distance from surface normal to the direction of flow, m

Black liquor, like many other polymer solutions and melts, exhibits non-Newtonian behavior under certain conditions. At low shear rates its viscosity is independent of shear rate. At higher shear rates, however, its apparent viscosity decreases with increasing shear rate. This is referred to as shear thinning. Flow calculations with non-Newtonian fluids are often complicated. Fortunately, most black liquors exhibit Newtonian behavior at normal conditions in pulp mill operations.

Note: both *dynamic* and *kinematic viscosities*, are reported for fluids. Dynamic viscosity (μ) is used in most engineering calculations, and all viscosity data included in this chapter are dynamic viscosity values. The two are related as $\mu = v/\rho$ where v is the kinematic viscosity and ρ is the density of black liquor.

Dynamic viscosity is reported in several different but equivalent sets of units. The relationship between them is shown in Table 3.

Table 3. Equivalent Units for Dynamic Viscosity

1 Poise = 1 dyne-second/sq centimeter

1 Pascal-second (Pa-s) = 1 Newton-second/sq meter

100 cP = 1 Poise

0.1 Pa-S = 1 Poise

1 centipoise (cP) = 1 mPa-s

Impact of temperature and dry solids content on the viscosity of black liquor

Viscosity data for a kraft black liquor from pulping of slash pine are presented in **Figure 1**, for a range of temperatures and dry solids content. These data show a rapid increase in viscosity with dry solids content, and a rapid decrease with temperature. The viscosity data follow the characteristic behavior for black liquors and many other fluids. The data in Figure 1 qualitatively are typical of all black liquors in the Newtonian behavior range.

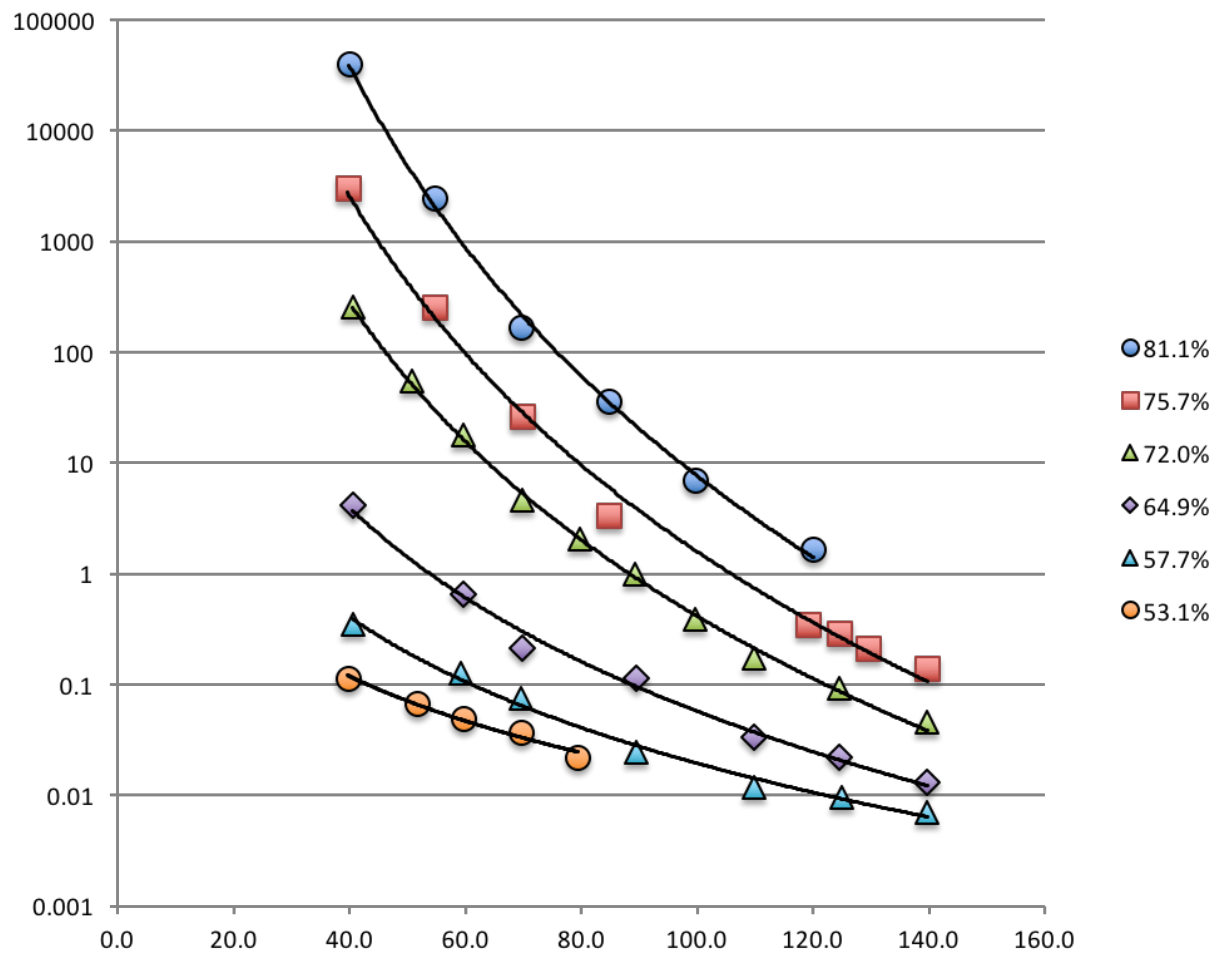


Figure 1. Viscosity versus temperature and dry solids content for a kraft black liquor from pulping of slash pine to Kappa 107.⁵

Impact of pulping conditions

The viscosity of black liquor is a strong function of the Kappa number to which wood is pulped. This is illustrated in Figures 2 and 3. Both figures contain data for black liquors from pulping of slash pine to four different kappa numbers. Viscosity was measured at high dry solids content, near 76%.

In Figure 2, the viscosity versus temperature data for the two black liquors from pulping to the highest (107) and lowest (18) kappa numbers are nearly the same. The viscosities of those from pulping to the two intermediate kappa numbers (40 and 51) are also nearly the same, but differ by a factor of 50 from the black liquors from high and low Kappa pulps.

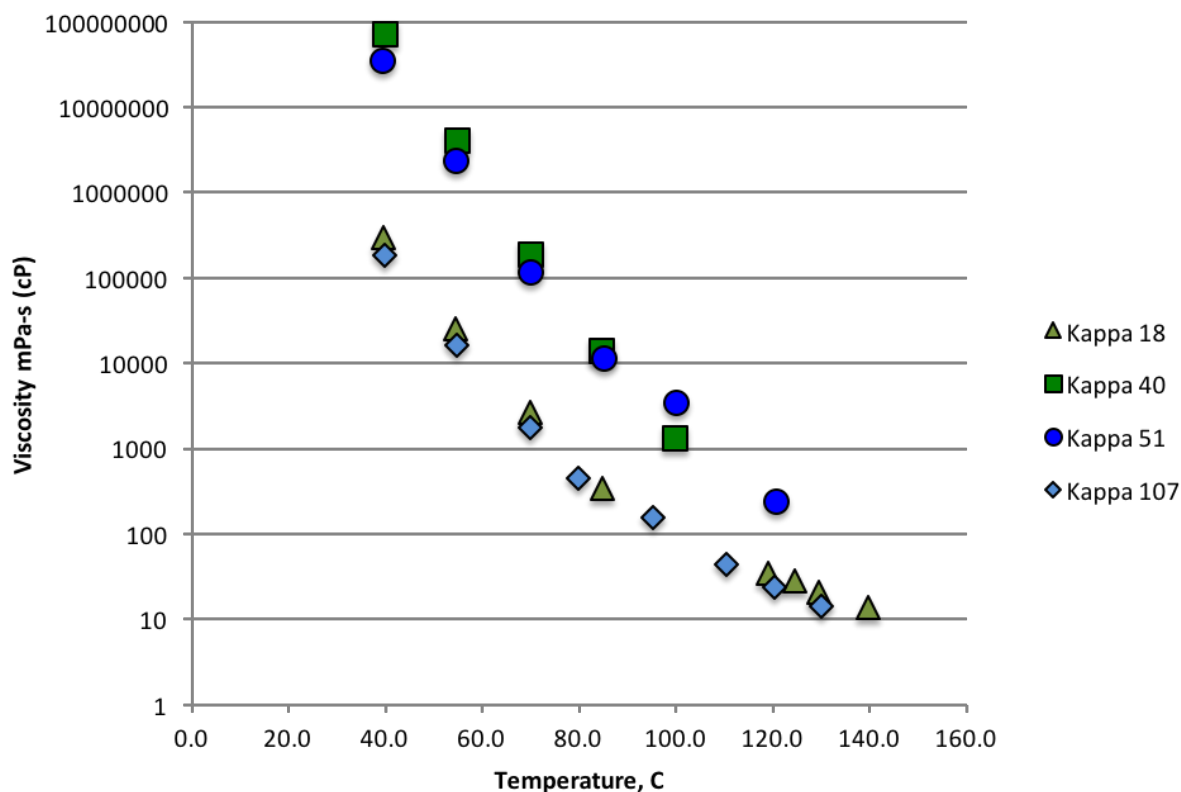


Figure 2. Viscosity versus temperature for black liquors from pulping of slash pine to the kappa numbers indicated.

Similar viscosity data, when plotted in Figure 3 versus kappa number of the pulp produced, show clearly a maximum in viscosity at intermediate kappa numbers (between 40 and 60). All viscosity measurements were made at the same temperature and dry solids content. The maximum in viscosity is attributed to differences in the length of the extracted lignin and carbohydrate molecules at different kappa numbers. Pulping to high kappa numbers extracts relatively little lignin and hemicellulose, and mostly the more soluble, lower molecular weight lignin. Pulping to low kappa numbers extracts most of the lignin and much of the hemicellulose in wood chips. The long pulping times required to reach low kappa numbers also shorten the length of the dissolved lignin and

hemicellulose molecules. At intermediate kappa numbers, significant amounts of lignin and hemicellulose are extracted but pulping times are not long enough for the larger, higher molecular weight molecules to be cut into shorter ones. It is the longer lignin and hemicellulose molecules in black liquor produced at intermediate kappa numbers that results in the highest viscosity black liquors in Figure 3. Differences in other composition factors such as organic to inorganic ratio, the concentrations of sodium, chloride, sulfite, and carbonate ions, and residual active alkali (REA) also contribute to differences in the viscosity of different black liquor.

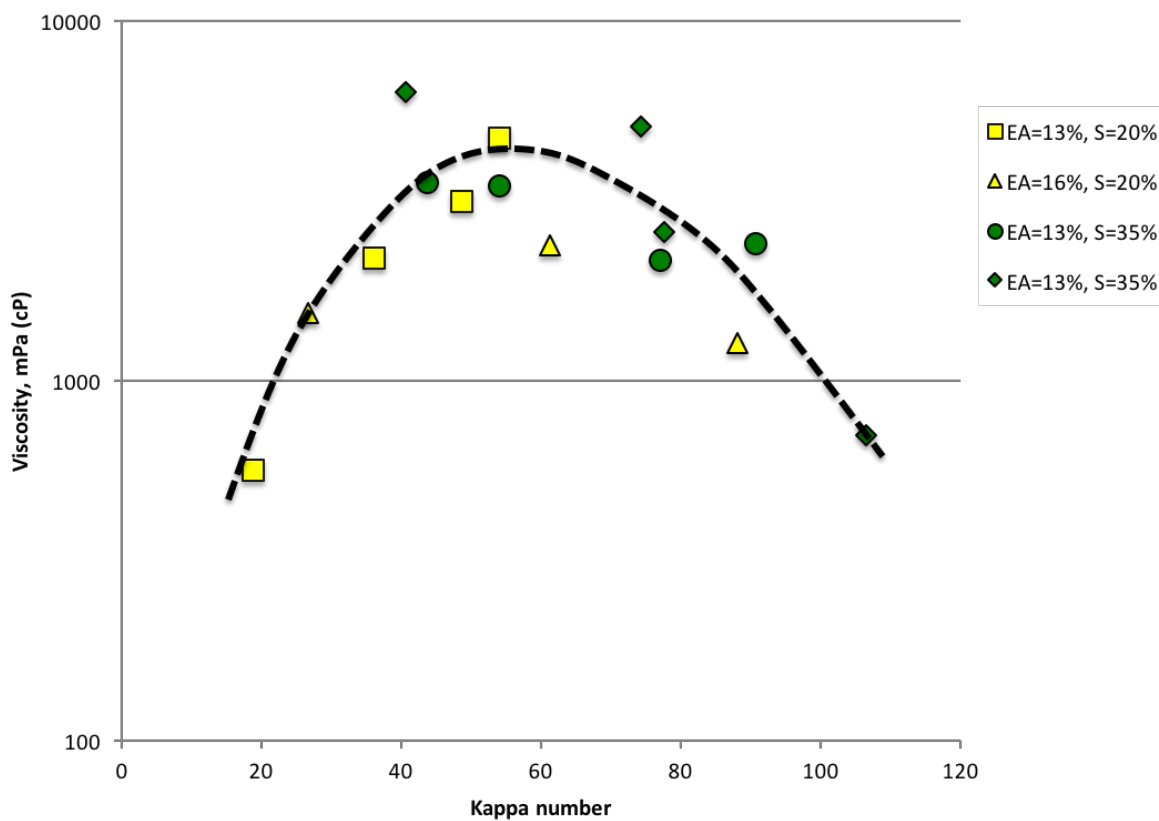


Figure 3. Viscosity of black liquors from pulping of slash pine to different kappa numbers

(Zaman and Fricke, 1996). Measurement conditions were 120°C, 80 wt-% dry solids content. EA is the effective alkali charge as wt-% of dry wood and S is the sulfidity of the white liquor used in pulping.

Comparison of the viscosity of black liquors from pulping of northern hemisphere woods and eucalyptus

The viscosity versus dry black liquor solids content data shown in Figure 4 are for kraft black liquors from three different northern wood species. Viscosity data for a eucalyptus black liquor and a Nordic softwood sulfite liquor are included for comparison. The liquor-to-liquor variation in viscosity among these data increases from a factor of 2 at 45 wt% dry solids content to a factor of 10 at 70 wt% dry solids content. This much variation makes it difficult to find correlative models to predict black liquor viscosity from different wood species at the same temperature and dry black liquor solids content. Measured viscosity data from representative black liquors is essential for process design and evaluation of black liquor evaporators. Laboratory-generated liquors used for viscosity measurements must be produced at pulping conditions as closely as possible to those expected to be employed at full scale.

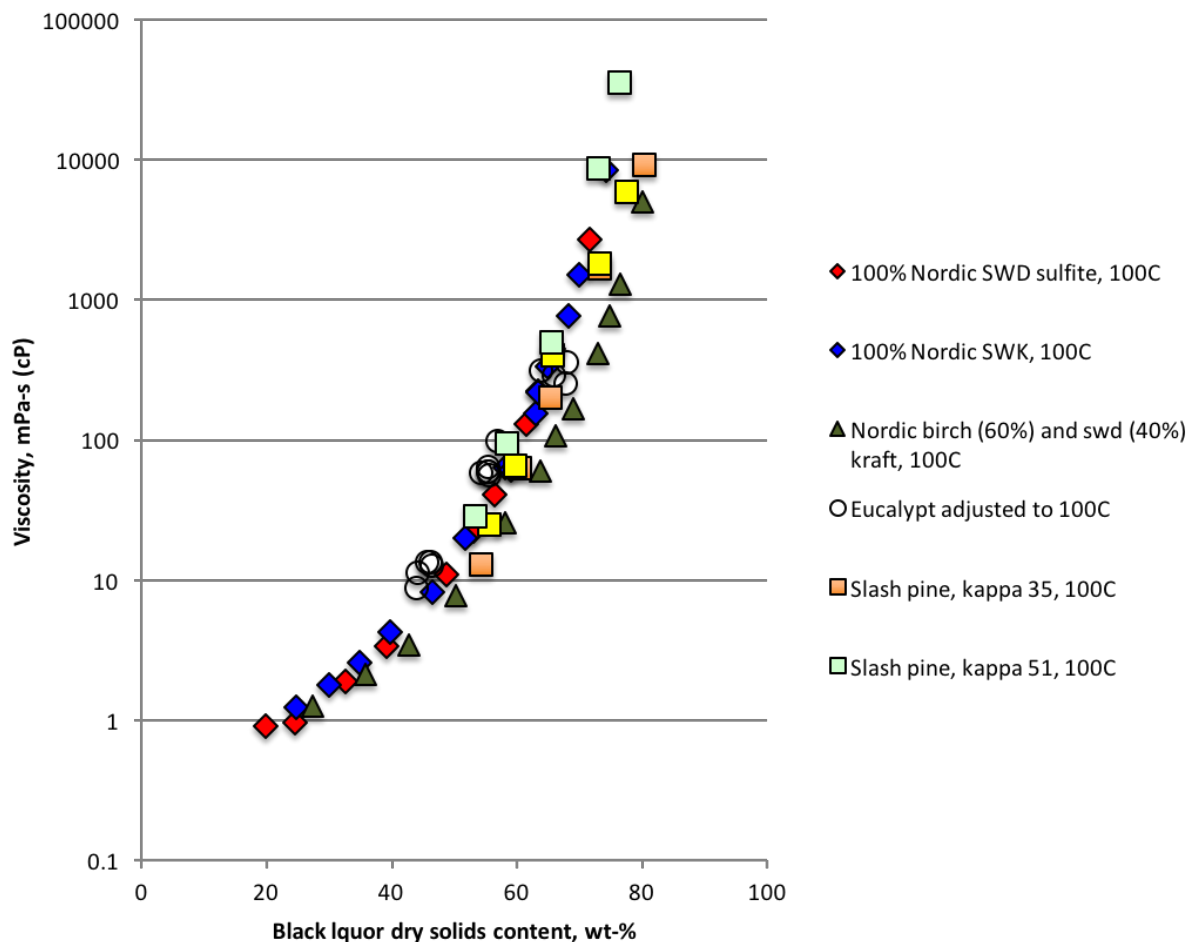


Figure 4. Viscosity of six different black liquors, measured at 100C and the dry solids contents indicated.

Viscosity of black liquor from pulping of agricultural residues and other non-wood raw materials

Black liquors from pulping of these fiber sources are notoriously difficult to evaporate. The viscosity data in Figure 5 are for black liquors from bagasse, straw, and bamboo. Data for two northern softwood black liquors are included for reference. The viscosities of the non-wood

black liquors in Figure 4 are a factor of 10 higher than those of the northern softwood black liquors when compared at 60 wt-% dry black liquor solids. This is typical of the viscosity of those liquors.

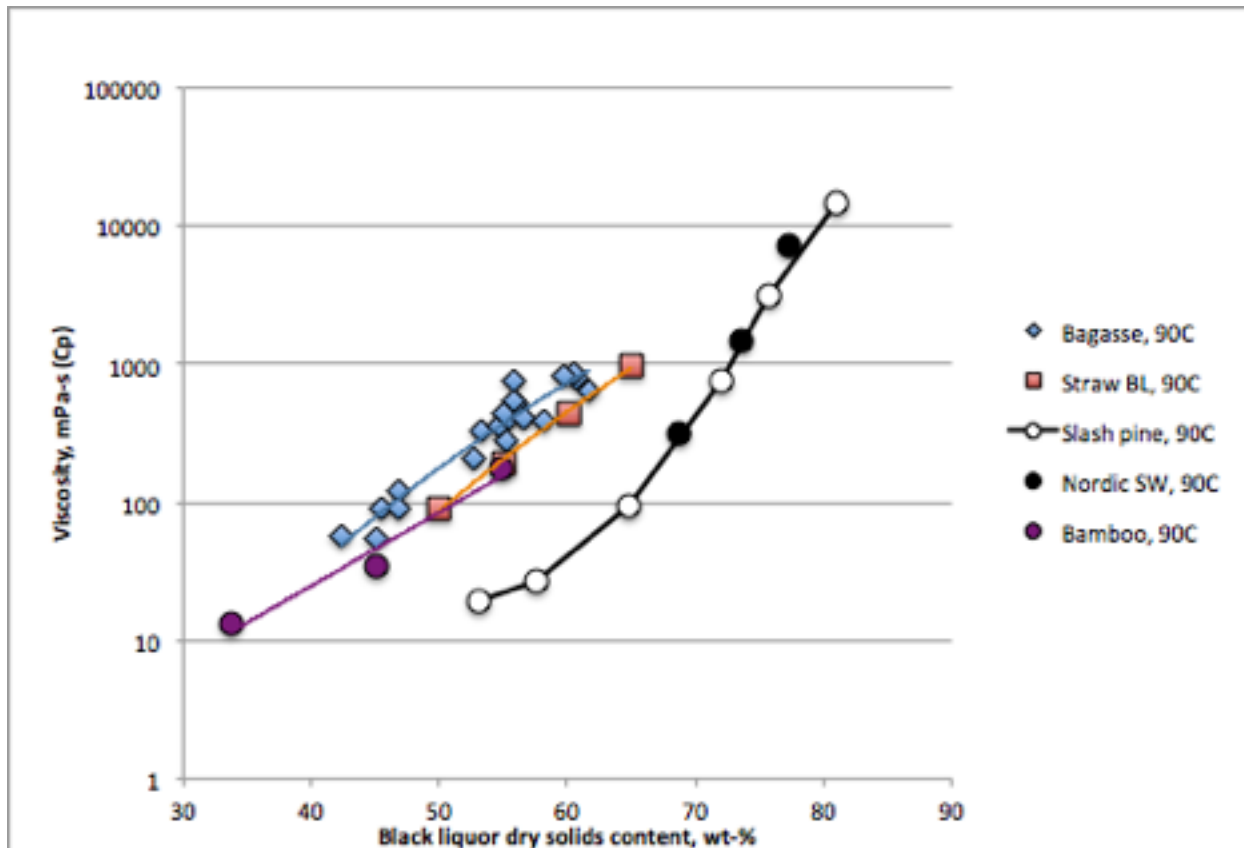


Figure 5. Comparison of viscosities of black liquor from pulping of bagasse^{6, 7, 8} straw, bamboo^{9,10}, and two northern softwoods.

The data in Figure 6 show how liquor temperature, dry solids content, and residual alkali content impact the viscosity of black liquor from pulping of bagasse. The impact of dry solids content and liquor temperature are as expected from other viscosity data. Differences in residual active alkali (RAA) content have a large impact on viscosity. The viscosity of bagasse black liquor decreases by a factor of 10 when the residual active alkali (RAA) is doubled. A low residual

active alkali content is common for black liquors produced from pulping of non-wood fiber raw materials. This can account for the high viscosity that makes these black liquors difficult to concentrate by evaporation.

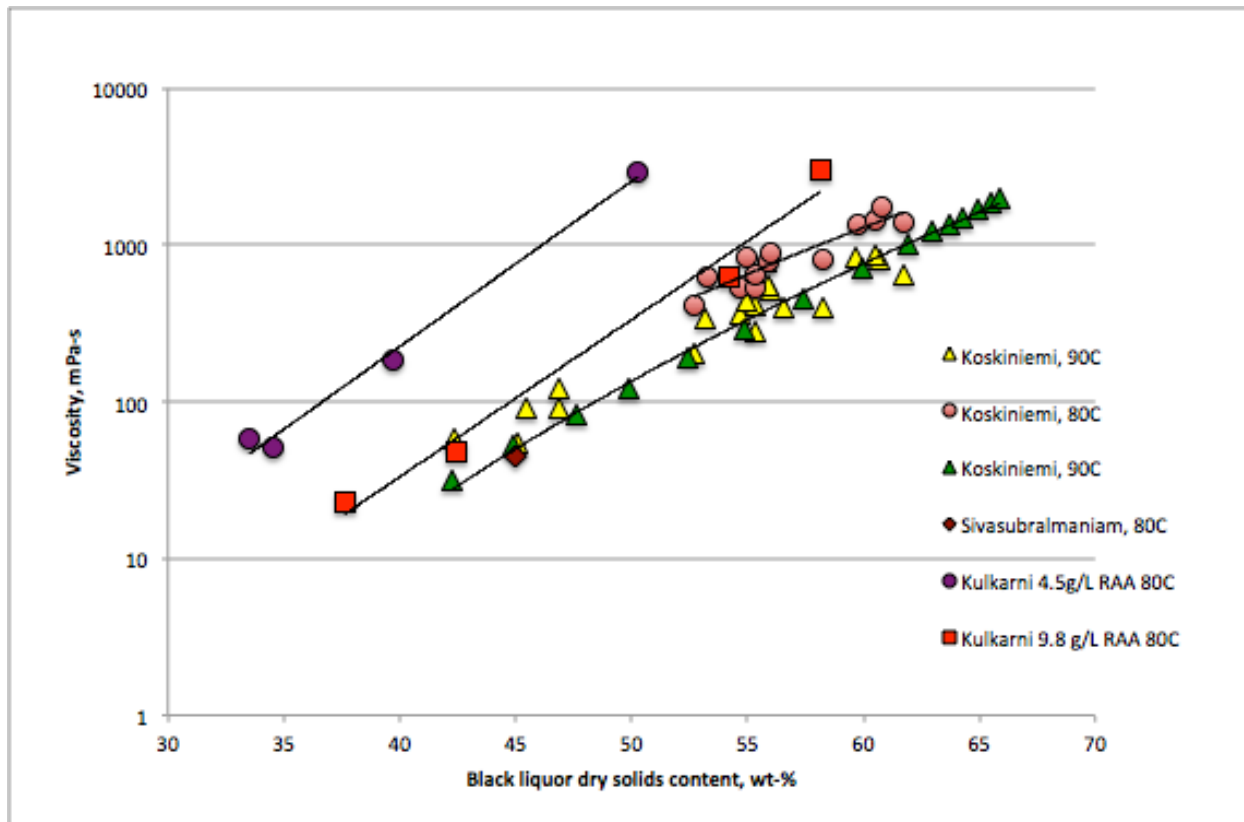


Figure 6. Viscosity of black liquor from pulping of bagasse at different conditions.⁶⁷⁸

High concentrations of silica in black liquors from pulping of non-wood fiber sources increases the viscosity of these black liquors. Figure 7 shows viscosity values for a straw black liquor that contained 3.5 g/L silica as SiO_2 and the same liquor after acidification to pH 9.9 and filtration which removed 93% of the silica. That amount of silica removal decreased the liquor viscosity

by a factor of 3. Methods for desilification of black liquor have been developed. A critical concern is how to acidify to remove silica without precipitating lignin.¹¹

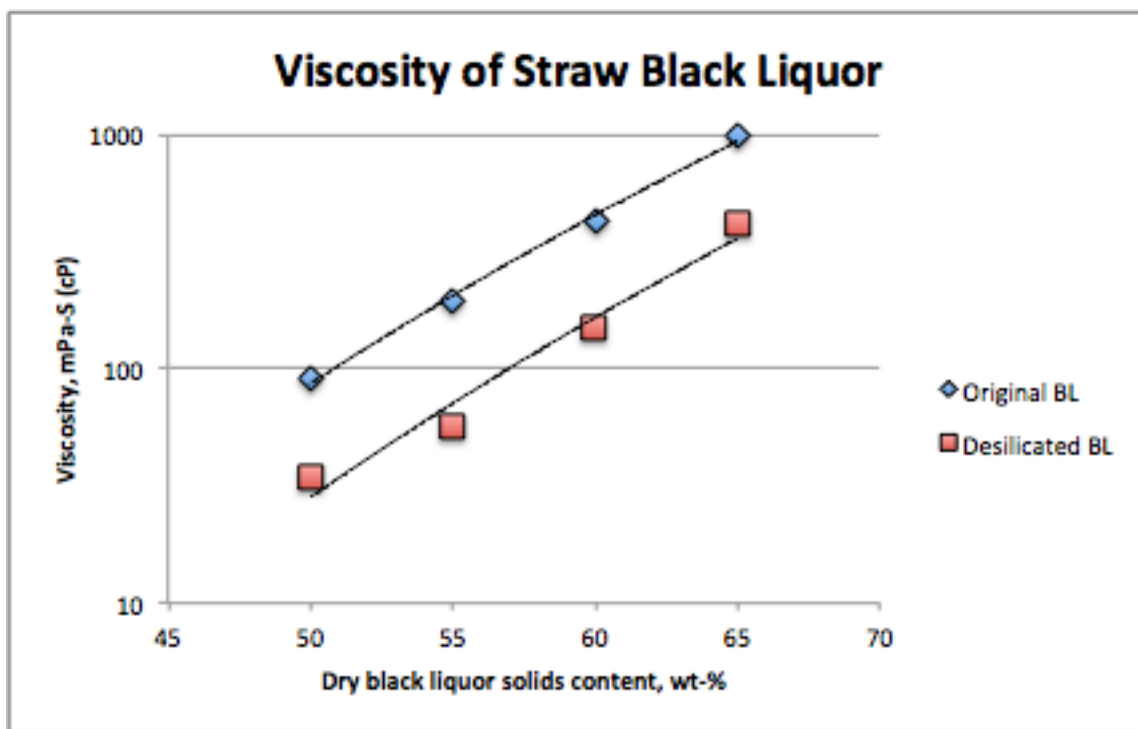


Figure 7. Impact of desilification of straw black liquor on its viscosity.⁶ Viscosity measured at 125°C.

Impact of lignin removal

Removing lignin from black liquor decreases its viscosity. Figures 8 and 9 show the extent of viscosity reduction when lignin was precipitated from black liquor by acidification with CO₂ and then removed by filtration. The liquors used in both cases were Nordic softwood black liquors. The data in Figure 8 suggest that up to an 80% reduction in viscosity can be obtained by removing 60% of the lignin in black liquor. The data in Figure are consistent with that, showing that a nearly 90% reduction in viscosity can be obtained by removing 67% of the lignin. Viscosity reductions of 20% to 70% were obtained with other black liquors, with greater viscosity reductions when more lignin was removed.¹²

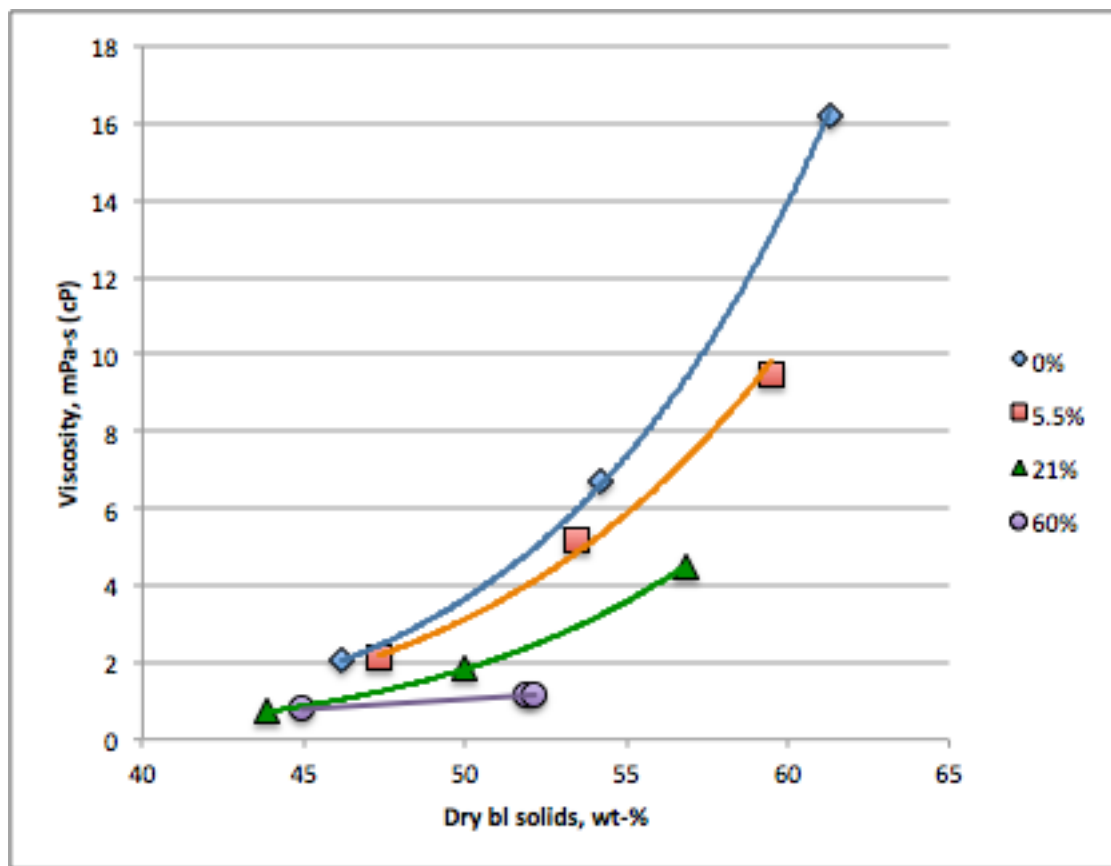


Figure 8. Viscosity at 120°C of an unmodified Nordic softwood kraft black liquor and the same liquor at three different levels of lignin removal.¹³

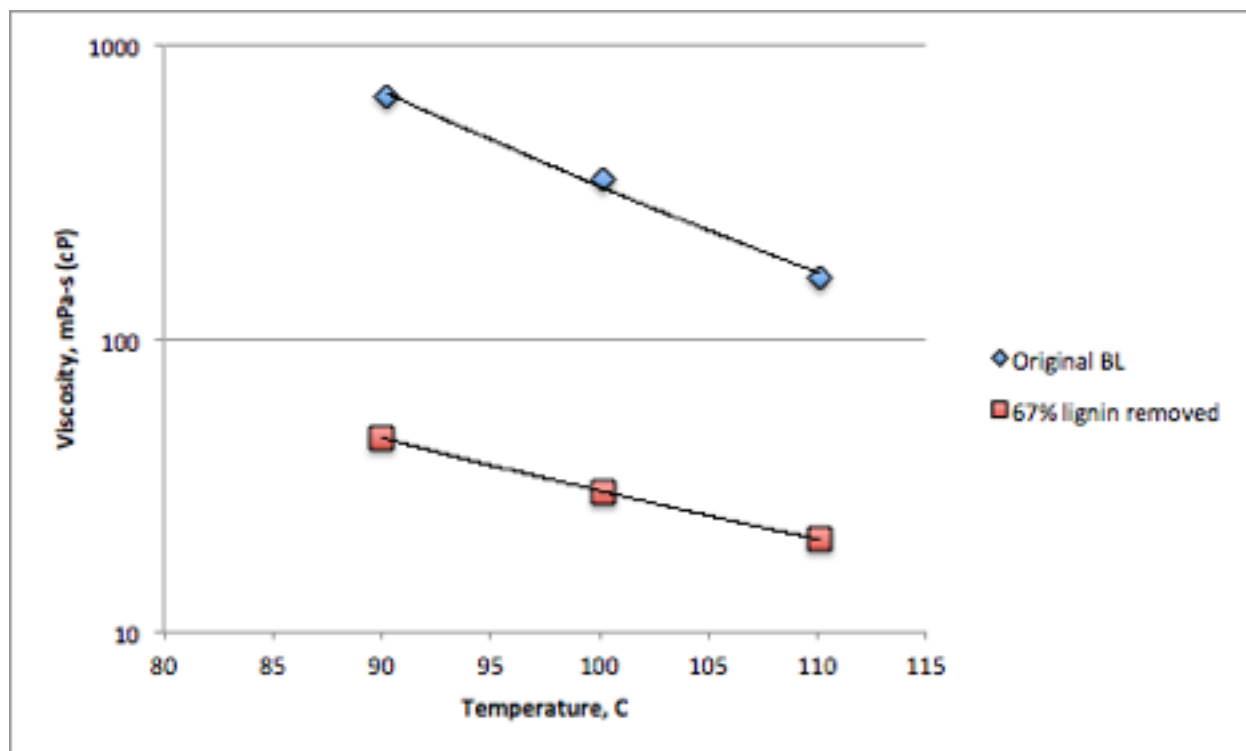


Figure 9. Viscosity of unmodified Nordic softwood kraft black liquor and the same liquor modified by removing 67% of lignin. Viscosity was measured at 102°C. Data are from Moosavifar, 2009.

Non-Newtonian behavior

At normal processing conditions black liquor almost always behaves as a Newtonian fluid - that is, its viscosity depends only on temperature and dry solids content but not on shear rate. At very high dry solids contents or certain other conditions, however, black liquors can exhibit thixotropic or pseudoplastic behavior. These are defined as follows.

Pseudoplastic behavior is reversible shear thinning. It usually occurs because the shear forces disturb the long polymer molecule chains from their preferred conformation. Thixotropy is a characteristic which causes the apparent viscosity of a material to change from its initial value when shear is first applied to another steady value after some period of shearing. This usually

occurs when polymer molecules that are initially entangled become aligned in flow, making it easier for shear layers to slip past each other. Thixotropy is similar to pseudoplasticity, but the time scale for changes in the fluid is not negligible. Both thixotropic and pseudoplastic behavior has been observed in hardwood kraft liquors including eucalyptus liquors, but not in softwood kraft liquors. The degree of pseudoplasticity correlated well with the carbohydrate content of the liquors.

Thixotropic behavior per se should not be a concern in a practical mill environment because every effort is made to keep the liquor reasonably thin (viscosity < 300 mPa-s (cP) by maintaining a high liquor temperature and keeping it flowing. This is not the case during a shutdown or mill upset, when liquor cooling can result in the extraordinarily high viscosity of thixotropic liquors. This can make black liquor very difficult to process in equipment that was not designed for such a high viscosity fluid.

Reduction of black liquor viscosity by thermal treatment

It is possible to change the viscosity of black liquor irreversibly by heating it to temperatures above those normally encountered during evaporation. When sufficient alkali is present, thermal treatment above 140°C results in depolymerization of the high molecular weight lignin - in effect the dissolved lignin and polysaccharides continue to be “pulped”. The data in Figure 10 illustrate the effect of thermal treatment on black liquor viscosity. A commercial process for controlling black liquor viscosity is available and is in use in mills in several countries.¹⁴

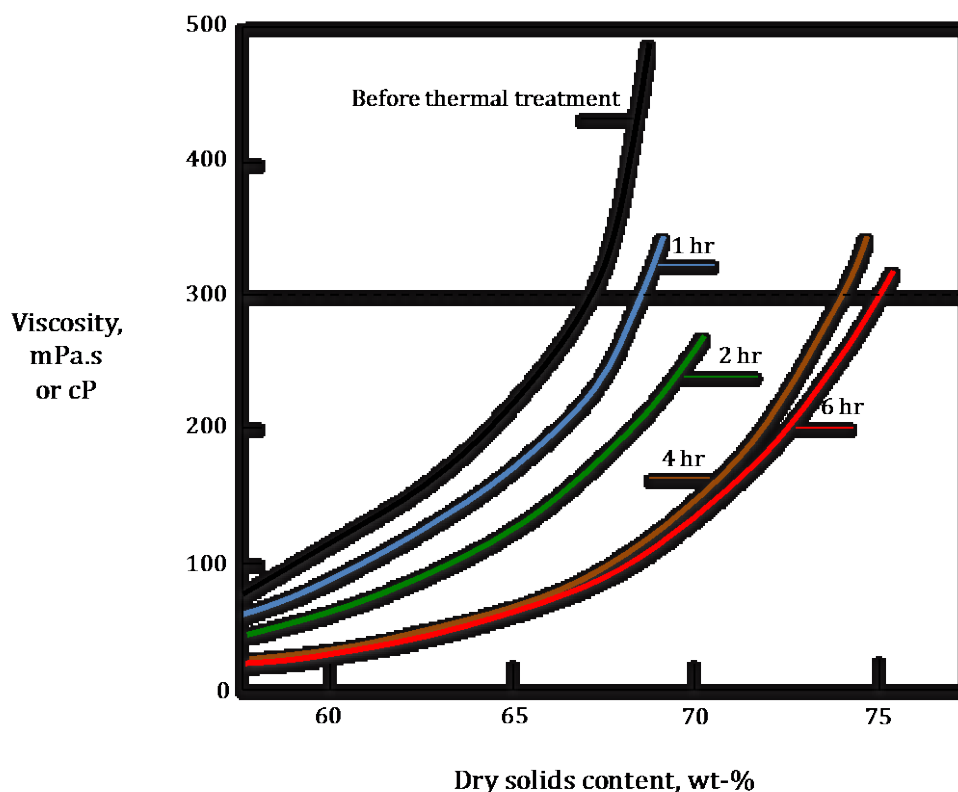


Figure 10. Impact of thermal treatment on the viscosity of a kraft black liquor.¹⁵

Controlling black liquor viscosity in the mill environment

From an operating point of view, the viscosity of black liquor can be controlled by temperature and alkali addition. Temperature is the normal control variable and is the most effective for liquors of normal alkali content. The effect on viscosity of changing the temperature of black liquor can be determined from the data in Figures 1, 2, and 6.

Alkali addition to either the black liquor or the digesters is effective in reducing the viscosity of black liquors with low alkali content. Adding alkali to the black liquor is easy and useful in solving short-term high viscosity problems or stabilizing viscosity after black liquor oxidation. It can be added to the weak or heavy liquor, or at the cascade evaporator. Adding alkali to black liquor before the evaporators can solve problems with poor evaporator performance associated with high liquor viscosity. For mills with black liquor oxidation, alkali can be added before

and/or after the oxidation towers if poor control of the oxidation towers creates high viscosity problems. Determining the optimal EA can be accomplished by having viscosity measurements made at varying levels of hydroxide addition.

Thermal treatment of black liquor has been used successfully to reduce the viscosity of black liquor. Removal of lignin from black liquor, a newer alternative, is being explored for that purpose. The effective lignin removal on black liquor viscosity is illustrated in Figure 8.

3.3.2 *BOILING POINT RISE*

The addition of soluble material to a liquid changes its boiling characteristics. Boiling point rise (BPR) is the difference in boiling temperature between the solution and the pure solvent when measured at the same pressure. BPR increases with solute concentration and can be as high as 27°C (49°F) for concentrated black liquor. This property is crucial to the design and performance of black liquor evaporators. Heat transfer in these units is dependent on temperature difference between the condensing steam and the evaporating black liquor. BPR can reduce substantially the available ΔT for heat transfer.

Boiling point rise is a colligative property, proportional to the molal concentration of solute. Inorganic compounds are the dominant component in determining the BPR of black liquor. The reason for this is that they make up over 90% of the solute on a molar basis. As the dry solids content increases, the concentration of the inorganic salts in solution also increases, and this increases the BPR.

Shown in Figure 11 is a plot of BPR as a function of dry solids content from several sets of measurements using different measurement techniques.^{16,17,18,19} Above 50% dry solids content the BPR increases very rapidly with increasing solids content. The BPR values from the four sets of measurements do not differ much even though they are from four different black liquors measured with four different methods. The small differences between the data sets are due both

to composition differences in the liquors tested and in the test methods used. No single technique has been accepted as a standard for measurement of BPR.

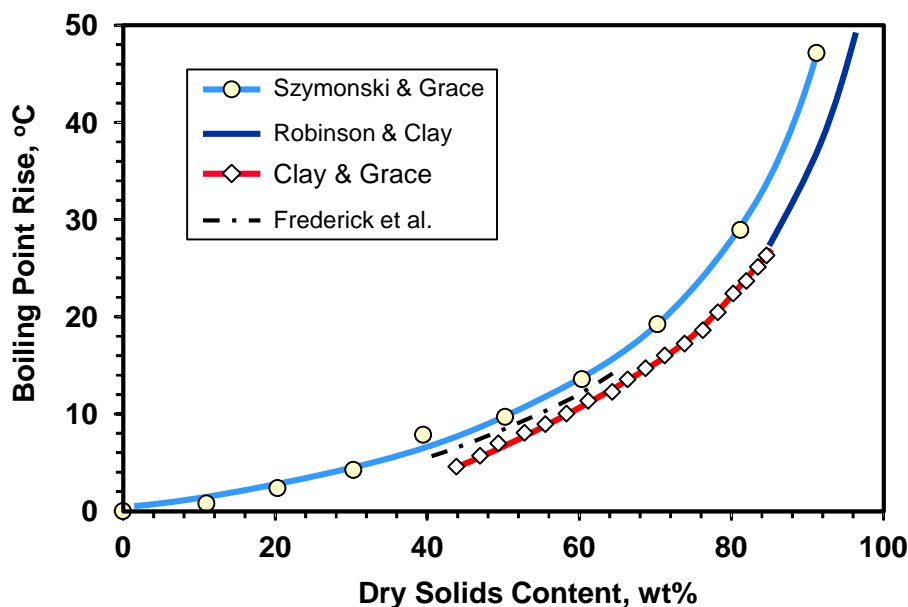


Figure 11. Black liquor boiling point rise data from various sources.^{16,17,18,19}

One aspect of the inorganic composition which has been demonstrated to affect BPR is the solubility of Na_2CO_3 and Na_2SO_4 . As will be discussed further in the following section, these two components crystallize from solution first as black liquor concentration is increased. For typical black liquors, these components begin to crystallize at about 50% dry solids content. When this occurs, the influence of concentration on BPR changes as shown in Figure 12. The BPR for two liquors is plotted as a function of the solids-to-water ratio ($S/100-S$) in this figure. At low concentrations the BPR is a linear function of solids-to-water ratio, though the slope is different for the two liquors. At about 50% dry solids content (solids-to-water of 1.0) the slope decreases due to the precipitation of some of the inorganic. This is further demonstrated by increasing the dry solids content of one of the liquors by adding Na_2CO_3 instead of by evaporating water. At 50% dry solids content the additional Na_2CO_3 is no longer soluble in the aqueous solution portion of the black liquor so no increase in BPR was observed. This is indicated by the horizontal line in Figure 12.²⁰

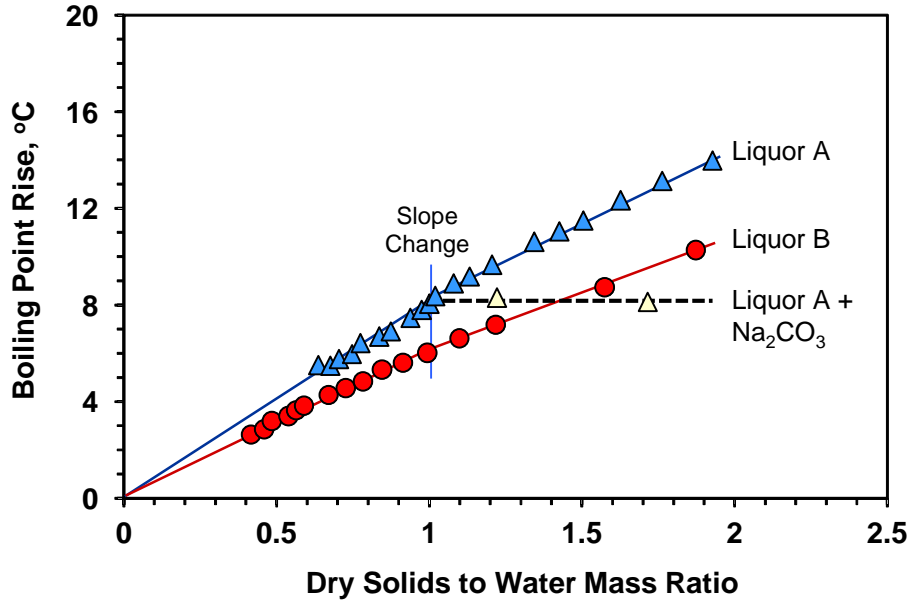


Figure 12. Boiling point rise as a function of the solids/water ratio for two black liquors. ¹⁶

The method of plotting the BPR data in Figure 12 is suggested by Raoult's law. For dilute solutions, the vapor pressure of the solvent is decreased in direct proportion to the molal concentration of solute. The boiling point increases as a result. For small changes in boiling point, the rise is directly proportional to the molality of the solute, which in turn is proportional to the ratio of dry solids content to water. Up to 50% dry solids content, the following expression can be used to estimate BPR:

$$\Delta T_b = \Delta T_{b50} (S/100-S) \quad (5)$$

where:

ΔT_b = boiling point rise (BPR), °C

ΔT_{b50} = boiling point rise at 50% dry solids content, °C

When no data is available, using ΔT_{b50} equal to 7.5°C (13.5°F) should result in no greater than 20% error for kraft black liquor. Above 50% total dry solids, the BPR of black liquor depends in a complex way on the solubility of the inorganic components. The upper limit on the BPR for dry solids content approaching 100 wt-% is approximately 55°C (99°F).

The most accurate method of estimating boiling point rise requires measured values of heat capacity and normal boiling point versus dry solids content. These parameters must be available for the black liquor of interest. The estimates are based on standard thermodynamic calculation methods.^{21,22} This method both gives excellent agreement with experimentally measured boiling point rise curves (shown in Figure 13) and can be used to estimate boiling point rise for liquors above the normal boiling point. It does, however, require extensive and accurate measurements of heat capacity and normal boiling point over the range of dry solids contents of interest.

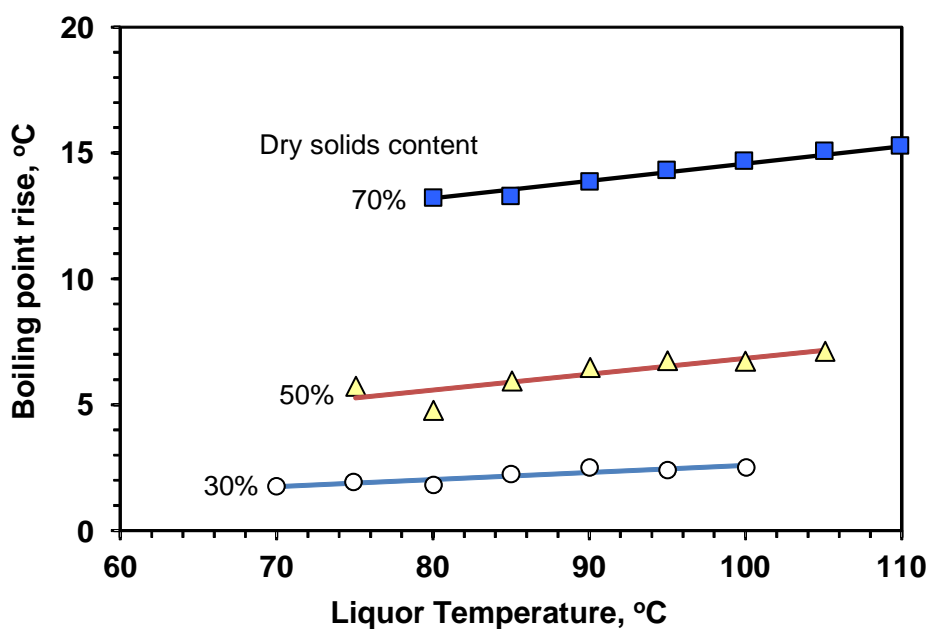


Figure 13. Comparison of experimental boiling point elevation data for black liquor (symbols) with values obtained from the vapor pressure estimations and extrapolations from Equations 8 and 9 (lines).

When heat capacity and normal boiling point data are not available, the data in Figure 11 can be used to estimate BPR at higher solids concentration and at atmospheric pressure. A small amount of data for one liquor is available which shows the trends above 50% solids for pressures above atmospheric pressure. When no data at elevated pressure are available, equation 6 for estimating the effect of pressure on boiling point rise for dilute solutions can be used to estimate the boiling point rise for black liquor at elevated pressure. The relationship between boiling point rise, salt concentration, and the properties of water is:

$$\Delta T_{\text{bpr}} = m \text{ MW}_w R (T_{\text{bw}})^2 / \Delta H_v \quad (6)$$

where:

ΔT_{bpr} = boiling point rise of black liquor, °C

$\Delta T_{\text{bpr}} = m$ = molality of the solution

MW_w = molecular weight of water, 18

R = gas constant, $8.314 \times 10^{-5} \text{ m}^3 \text{ bar} / \text{mol K}$

T_{bw} = boiling point of pure water, °K

ΔH_v = enthalpy of vaporization of water, J/kg

From this, the BPR at 50% solids for any pressure can be estimated in terms of the value at one atmosphere. The molality at 50% dry solids content is independent of pressure for particular liquors as long as the inorganic is completely dissolved, so that the value of ΔT_{bpr} at any pressure can be estimated as:

$$\Delta T_{\text{bpr}50\text{-p}} = \Delta T_{\text{bpr}50\text{-1 bar}} [T_{\text{bw-p}}^2 / T_{\text{bw-1 bar}}^2] [\Delta H_{v\text{-1 bar}} / \Delta H_{v\text{-p}}] \quad (7)$$

where:

$$\Delta T_{b50-p} = \text{BPR at 50\% dry solids content at pressure } p, ^\circ\text{C}$$

$$\Delta T_{b50-1 \text{ atm}} = \text{BPR at 50\% dry solids content at one atmosphere, } ^\circ\text{C}$$

$$T_{bw-p} = \text{boiling point of water at pressure } p, ^\circ\text{K}$$

$$T_{bw-1 \text{ atm}} = \text{boiling point of water at one atmosphere} = 373 ^\circ\text{K} (100^\circ\text{C})$$

$$\Delta H_{v-1 \text{ atm}} = \text{enthalpy of vaporization of water at one atmosphere} = 2,244 \text{ kJ/kg}$$

$$\Delta H_{v-p} = \text{enthalpy of vaporization of water at pressure } p, \text{ kJ/kg}$$

Equation 7 provides a method for extrapolating boiling point rise data taken at one pressure to other pressures of interest. Extrapolated values of boiling point rise based on Equation 8 are compared with the experimental data in Figure 13. The 90°C liquor data was used as the base point at each dry solids content, and the boiling point rise at other liquor temperatures was calculated using Equation 7. The results, shown as solid lines in Figure 13, agree very well with the experimental data.

3.3.3 DENSITY

The density of black liquor is important in the calculation of flow characteristics and static heads. At very low dry solids contents the density is close to that of water at the same temperature. At higher dry solids contents the density depends on the organic and inorganic material that constitute the solids. The density of most organic matter, including the lignin and carbohydrates in black liquor, is close to that of water while that for the inorganic matter is nearly twice that of water. As a result, the inorganic matter has the stronger effect on density of black liquor. The apparent density of black liquor can also be affected by entrained air. This air enters the liquor during various processing stages including brownstock washing and black liquor oxidation. It is

not an inherent constituent of the liquor, but can affect the apparent density both in operation and in laboratory testing.

Some data for the density of various spent pulping liquors is shown in Figure 15. as the specific gravity at 25°C (77°F) based on the density of water at 16°C (60°F). The specific gravity is very nearly a linear function of the dry solids content of the liquor for each type of liquor up to at least 55 w% dry solids content.

The slopes of the specific gravity versus dry solids content data in Figure 15 depend on the inorganic content of the individual liquors. These slopes correlate well with the inorganic content (expressed as ash equivalent) of the dry solids as shown in Figure 16. The inorganic content was obtained by ashing in oxygen rather than by the sulfated ash technique. Though the composition of the inorganic is changed by this process, the result is a direct measure of the inorganic content of the liquor. The excellent correlation between inorganic content and the specific gravity slope for the four pulping processes indicated the dominant role of inorganic matter in black liquor density.

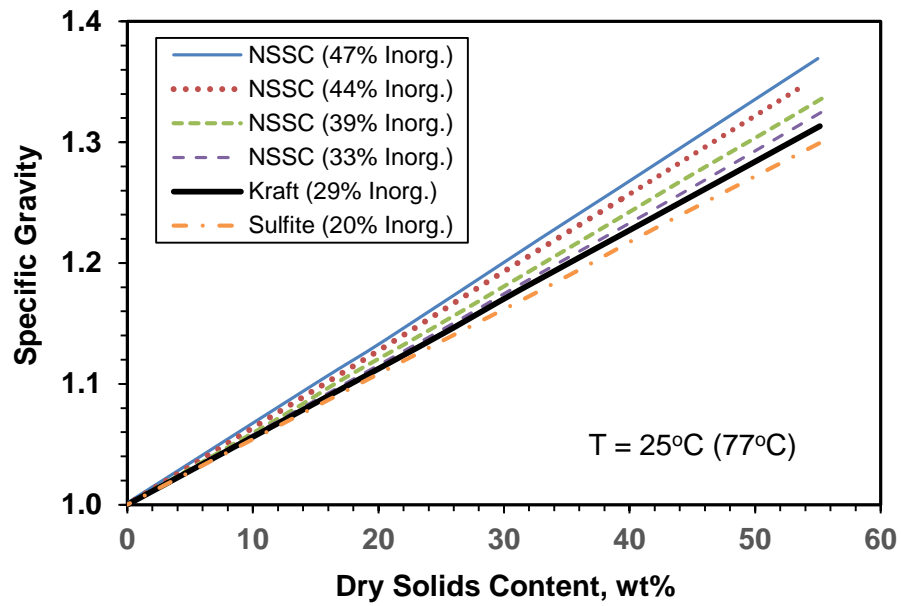


Figure 15. Specific gravity of typical spent pulping liquors.²³

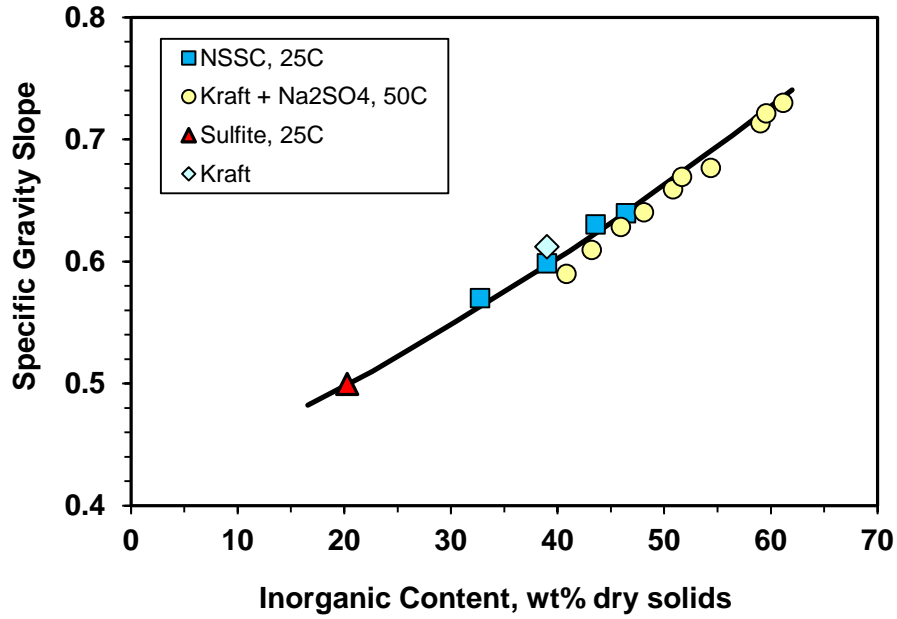


Figure 16. Slopes of specific gravity as a function of inorganic content for four spent pulping liquors.

A correlation that predicts the density of black liquors at 25°C and dry solids contents up to 50% is:²⁴

$$\rho_{25} = 997 + 649 S \quad (8)$$

where:

ρ_{25} = black liquor density at 25°C, kg/m³

S = black liquor dry solids mass fraction

The inorganic content of the liquor solids has the predominant effect on the density. Equation 8 predicts a higher specific gravity than indicated in Figure 15 because it represents the density of degassed black liquor.

The characteristic behavior of black liquor density is shown in Figure 17 at dry solids contents to 100%. At 25°C, density is a linear function of dry solids content up to 65%. At higher dry solids content, the density first increases more rapidly with dry solids content (to about 80% dry solids content) and then increases much more slowly until it reaches the final density of dry black liquor solids. The change in slope near 65% dry solids content is associated with the shift from a water-continuous phase to a polymer-continuous phase, while the second transition may be due to change in plasticization (by water) of the glassy polymer continuous phase.

The effect of temperature on black liquor density can be estimated from Eq. 9 for the temperature range 20-100°C and dry solids contents to 65%.

$$\rho_T/\rho_{25} = 1 - 3.69 \times 10^{-4} (T-25) - 1.94 \times 10^{-6} (T-25)^2 \quad (9)$$

where:

ρ_T = black liquor density at temperature T, kg/m³

T = temperature, °C

For dry solids contents above 65%, the data in Figure 17 provides an estimate currently available for the density of black liquor at 25-30°C.

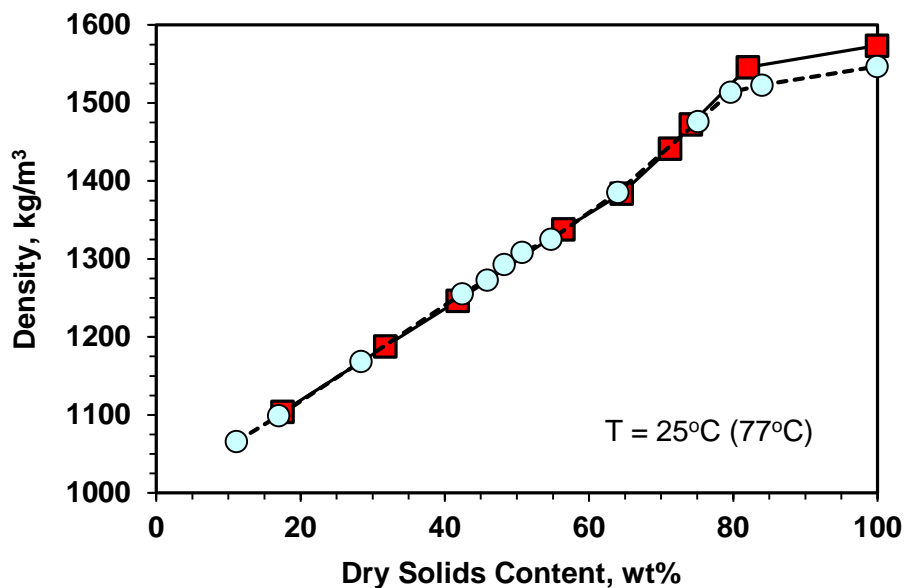


Figure 17. Density versus dry solids content for two North American kraft black liquors at 25°C.

3.3.4. Heat Capacity and Enthalpy

Enthalpy data for black liquor is needed to estimate the preheat requirements during evaporation. It can also be used with normal boiling point data to estimate the boiling point rise of concentrated black liquor.

Figure 18 contains heat capacity data from three sources. The heat capacities of most organic and inorganic matter typical of black liquor are about half the value for water. As the dry solids content increases from zero, the heat capacity falls approximately linearly from the value for pure water to a value about half that of water. Considering the differences in wood species and test methods, the data agree well. The data from Masse et al. is far more detailed and shows an

important feature of heat capacity for black liquor - the small deviation from a linear dependence on solids between 40% and 100% solids.

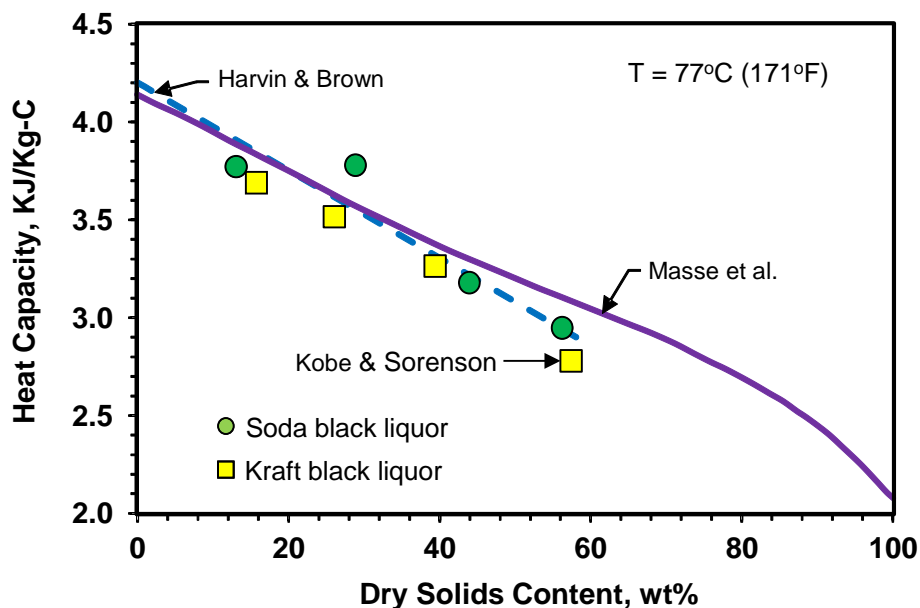


Figure 18. Comparison of heat capacity data for black liquor at 77°C.^{25,26,27}

A linear mixing rule for water and black liquor solids works for estimating the heat capacity of black liquor below 50% dry solids content. However, the character of black liquor changes substantially above about 50% solids. At low solids, black liquor is a mixture of an aqueous solution of inorganic salts, sugar acids, and colloidal material, with water as the continuous phase. Above 50% dry solids content, some inorganic salts begin to crystallize from the aqueous phase, and eventually the organic matter becomes the continuous phase. A mixing rule that incorporates an excess property function is used account for the departure of the heat capacity of black liquor versus dry solids content from that of an ideal mixture. The general form of the expression is:

$$C_{p_{bl}} = (1-S) C_{p_w} + (S) C_{p_s} + C_{p_E} \quad (10)$$

where:

$C_{p_{bl}}$ = heat capacity of black liquor, J/kg°C

C_{p_w} = heat capacity of water, 4216 J/kg°C (1.008 Btu/lb_m°F)

C_{p_s} = heat capacity of black liquor solids, J/kg°C

C_{p_E} = excess heat capacity, J/kg°C

S = dry solids mass fraction

T = temperature, °C

The heat capacity of dry black liquor solids changes with both temperature and composition. Typically, the heat capacity at 60°C falls within 10% of 1950 J/kg°C (0.46 Btu/lbm°F) and at 105°C within 10% of 2160 J/kg°C (0.51 Btu/lbm°F). An equation that describes the temperature dependence of the heat capacity of dry black liquor solids is

$$C_{p_s} = 1684 + 4.47 T \quad \pm 8\% \text{ for } T \text{ in } ^\circ\text{C} \quad (11)$$

An equation for the excess heat capacity term is:

$$C_{p_E} = (4930 - 29 T)(1 - S)(S)^{3.2} \quad (12)$$

The constants in equations 11 and 12 are based on C_p data for several black liquors. They and equation 10 predict the heat capacity for these liquors within 3%.

3.3.5 Heat of Dilution

Heat effects due to dilution arise because the forces of attraction between dissimilar molecules differ from those between similar molecules. The heat effect can be either exothermic or endothermic.

Heats of dilution data for black liquors diluted from 80% to 1.7% dry solids content are shown in Figure 19. The heat effect is exothermic, with a magnitude of about 150kJ/kg black liquor solids for dilution from 80% solids to infinite dilution. This is a significant energy release, equivalent to the energy required to increase the temperature of black liquor at 50% dry solids content by 20°C.

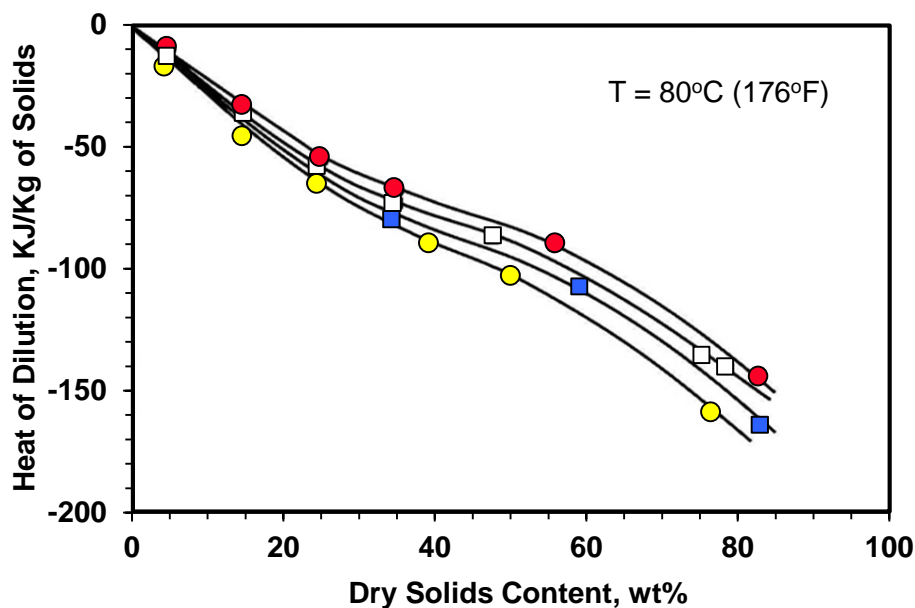


Figure 19. Heat of dilution data for four kraft black liquors from 80% dry solids content to infinite dilution at 80°C.

3.3.6 Enthalpy

The enthalpy of any solution, relative to an arbitrary reference state, can be calculated from heat capacity and heat of dilution data. The enthalpy of black liquor at 80°C, relative to water at 0°C, and dry solids contents from zero to 80%, can be calculated from Equation 13.

$$H_s = (1-S) H_w - (S) Q^\infty \quad (13)$$

where:

H_s = the enthalpy of black liquor at 80°C and dry solids content S

H_{aw} = the enthalpy of water at 80°C relative to 0°C

Q^∞ = the heat of dilution per unit mass of solids at 80°C from solids S to infinite dilution

Figure 21 contains enthalpy-concentration data for a kraft black liquor. The enthalpy values were calculated using Equation 13 and a measured heat of dilution for the liquor. They increase with increasing temperature and with decreasing dry solids content. They are less sensitive to dry solids content at higher dry solids content and lower temperatures. For different black liquors, the enthalpy at fixed temperature and dry solids content can vary by more than 20 kJ/kg. Enthalpy-concentration data for a variety of kraft black liquors are available.

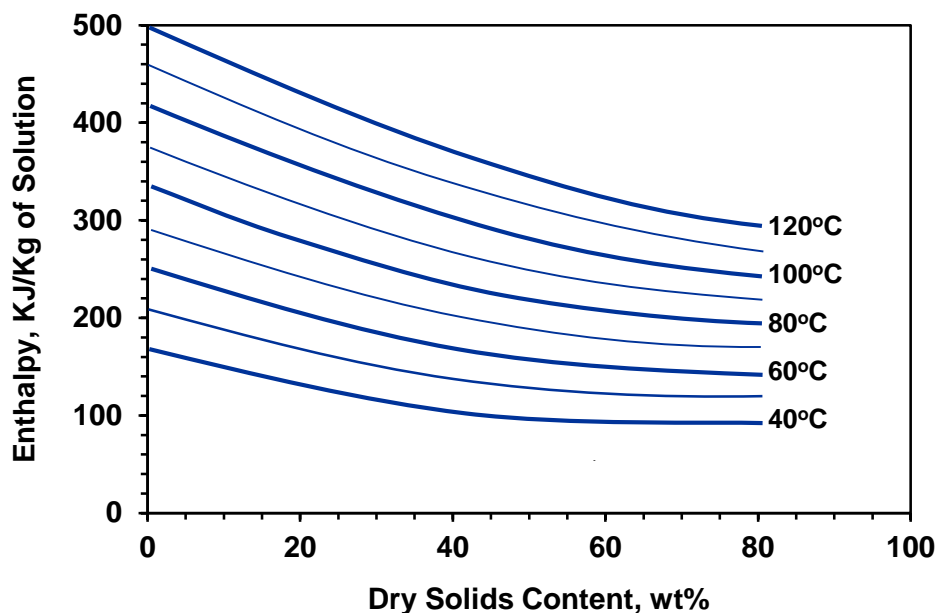


Figure 21. Enthalpy-concentration data for a kraft black liquor.

3.4 Surface Tension

Surface tension is the property of a liquid that tries to minimize its surface area. This property is responsible for the rise of a liquid in a capillary, and for the break-up of fluid filaments into spherical droplets during spray formation.

The surface tension of pure liquids depends on its temperature, while for solutions it depends also on the nature of the dissolved compounds. The surface tension of water is unusually high. At 20°C it is 72.8 dynes/cm ($72.8 \times 10^{-3} \text{ N/m}$). It decreases linearly with temperature to a value of 58.9 dynes/cm ($58.9 \times 10^{-3} \text{ N/m}$) at 100°C. Inorganic compounds increase the surface tension when dissolved in water, though this effect is mild within the range of concentrations found in black liquor. Of the main inorganic constituents in black liquor, NaOH has the strongest effect on the surface tension of water, increasing it approximately 1% for each 1 wt-% NaOH in solution. Organic surfactants such as tall oil soap decrease the surface tension of water by a factor of two

to three. This effect of surfactants on surface tension is very non-linear with concentration, and larger quantities of organic in water have only a very small additional effect on surface tension.

An important factor in both measurement surface tension and its impact on mill operation is the time factor for the surfactant concentration to equilibrate at a gas-liquid interface. When a liquid surface is formed, the concentration of the surface-active molecules at the liquid surface increases compared to that in the bulk liquid. Surface-active molecules are usually large, and the time needed for them to diffuse to a newly formed liquid surface can be long. Most surface tension measurement techniques measure the *static* surface tension—in these measurements the liquid surface is formed long enough before the measurement that the surface-active molecules have reached their equilibrium concentration at the surface. A method for measuring *dynamic* surface tension is available.²⁸ With this method the surface tension can be measured within 0.1 second of the time after which a surface is formed.

The effect of dry solids content on surface tension for five kraft black liquors is shown in **Figure 22** at temperatures from 22-90°C. The general pattern for the effect of solids is clear. Surface tension falls rapidly with increasing dry solids content to a value between 40% and 60% of that of pure water. The surface tension may go through a flat minimum between 15% and 40% solids.

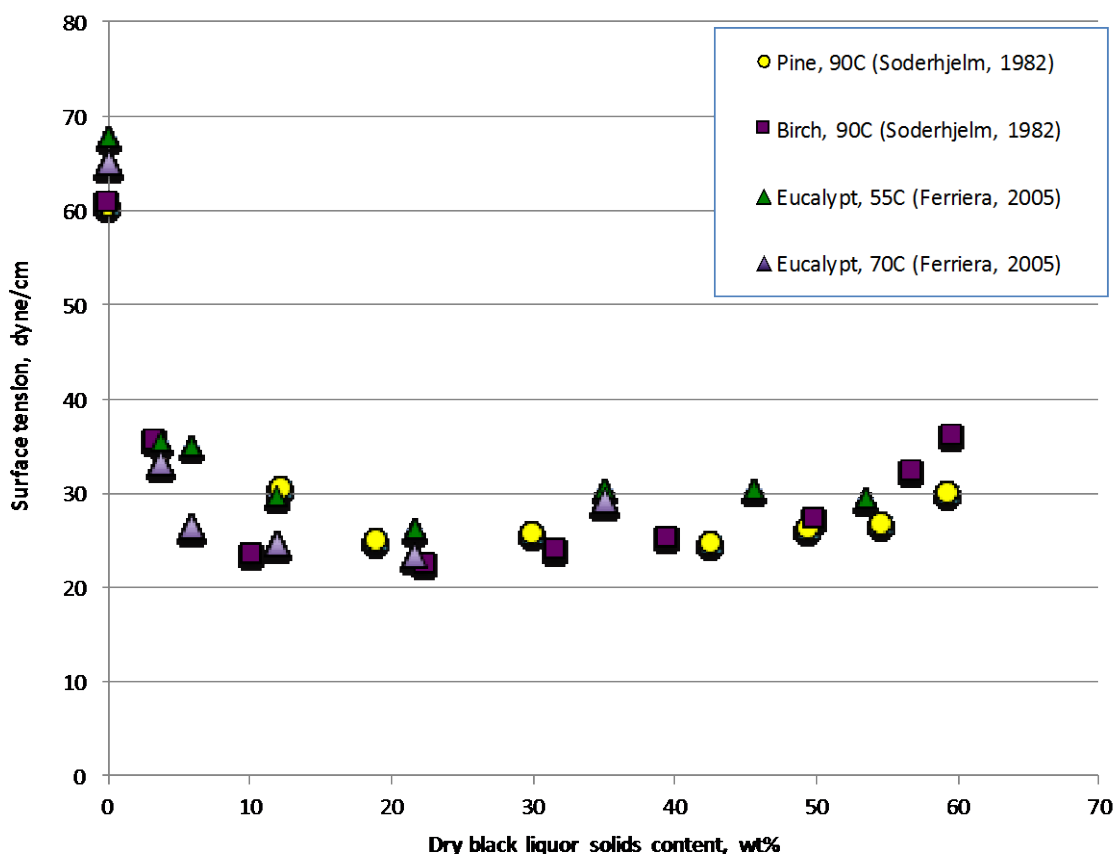


Figure 22. Effect of dry solids content on black liquor surface tension.^{29,30}

The effect of temperature on the surface tension of black liquor is about the same as for pure water.

The exact composition of the organic portion of black liquor undoubtedly influences measured results. Tall oil and turpentine are agents that depress the surface tension of black liquor. Beyond this there is little data on the effect of liquor composition on surface tension.

During spraying, black liquor droplets form in a millisecond time frame, and here dynamic surface tension is the more important one. Immediately after a new surface is formed, the

dynamic surface tension of black liquor is higher than the static value measured at longer times. The change in surface tension with time after a surface is formed is shown in **Figure 23** for one liquor. At low to moderate dry solids content, the dynamic surface tension of black liquor approaches that of water. At dry solids contents above 50%, the *apparent* surface tension increases with increasing dry solids content as suggested by the data in **Figure 22**. At these dry solids contents, the apparent surface tension data reflect the effects of liquor viscosity as well on the apparent (measured) surface tension changes; water is not the continuous phase, and the dynamic surface tension may depend on the composition of the organic matter as well as the effect of viscosity.

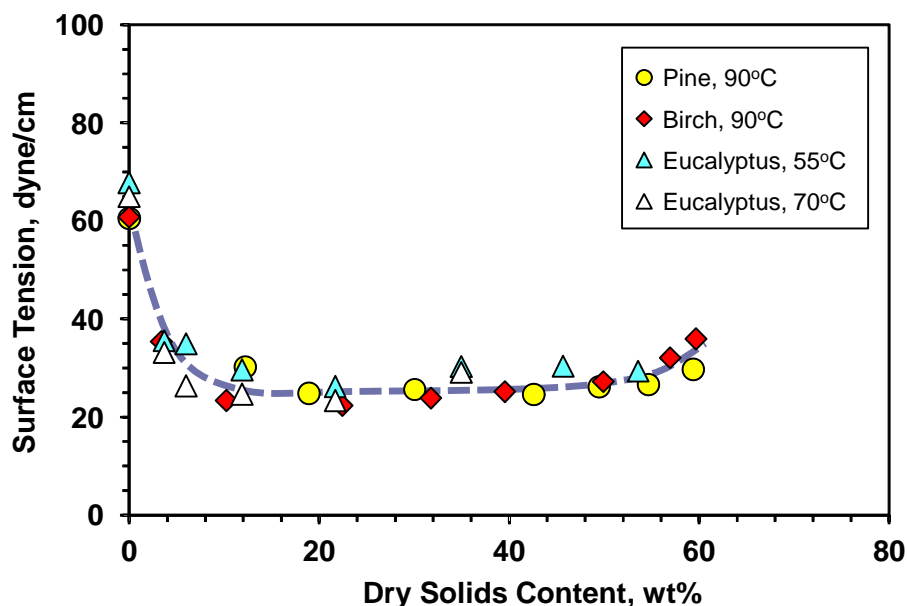


Figure 23. Time to reach equilibrium surface tension after the formation of new black liquor surface.³¹

3.5 Thermal Conductivity

Thermal conductivity is the property of a material that characterizes its ability to conduct heat. It is important in calculations of heat transfer rates during black liquor evaporation and in assessing the heat transfer within droplets as they dry and devolatilize under high temperature furnace conditions.

The data available on the thermal conductivity of black liquor is quite limited. Figure 25 shows the effect of dry solids content (to 55%) and temperature (to 100°C) for one kraft black liquor. The thermal conductivity decreases with increasing dry solids content and increases with increasing temperature.

Figure 26 compares thermal conductivity data for a second kraft black liquor with interpolated and extrapolated values from Figure 26. There is essentially no difference in thermal conductivity between these two liquors.

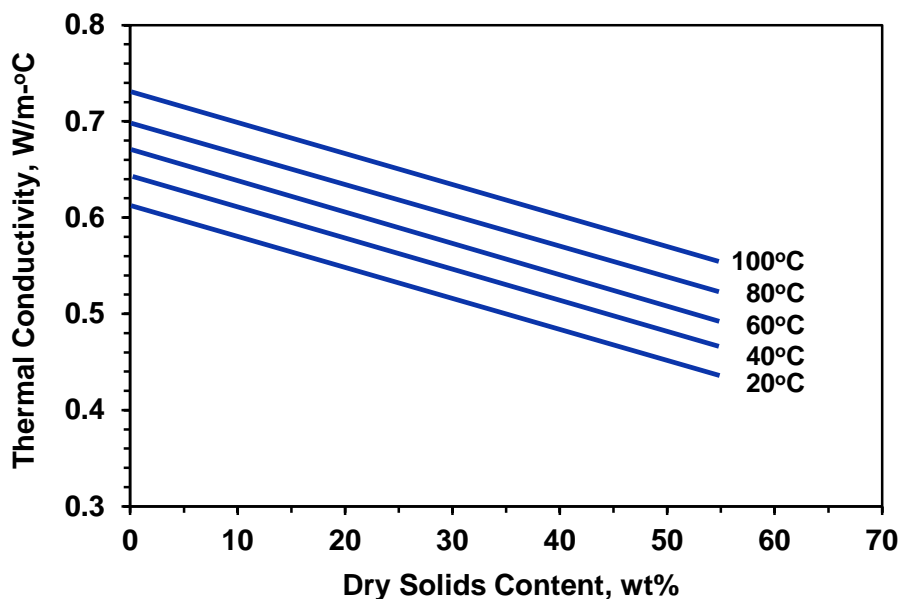


Figure 25. Thermal conductivity of a kraft black liquor (Harvin, 1955).

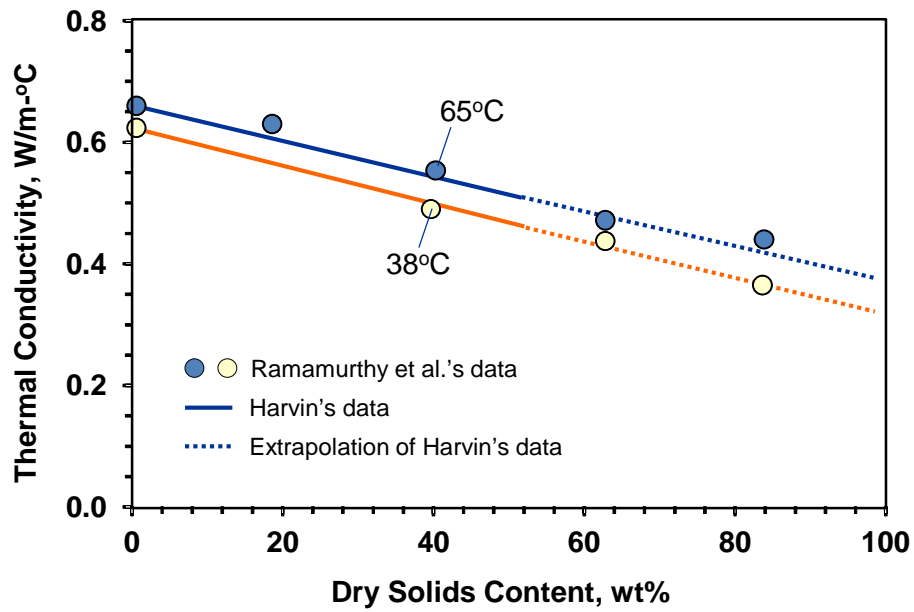


Figure 26. Thermal conductivity versus dry solids content for two kraft black liquors (Ramamurthy, 1993).

An empirical equation which fits the data in Figure 3-25 within $\pm 4\%$ is:

$$k = 1.44 \times 10^{-3} T - 0.335 S + 0.58 \quad (13)$$

where:

k = thermal conductivity, W/m°C

S = dry solids mass fraction

T = temperature, °C

Equation 13 can be used with confidence over the range of dry solids contents 0-82% and temperatures from 20-100°C. Extrapolation to higher temperatures is probably valid as well, although there is no data available with which to test this.

The general pattern of Figures 25 and 26 is reasonable. Dissolved inorganic matter changes the thermal conductivity of water only slightly, while organic matter typically has a thermal conductivity about half that of water. Thermal conductivity of the liquor would then be expected to fall from the value for water to approximately half that value as the dry solids content is increased to toward 100 wt%. Beyond this, there is too little data to assess the effects of black liquor composition.

3.6 Summary

Black liquor is a complex mixture of materials. The variety of raw materials and diverse pulping practices result in a very wide range of chemical and physical characteristics of black liquor. This has made chemical characterization and determination of physical properties a difficult task. Procedures are available for measuring key chemical constituents such as dry solids content, sulfide content, residual active or effective alkali, other major inorganic species, and trace metals. Data are available for a variety of mill and laboratory liquors, and at least a reasonable understanding of the important liquor properties is available. In some cases the results have been comprehensive enough to draw general conclusions about black liquor, and to yield estimating procedures useful in mill practice.

Some of the properties of black liquor can now be estimated with reasonable confidence using only a very small amount of data for a specific liquor or mill operation. Liquor density, surface tension, and heat capacity fall in this category. Particular mill conditions can change each of these properties, but the range of variation is not very wide, and relatively easy means are available to estimate them for any particular mill condition.

There are three other properties for which there is a data base useful in identifying general levels and trends as a function of liquor state. These are viscosity, boiling point rise, and the solubility of the main inorganic components. Although general information is available for each of these,

all three are sensitive to liquor composition. Also, all have an important impact on the design and operation of evaporators. Measurement procedures for each of these properties are relatively complex and have not been reduced to routine mill procedures.

In a category by itself is the thermal conductivity of black liquor. The small amount of data available on thermal conductivity of two black liquors suggests that liquor-to-liquor differences may be small. Although this data is reasonable, it does not cover a range of liquors which would allow the data to be used with confidence. The values reported might be adequate to estimate the range of thermal conductivity for gross heat transfer calculations, but would not be adequate for estimating heat transfer coefficients in evaporators which now must be obtained empirically. More data and analysis is need on the thermal conductivity of black liquor.

Nearly all of the published data on the properties of black liquor are based on measurements of liquors produced from North American or northern European wood species. Data on South American and Asian wood species such as eucalyptus are becoming more available, but more data are needed for comparison with the North American and European data. Progress is being made in this direction.

3.7 Nomenclature

C_p = heat capacity of black liquor, J/kg°C

C_{pE} = excess heat capacity, J/kg°C

C_{ps} = heat capacity of black liquor solids, J/kg°C

C_{pw} = heat capacity of water, J/kg°C

H_s = enthalpy of black liquor at 80°C and dry solids content S

H_w = enthalpy of water at 80°C relative to 0°C

ΔH_v = enthalpy of vaporization of water, J/kg

$\Delta H_{v-1 \text{ atm}}$ = enthalpy of vaporization of water at one atmosphere = 2244 kJ/kg

ΔH_{v-p} = enthalpy of vaporization of water at pressure p, kJ/kg

k = thermal conductivity, W/m°C

m = molality, mol/kg water

MW_w = molecular weight of water = 18

Na_{bls} = wt% sodium in black liquor solids, %

Q^T = heat of dilution per unit mass of solids at 80°C from dry solids content S to infinite dilution.

R = ideal gas constant, 8.2×10^2 atm m³/kmole °K

S = black liquor dry solids mass fraction

sulfidity = liquor sulfidity based on AA, %

T = temperature, °K

T^* = arbitrary reference temperature = 373°K

T_{bw} = boiling point of pure water, °K

T_{bw-1 atm} = boiling point of water at one atmosphere = 373 °K (100°C)

T_{bw-p} = boiling point of water at pressure p, °K

ΔT_b = boiling point rise (BPR), °C

ΔT_{b50} = boiling point rise at 50% dry solids content, °C

$\Delta T_{b50-1 \text{ atm}}$ = BPR at 50% solids at one atmosphere, °C

ΔT_{b50-p} = BPR at 50% solids at pressure p, °C

v = velocity, m/s

y = distance from surface, m

μ = viscosity, Pa-s

ε_{caust} = causticizing efficiency, %

ε_{reduct} = reduction efficiency, %

ρ = black liquor density, kg/m³

| = shear stress, Pa

Subscripts

bl = black liquor

w = water

T = temperature

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